

ANALYSIS AND TESTING TO IMPROVE  
THE FLOW FROM THE PLENUM OF A  
SUBSONIC CASCADE WIND TUNNEL

Richard Carl Moebius

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## Monterey, California



# THESIS

ANALYSIS AND TESTING TO IMPROVE  
THE FLOW FROM THE PLENUM OF A  
SUBSONIC CASCADE WIND TUNNEL

by

Richard Carl Moebius

March 1980

Thesis Advisor:

Raymond P. Shreeve

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Analysis and Testing to Improve  
the Flow from the Plenum of a  
Subsonic Cascade Wind Tunnel

by

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Submitted in partial fulfillment of the  
requirements for the degree of

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## ABSTRACT

The original design of the plenum and bellmouth contraction arrangement of a subsonic cascade test facility did not produce sufficiently uniform flow conditions at the bellmouth exit plane. Pneumatic measurements revealed sizeable blade-to-blade variations in velocity and flow angle. A finite element numerical analysis of the inviscid flow field was carried out which both confirmed the need for and guided the design of a modification to the bellmouth contraction from the plenum. Following the modification, in which the original contraction was changed to two two-dimensional contractions in series, and a program of development tests, acceptably small variations in velocity and flow angle were measured at the bellmouth exit plane.



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## I. INTRODUCTION

In 1963 construction was completed on the Naval Post-graduate School's subsonic cascade test facility. Its primary function was to be used to provide information applicable to compressor and turbine blade design. As described in Ref. 1, several unique features were incorporated in the design of the cascade to ensure a two-dimensional, periodic flow at the test blading. Previously the lack of good two-dimensionality in other facilities had plagued efforts to correlate air turning angles and blade surface pressure distribution data [Ref. 2].

Initial evaluations of the facility were carried out and reported in references 1 and 3. The test section entrance flow was examined and initially found to be unacceptably non-uniform. Several measures were taken at that time to correct the deficiency, but an examination of subsequent data revealed that only partial success was obtained. In particular, an investigation was performed using NACA 65 series blades as inlet guide vanes [Ref. 4]. Difficulty in obtaining two-dimensionality was reported and a strong recommendation was made that the test section inlet flow conditions be improved.

In preparing for tests of new compressor blade designs, the goal of the present study was to take whatever steps were necessary to first obtain a two-dimensional and periodic flow at the entrance to the cascade test section.



For purposes of this report, a two-dimensional and periodic flow will be considered to exist when the dynamic pressure variations are less than  $\pm 1\%$ , total pressure variations are less than  $\pm .5\%$ , and flow angles are within  $\pm 1.5$  degrees of their centerline value.

Preliminary pneumatic measurements indicated that the flow from the plenum itself was not uniform. A numerical analysis of the flow field through the plenum chamber and contraction bellmouth was therefore performed using the finite element method, (FEM). Based on the results of the preliminary measurements and the FEM analysis, plenum redesign and development test programs were carried through. The final pneumatic survey results showed acceptable levels of dynamic pressure, total pressure, and flow angle variations.

This report documents the experimental and analytical steps taken to obtain a suitable plenum geometry for the cascade facility. Following a description of the cascade and measurement techniques given in Section II, the steps taken in the development of a successful plenum arrangement are first outlined in Section III and then described in the following three sections. In order to present a clear account of the development, details of each section of the work have been confined to appendices. First, the FEM program is given in detail in Appendix A. Data from the original plenum configuration are given in Appendix B and the plenum development test results in Appendix C. Finally, detailed drawings of



the construction of the modification to the plenum and bellmouth are given in Appendix D.



## II. FACILITY AND MEASUREMENT APPROACH

### A. CASCADE AND INSTRUMENTATION

A description of the Naval Postgraduate School's Rectilinear Cascade as it was originally designed is given in reference 1. The layout of the complete facility is shown in Figure 1. The addition of two fine mesh screens at the bellmouth entrance to improve flow stability is the only modification to the plenum which is reported in reference 1. A follow-on study into the cascade performance was conducted by Bartocci and is reported in reference 3. Plenum turning vanes were installed by Bartocci to direct plenum inlet air towards the bellmouth entrance and to decrease the total pressure fluctuations. Multiple views of the plenum chamber and the bellmouth contraction arrangement as it was at the inception of the present work are shown in Figure 2.

All testing in the present study was conducted with the cascade test section walls vertical, with all inlet guide vanes (Fig. 1) and test blading removed. The effects of changes in plenum geometry were evaluated primarily by conducting pneumatic probe surveys at the exit of the bellmouth, hereafter referred to as the test plane. Additional information was obtained using tufts for surface flow visualization in the plenum and from static pressure distributions displayed on water columns. A Datametrics Series 100 VT Air Velocity Meter was also used to measure velocities in the plenum.





## B. PNEUMATIC PROBE SURVEYS

### 1. Probe and Installation

The installation for probe surveys at the test plane is shown in Figure 3. A wooden two-by-four was used to traverse the probe in the blade-to-blade direction. The coordinate system adopted is shown in Figure 4. The X-wise displacement of the probe was manually set by sliding the bar. The displacement in the Y direction and rotation of the probe were adjusted using the traversing probe mount shown in Figure 5, which was attached to the two-by-four and incorporates potentiometers for the measurement of the immersion (Y) and yaw ( $\beta$ ) displacements.

The five hole United Sensor D-125 probe shown in Figure 6 was used to determine total and static pressure, and yaw and pitch angle. A U-tube water manometer connected to the  $P_2$  and  $P_3$  ports was used to indicate when the probe was correctly aligned with respect to yaw angle. An approximate value for the static pressure was obtained by pneumatically averaging  $P_2$  and  $P_3$ . An exact calibration of the probe was not carried out since only relative measurements (showing the degree of uniformity across the test plane) were required.

### 2. Data Acquisition and Reduction

Probe and other reference pressures were multiplexed via a Scanivalve to a single transducer referenced to atmospheric pressure. The transducer and Scanivalve were connected to a B and F scanning data system. Signal conditioning



circuits in the data system were adjusted so that pressures were recorded in units of  $10^{-1}$  inches of water. Data was printed and also output on punched paper tape.

A Hewlett-Packard Model 9830A Programmable Calculator System with a two Megabyte mass memory disc unit and X-Y plotter were used to process the probe survey data from the paper tape.

The measured quantities and uncertainty intervals are given in Table I. The uncertainties in the pneumatic quantities were calculated from the  $2\sigma$  intervals of 25 consecutive readings at the test plane center. The position and yaw angle uncertainties were determined by the accuracy to which the appropriate scales could be read.

Since test plane Mach number did not exceed .25, compressibility effects were ignored. Test plane Reynolds number was  $1.5 \times 10^6$  per foot scale, with variations of  $\pm 5\%$ . In addition to the measured yaw angles, the following (relative) dimensionless flow properties were evaluated from the measurements in Table I, and their distributions across the test plane were examined:

Dynamic Pressure ( $\bar{Q}$ )

$$\bar{Q} = \frac{Q(X,Y)}{Q_{REF}}, \quad \text{uncertainty} = \pm .01,$$

$$\text{where } Q(X,Y) = P_T(X,Y) - P_S(X,Y)$$

$$Q_{REF} = P_T(30, 5) - P_S(30, 5) = Q(30, 5)$$

and the uncertainty in  $Q$  is  $\pm .008$ .



### Total Pressure ( $\overline{\Delta P}_T$ )

$$\overline{\Delta P}_T = \frac{\Delta P_T(X,Y)}{Q_{REF}} , \quad \text{uncertainty} = \pm .05,$$

where  $\Delta P_T(X,Y) = P_{PL} - P_T(X,Y)$  ,

and the uncertainty is  $\pm .049$ .

### Static Pressure ( $\overline{\Delta P}_S$ )

$$\overline{\Delta P}_S = \frac{\Delta P_S(X,Y)}{Q_{REF}} , \quad \text{uncertainty} = \pm .015,$$

where  $\Delta P_S(X,Y) = P_S(X,Y) - P_W$  , uncertainty =  $\pm .01$ .

Note that the uncertainties in  $\overline{\Delta P}_T$  and  $\overline{\Delta P}_S$  are rather high since these quantities are derived from small differences between two larger quantities. Note also that the values of  $Q_{REF}$  changed over the period of a test run due to variations in the plenum pressure. The value of  $Q_{REF}$  at each (X,Y) was therefore derived from the plenum pressure reading taken at the time of the  $Q(X,Y)$  measurement. This technique assumed a linear relationship between  $Q(30, 5)$  and  $P_{PL}$ .

## C. OTHER TECHNIQUES

Qualitative information which was simple to obtain, greatly accelerated the development of the final plenum configuration.

### 1. Tufts

Tufts were attached to the plenum walls and were observed and photographed using a Polaroid camera during testing.





(Average velocities of less than 10 ft/sec allowed the presence of an observer in the plenum.) A tufted probe was also used extensively.

## 2. Static Pressure Distribution

The uniformity, but particularly the steadiness, of the cascade sidewall static pressures displayed on water manometer columns were monitored as changes in configurations were made.

## 3. Velocity Meter

Velocities from the sound baffles in the plenum were measured using a Datametrics Series 100 VT Air Velocity Meter.



### III. PROGRAM OUTLINE

The goal was to obtain a plenum and contraction arrangement which would give at the test plane (outside the boundary layers) maximum peak-to-peak variations of  $\pm 1\%$  in dynamic pressure,  $\pm .5\%$  in the total pressure, and  $\pm 1.5$  degrees in the flow yaw angle.

The program was carried out in three steps. First, tests were made to determine the performance of the existing plenum configuration. Second, a finite element analysis was carried out which confirmed that the existing bellmouth could not generate a sufficiently uniform flow at the test plane. The analysis also guided the design of a suitable modification to the plenum. Third, an experimental program was carried out during which a major modification was made in the plenum geometry and a plenum and contraction configuration which gave satisfactory flow was subsequently developed.

These three steps in the program are documented in the following three sections of this report.

A summary of the configurations tested during the program is given in Table II. Effects of configuration changes were evaluated at the test plane by conducting blade-to-blade probe surveys (first at 50% span) and spanwise surveys at selected locations. The other measurement techniques listed in Section III.C. often were used first to determine whether a particular configuration should be measured in detail.



#### IV. PRELIMINARY TEST RESULTS

Blade-to-blade probe surveys were obtained with the original plenum arrangement (Configuration Ø) at 10, 30, 50, and 90% span. The results at 50% span are shown in Figures 7a, 7c, and 7e. (The remaining data are given in Appendix B.) The results showed unacceptable variations in total and dynamic pressure in both the blade-to-blade and spanwise directions. Spanwise variation of the various parameters can be seen by comparing values at given blade-to-blade locations in the results at 10, 30, 50, and 90% spans. The yaw angles were also outside the required bounds.

The source of the total pressure variation was traced to the screens at the bellmouth inlet. Clearly, the normal flow into the bellmouth would generate a velocity which would vary in magnitude and direction over the face of the screen, thereby generating a corresponding variation in the total pressure drop.

The screens were removed (Configuration 1) and the measurements repeated. Blade-to-blade survey results are shown in Figures 7b, 7d, and 7f. A spanwise survey was conducted at  $X = 30$  inches and is shown in Figure 8. The total pressure variation ( $\overline{\Delta P_T}$ ) across the test plane was seen to have decreased to  $\pm .5\%$ . The variation in  $\overline{Q}$  was within tolerance in the spanwise direction; however, an unacceptable gradient still existed in the blade-to-blade direction



at all spanwise stations surveyed. The yaw angles in Figures 7e and 7f suggested that the flow was still converging through the center 50% of the test plane. In addition, observed spanwise variations in flow angle near the ends in the blade-to-blade direction suggested the presence of large scale vorticity. This was verified using a small string tuft attached to the end of the pneumatic probe.

The question then was, - was the measured velocity field simply the result of the geometry of the bellmouth contraction, or could something be done in the plenum itself to correct the profile? An analysis of the flow from the plenum was required to answer this question.





## V. FINITE ELEMENT FLOW MODEL PREDICTIONS

A two-dimensional finite element analysis was used to analyze the potential flow field through the plenum and bellmouth contraction. Two questions were of specific interest. First, what was the velocity profile created by the bellmouth contraction at its exit plane, and secondly, what distance downstream was required to attain an acceptable uniform velocity profile? The details of the analysis and the Fortran programs developed for the IBM 360 to answer these questions are given in Appendix A.

The plenum, bellmouth, and exit passage were modeled in two ways as shown in Figure 9. Model 1 was the most simplified representation, and was used to obtain a first estimate of the velocity profile developed by the bellmouth contraction. Due to the vertical axis of symmetry in Model 1, only half of the model needed to be included in the calculations. A more realistic model was that shown in Figure 9 as Model 2. In Model 2 the flow entered the plenum from one side as in the actual plenum.

The computer analysis of each model was performed in a similar manner. Namely, the exit boundary condition, which specified flow to be parallel to the exit walls, was applied at boundary planes located at progressively increasing distances downstream of the bellmouth exit plane (Fig. 9). For each boundary plane, the computed velocity profiles across



the bellmouth exit plane and the boundary plane were compared with the results from the previous run. The following similarities were noted in the results of the two models:

1) The exit boundary plane velocity profiles were always more nearly uniform than the bellmouth exit plane velocity profiles.

2) As the exit boundary planes were moved farther downstream, the exit plane velocity profile became more nearly uniform. The bellmouth exit plane velocity profiles changed during this process.

3) As the exit plane velocity profile approached a uniform condition, the still non-uniform bellmouth exit plane velocity profile did not change as the exit boundary was moved even farther downstream.

The results of Model 1, shown in Figure 10, indicated that at exit plane (GH), 41.5 inches downstream of bellmouth exit plane (CL), the velocity profile had relaxed to give a maximum variation of 0.3%. The velocity profile at plane (CL) had a variation then of 14% between peak velocity at the contour wall and minimum velocity at the axis of symmetry. Similarly, the results of Model 2, shown in Figure 11, indicated that at exit plane (LM), 50 inches downstream of the bellmouth exit plane (FS), there existed only a 1.3% velocity variation. This velocity profile was not, however, symmetrical due to the asymmetry of the flow model. Higher velocities existed on the inlet side of the exit plane. The velocity profile at plane (FS) also exhibited the asymmetry and a



15% variation between maximum and minimum velocities. Figure 12 shows a comparison between the measured profile of  $\bar{Q}$  for Configuration 1 at 50% span, and that calculated from the velocity profile of Figure 11.

While the magnitudes are different, the trends in the computed and measured distributions of  $\bar{Q}$  are clearly similar. Since the analysis is a two-dimensional one, the magnitudes should not be expected to fully agree. While a three-dimensional inviscid analysis could be expected to be more representative, the inclusion of viscosity would be necessary to closely approximate the actual flow.

There is sufficient documentation in the literature to presume significant amounts of vorticity exist in this flow. The rectangular duct shape in itself can create sizeable corner vorticity [Ref. 5 and Ref. 6]. Velocity gradients in the boundary layer, under the influence of transverse pressure gradients, can produce sizeable amounts of vorticity also [Ref. 7]. As was stated by C. Truesdell [1954], "Even in nearly irrotational flows the relatively small amount of vorticity present may be of central importance in determining the major flow characteristics."

An analysis of the flow in the orthogonal plane (i.e., in the spanwise direction), was not performed. However, an approximate analytical solution for a similar but simpler model sheds light on the probable trends in the velocity distribution. For this analysis, the flow field is created by a circular cylinder immersed in the flow between two



parallel walls. This is illustrated in Figure 13. The velocity distribution across the stream at the plane of maximum contraction is given in Ref. 8 as the following:

$$V_X = U \left[ 1 - \frac{\sinh^2(\pi b/H) \cos(2\pi Y/H)}{\cosh^2(\pi X/H) - \cos^2(\pi Y/H)} + \frac{1/2 \sinh^2(\pi b/H) \sin^2(2\pi Y/H)}{[\cosh^2(\pi X/H) - \cos^2(\pi Y/H)]^2} \right]$$

The important result to note is that as the ratio  $b/H$  increases, the magnitude of the velocity variation decreases. Using a similar argument for the cascade bellmouth contraction, one would expect a difference in the maximum velocity variations, between the blade-to-blade and spanwise directions, of almost an order of magnitude. This was consistent with experimental observation.





## VI. PLENUM DESIGN MODIFICATION AND TEST RESULTS

The measurements reported in Section IV and the analysis reported in Section V suggested that the bellmouth contraction, as designed, could produce an acceptably uniform velocity profile in the spanwise direction, but not in the blade-to-blade direction. Several perturbations were subsequently made in the original plenum geometry, and the test results supported this conclusion. A complete account of the tests leading to, and subsequently verifying, the design of a modification to the plenum is given in Appendix C.

The basic idea in the modification was to arrange two two-dimensional contractions in series, each with sufficient length to allow relaxation to uniform velocity of the accelerated flow. The spanwise contraction of the original bellmouth contour was retained, and a new contraction section in the blade-to-blade plane was built down into the original plenum. The contours and construction of the new section are given in Appendix D. The final configuration (Configuration 7) of the plenum, including features which were determined to be necessary as a result of subsequent development testing (Appendix C), is shown in Figure 14 and Figure 15. Referring to these figures, the baffles were installed to dampen flow oscillations observed in earlier tests. The flow diverter fastened to the west end ceiling was installed to deflect



high velocity air from along the ceiling, downward through the baffles toward the plenum center. The velocity and yaw angle distributions (Configuration 7 at 50% span) are shown in Figure 16. Measurements from Configuration 1 are shown for comparison. The results of a spanwise traverse at  $X = 50$  for Configuration 7 are shown in Figure 17. The variations in both dynamic pressure and yaw angle were seen to have been brought to within the prescribed limits of  $\pm 1\%$  and  $\pm 1.5$  degrees respectively.



## VII. CONCLUSIONS

The original design of the plenum and bellmouth contraction arrangement did not produce sufficiently uniform flow conditions at the bellmouth exit plane. While the large area contraction ratio (40:1) ensured uniformity in the total pressure when no screens were introduced, the original bellmouth contraction contours in the blade-to-blade plane gave unacceptable variations in velocity and flow angle at the inlet guide vane station. The inlet guide vanes would normally be adjusted to control air-inlet angle to the test cascade.

Following the design and installation of a modification to the plenum, in which the original contraction was changed to two two-dimensional contractions in series, and a program of development tests, acceptably small variations in velocity and flow angle were measured at the inlet guide vane station.

Future tests will involve installation of the inlet guide vanes and an evaluation of the flow downstream at the cascade test blading location.



TABLE I  
MEASURED QUANTITIES

Item	Description	Method	Uncertainty
X	Blade-to-Blade dimension X = 0 in. East end X = 60 in. West end	Tape measure	$\pm .1$ in.
Y	Spanwise dimension Y = 0 in. South wall Y = 10 in. North wall	Immersion Potentiometer on probe mount	$\pm .01$ in.
$\beta$	Flow yaw angle	Yaw angle Potentiometer on probe mount	$\pm .25$ deg.
$P_T(X,Y)$	Total pressure at the test plane	Pressure $P_1$ on Pitot probe	$\pm .1$ in. $H_2O$
$P_{PL}$	Plenum total pressure	Static tap in plenum chamber $V \approx 0$	$\pm .1$ in. $H_2O$
$P_S(X,Y)$	Static pressure at the test plane	Pneumatic average of $P_2$ and $P_3$ on Pitot probe	$\pm .15$ in. $H_2O$
$P_W$	Static pressure at X = 30 in., Y = 0 in.	Static tap on south wall	$\pm .1$ in. $H_2O$
$P_{ATM}$	Atmospheric pressure	Mercury barometer	$\pm .01$ in. $H_2O$
$P_{TARE}$	Tare reading across pressure transducer	$P_{ATM} - P_{ATM}$	$\pm .01$ in. $H_2O$

Note: All pressures are gauge pressure.





TABLE II  
PLENUM/BELLMOUTH TEST CONFIGURATIONS

Configuration Number	Description	Figure Number
Ø	Original design	1 and 2
1	Removed screens from bellmouth entrance	1 and 2 minus screens
2	Simulated "relaxation" concept Added plywood vertical wall	C-1
3	Added eastern and western lower contractions	C-4 minus fillets
4	Same as above, but moved turning vanes 3 ft. to the east	As above
5	Inserted cardboard fillets between upper and lower contraction	C-4
6	Removed plenum turning vanes	C-4 minus turning vanes
7	Installed baffles and flow diverter	14 and 15



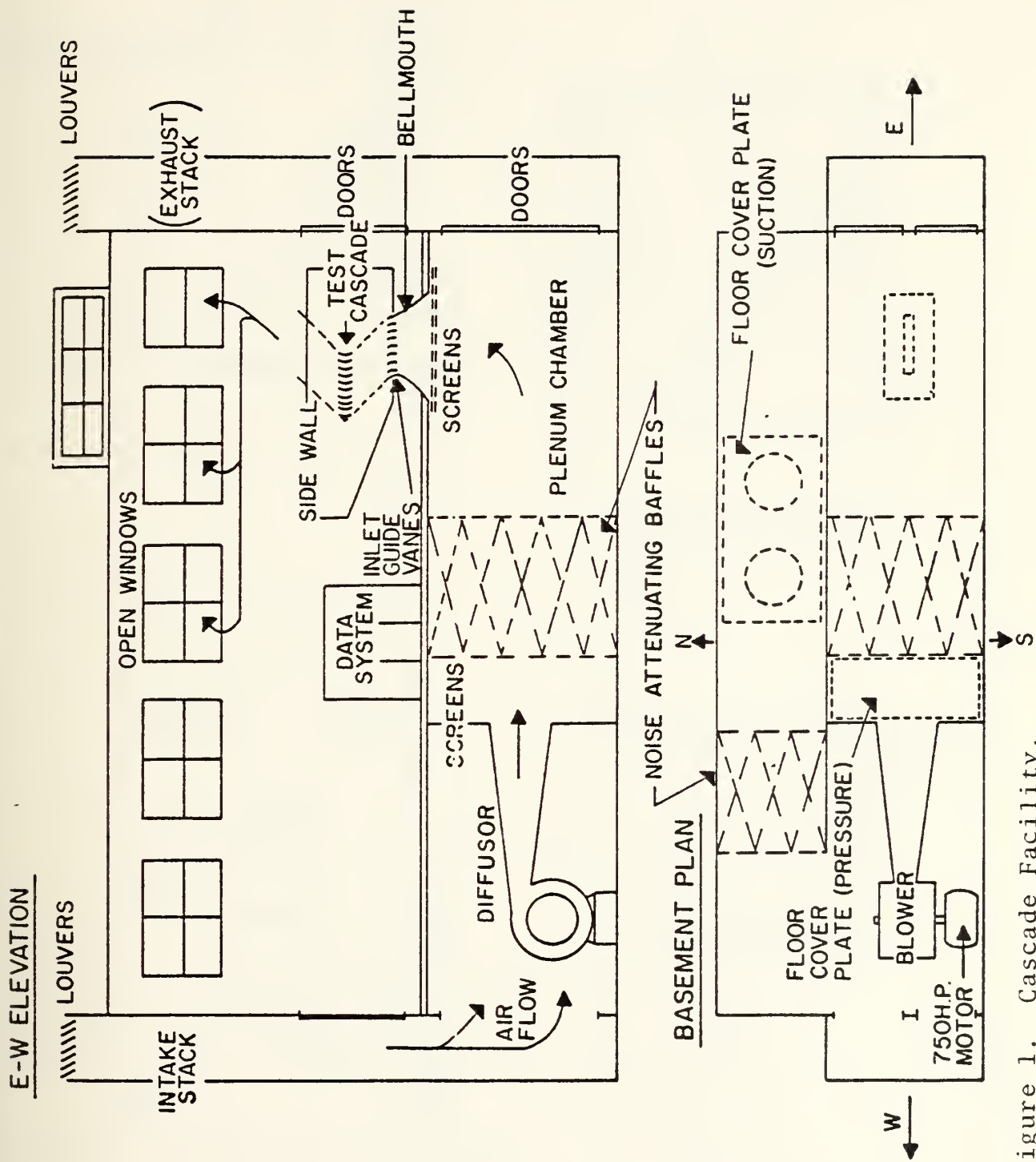


Figure 1. Cascade Facility.



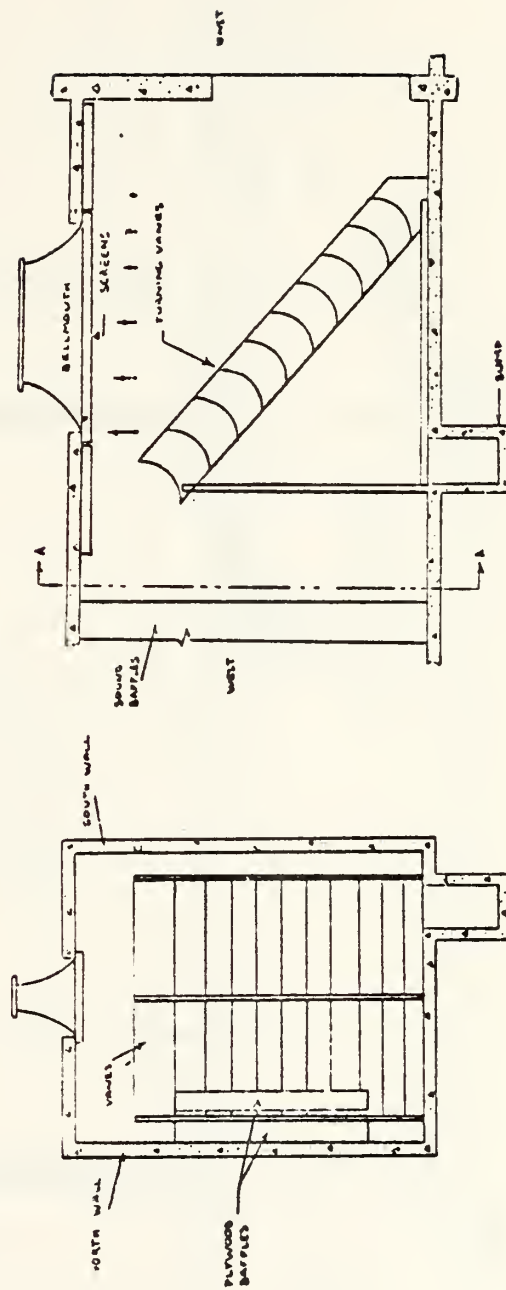
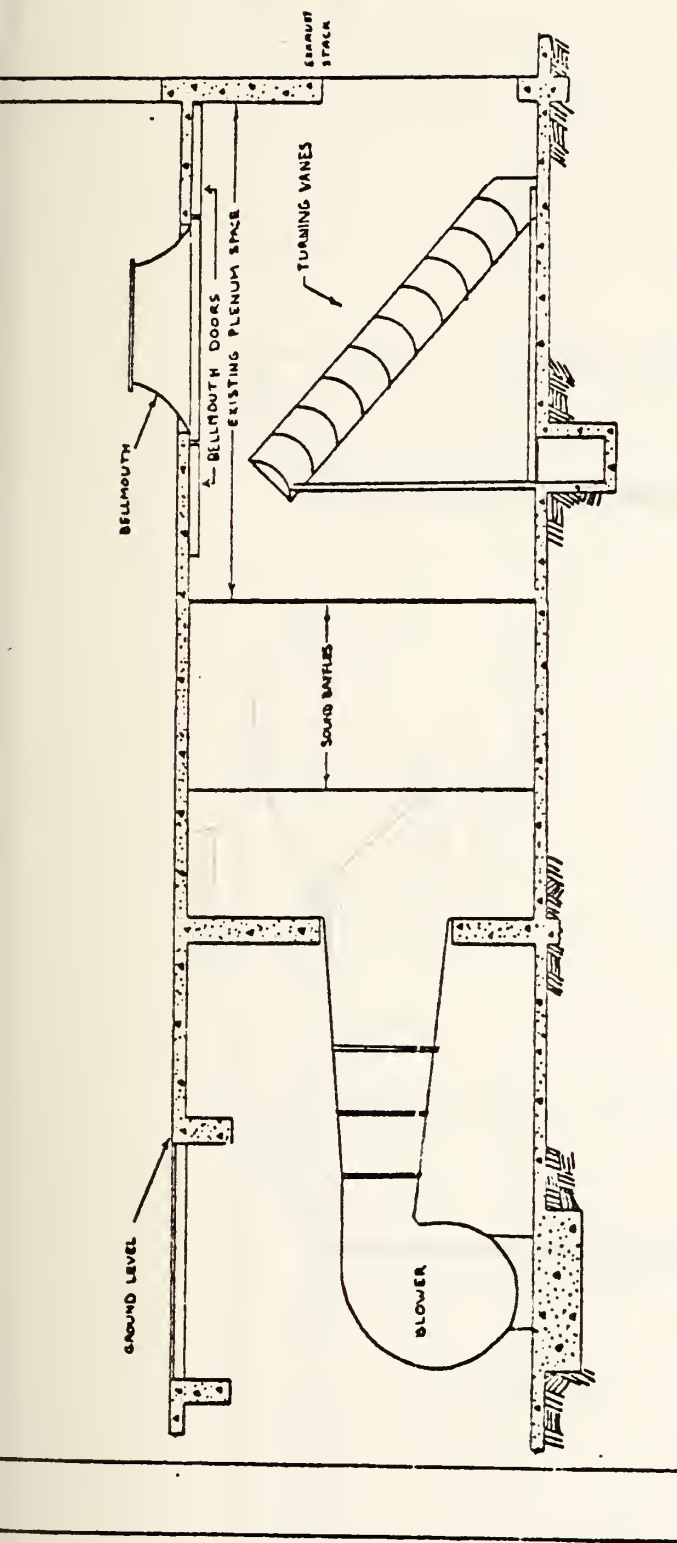


Figure 2. Plenum - Configuration #0



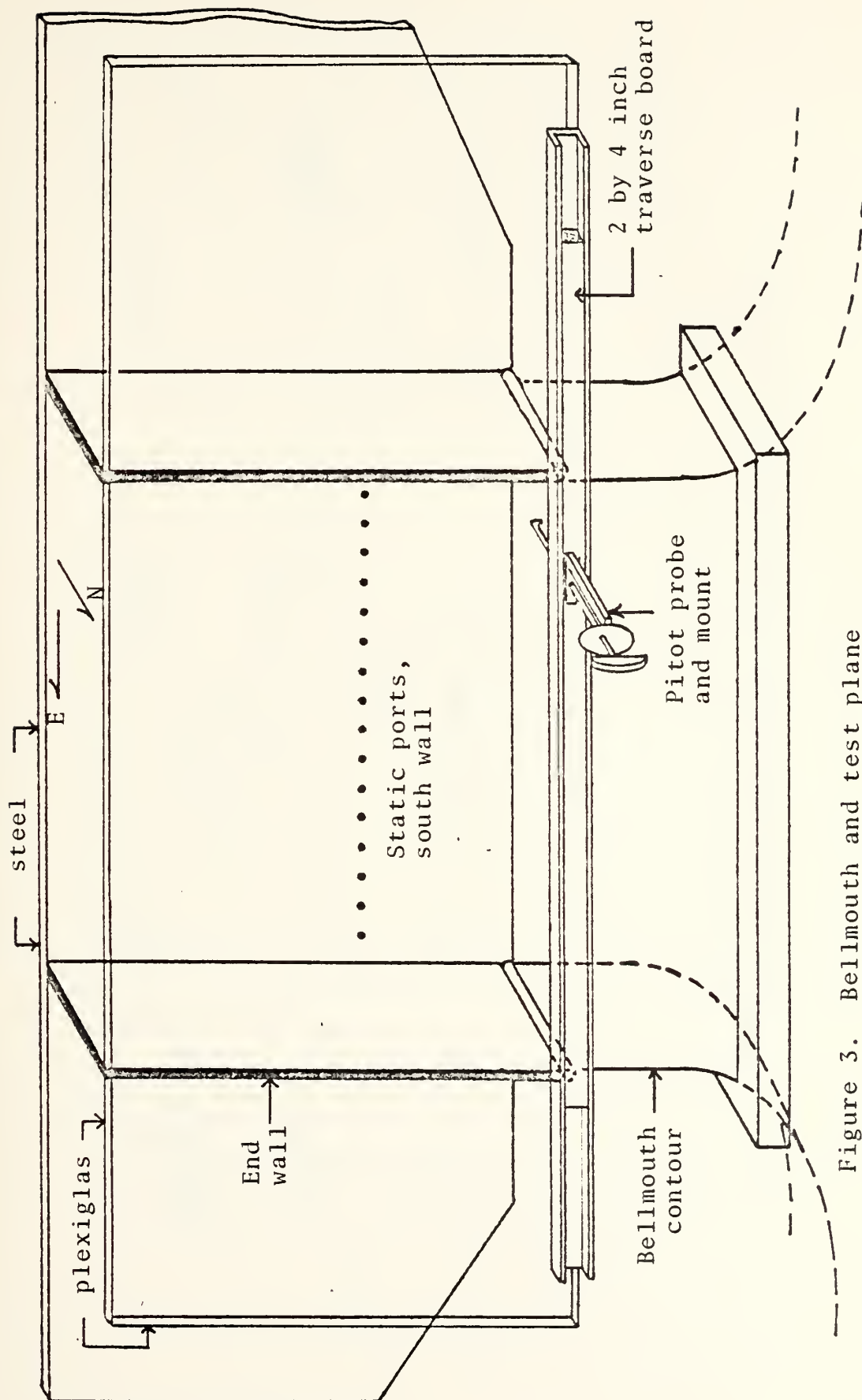


Figure 3. Bellmouth and test plane





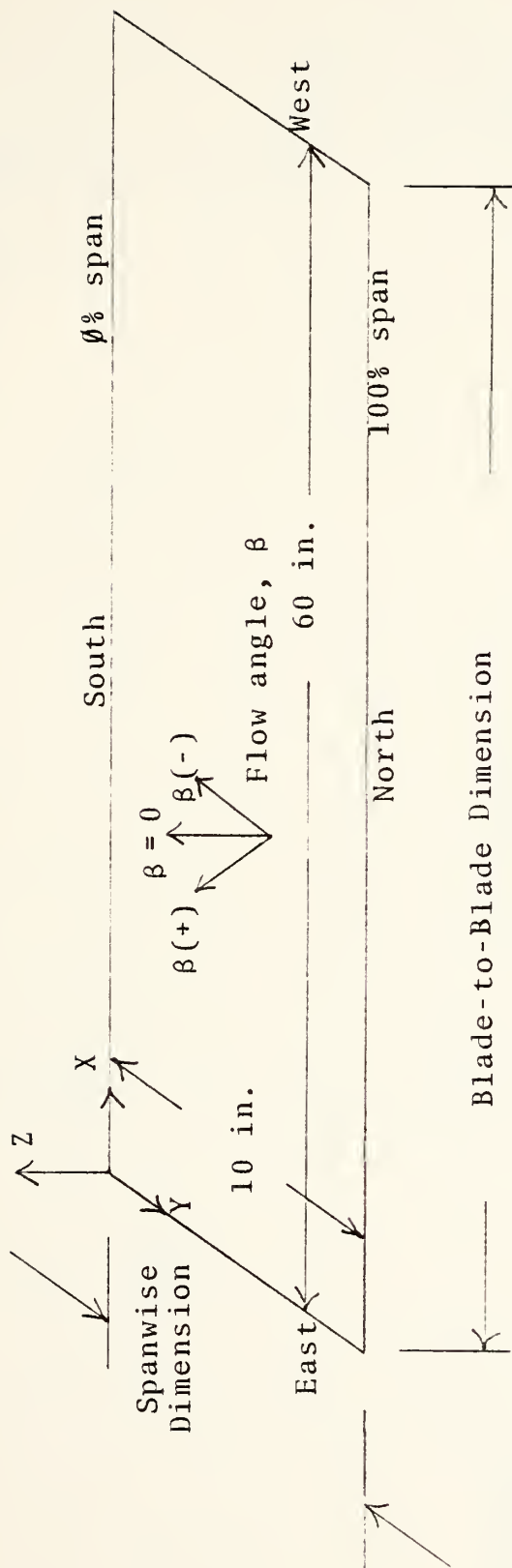


Figure 4. Cascade Nomenclature (Test Plane at Bellmouth Exit)



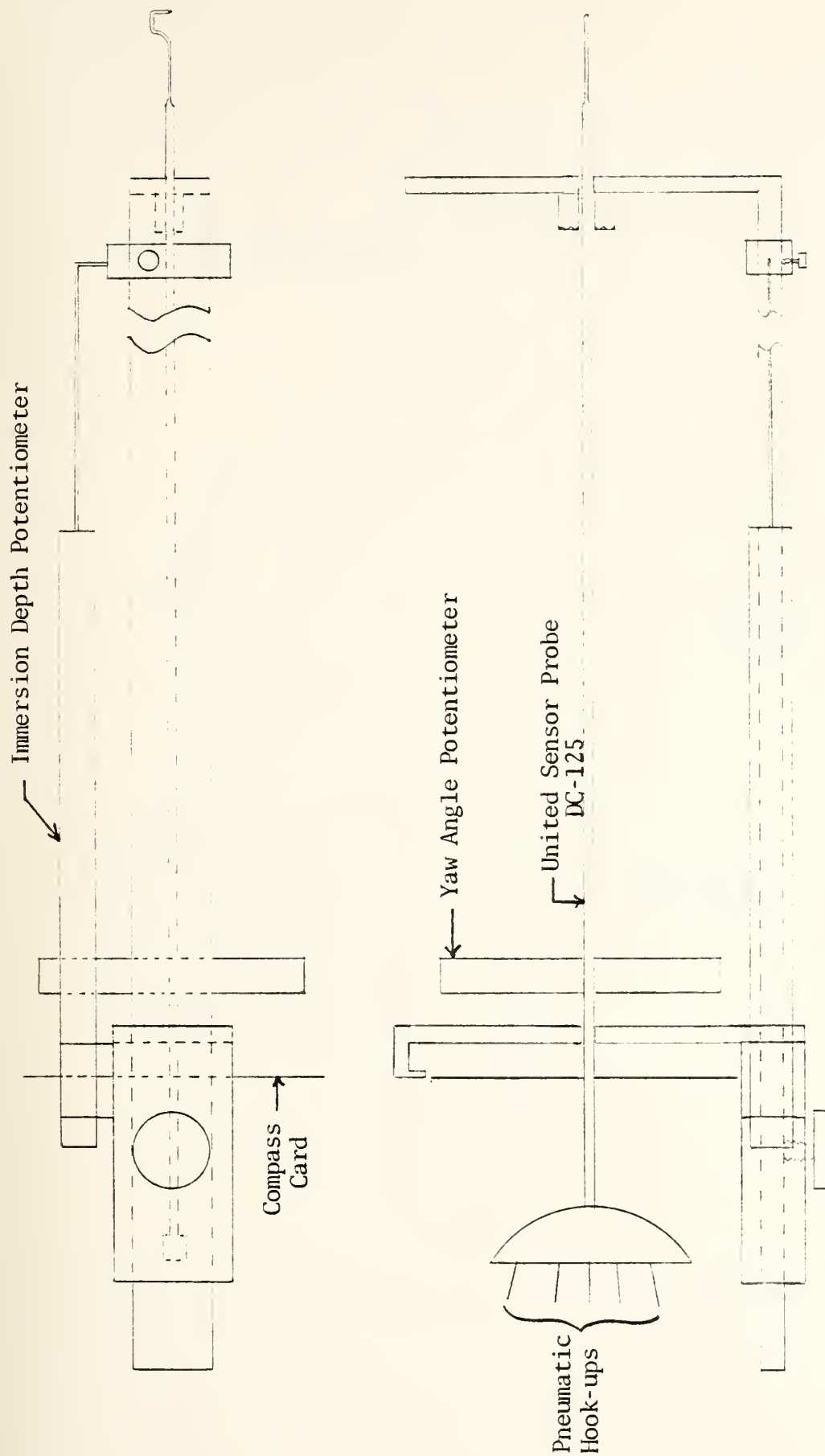
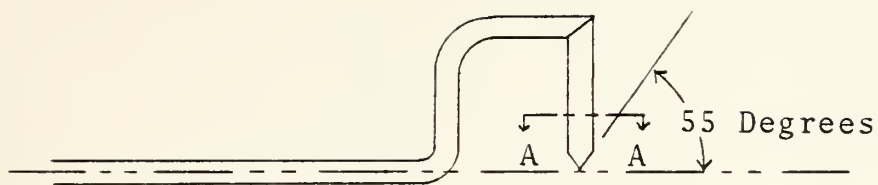


Figure 5. Pneumatic Probe and Traversing Probe Mount





A-A

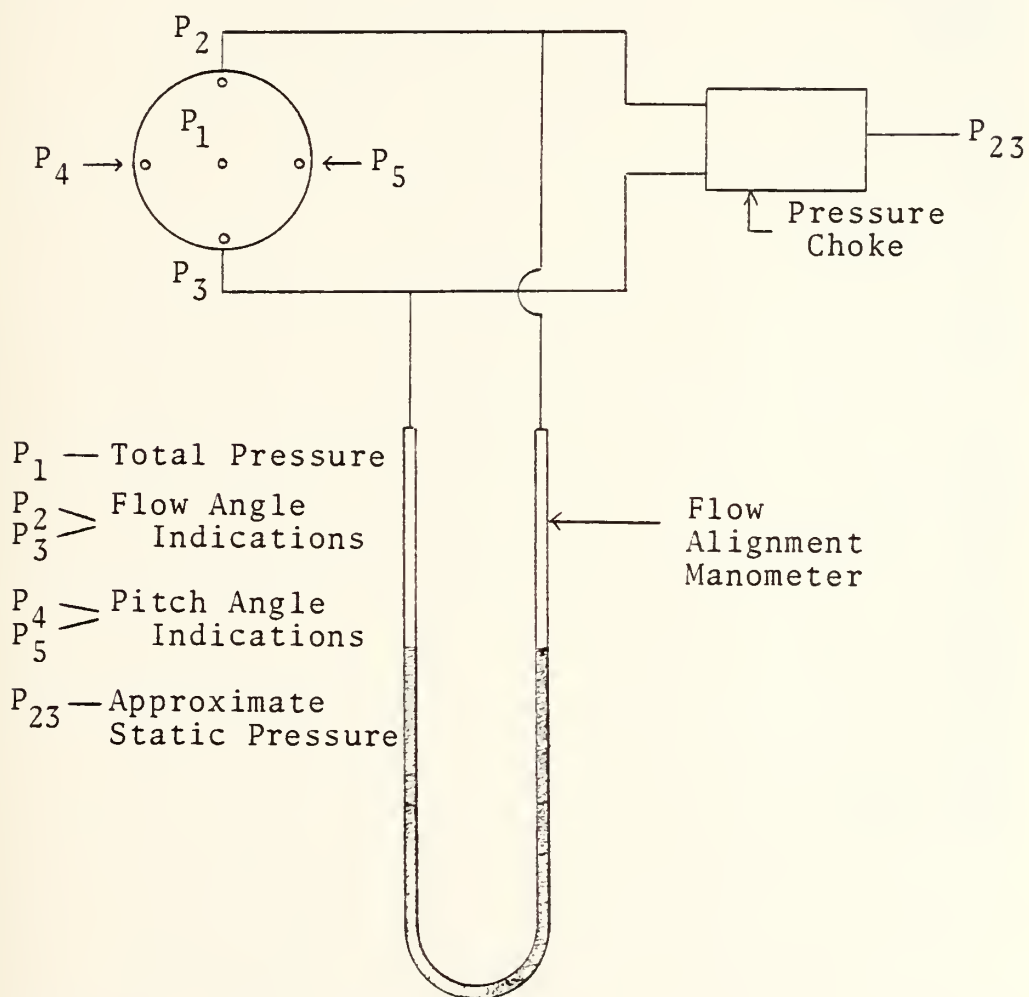


Figure 6. United Sensor DC-125 5-Hole Pneumatic Probe



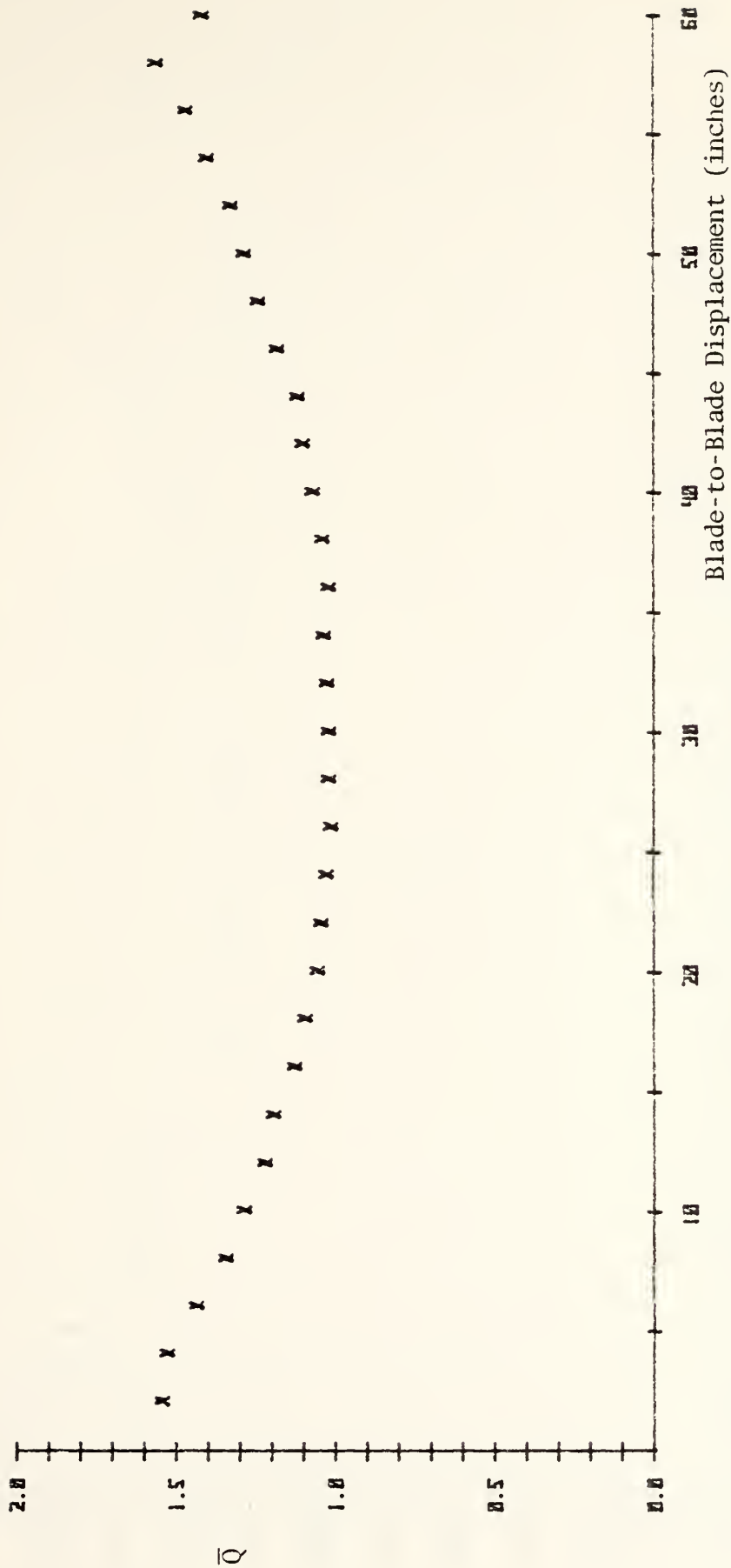


Figure 7a. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 230 ft/sec) - Configuration #0







Figure 7b. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 270 ft/sec) - Configuration #1



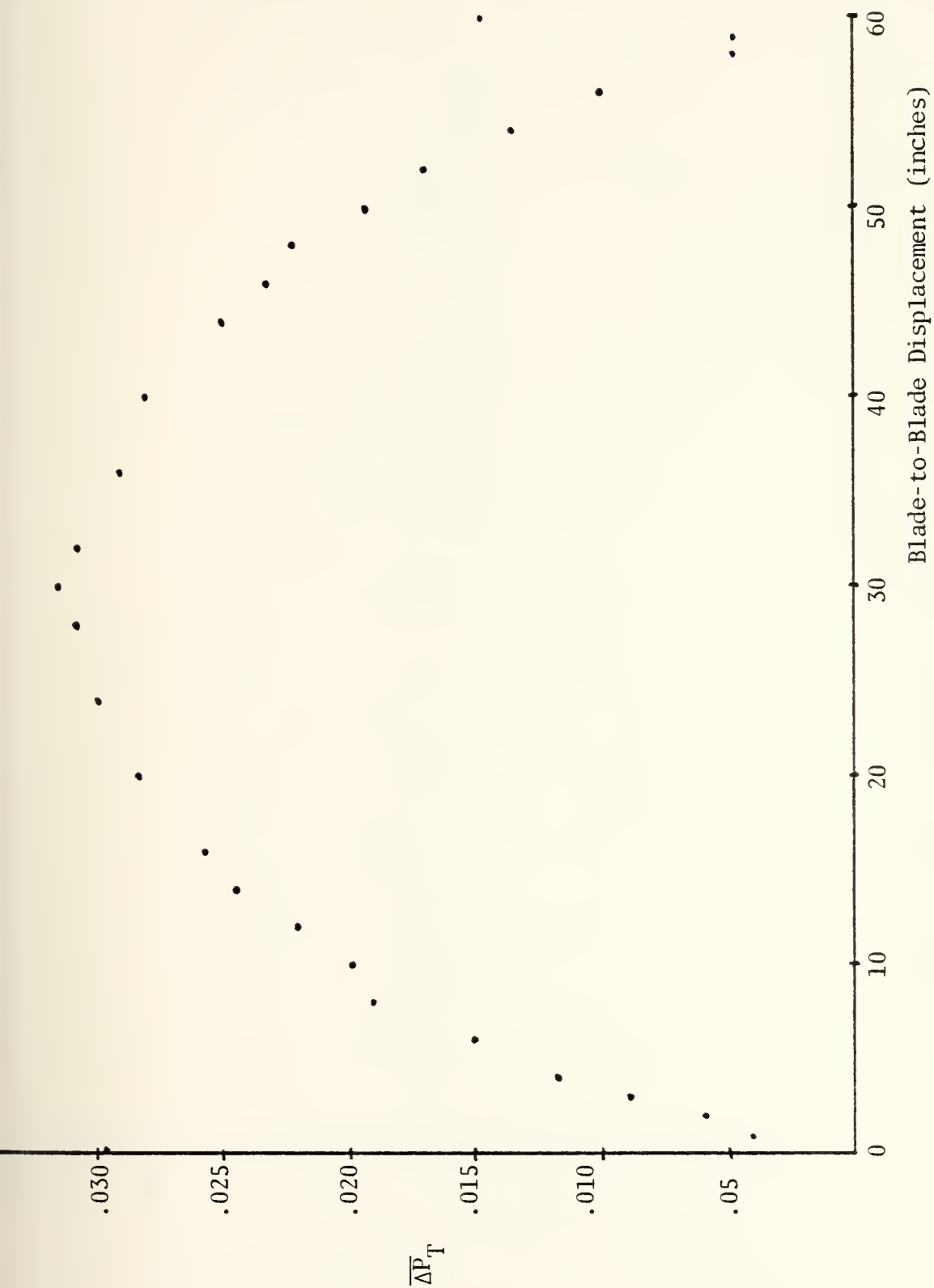


Figure 7c. Blade-to-Blade Total Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 230 ft/sec) - Configuration #0



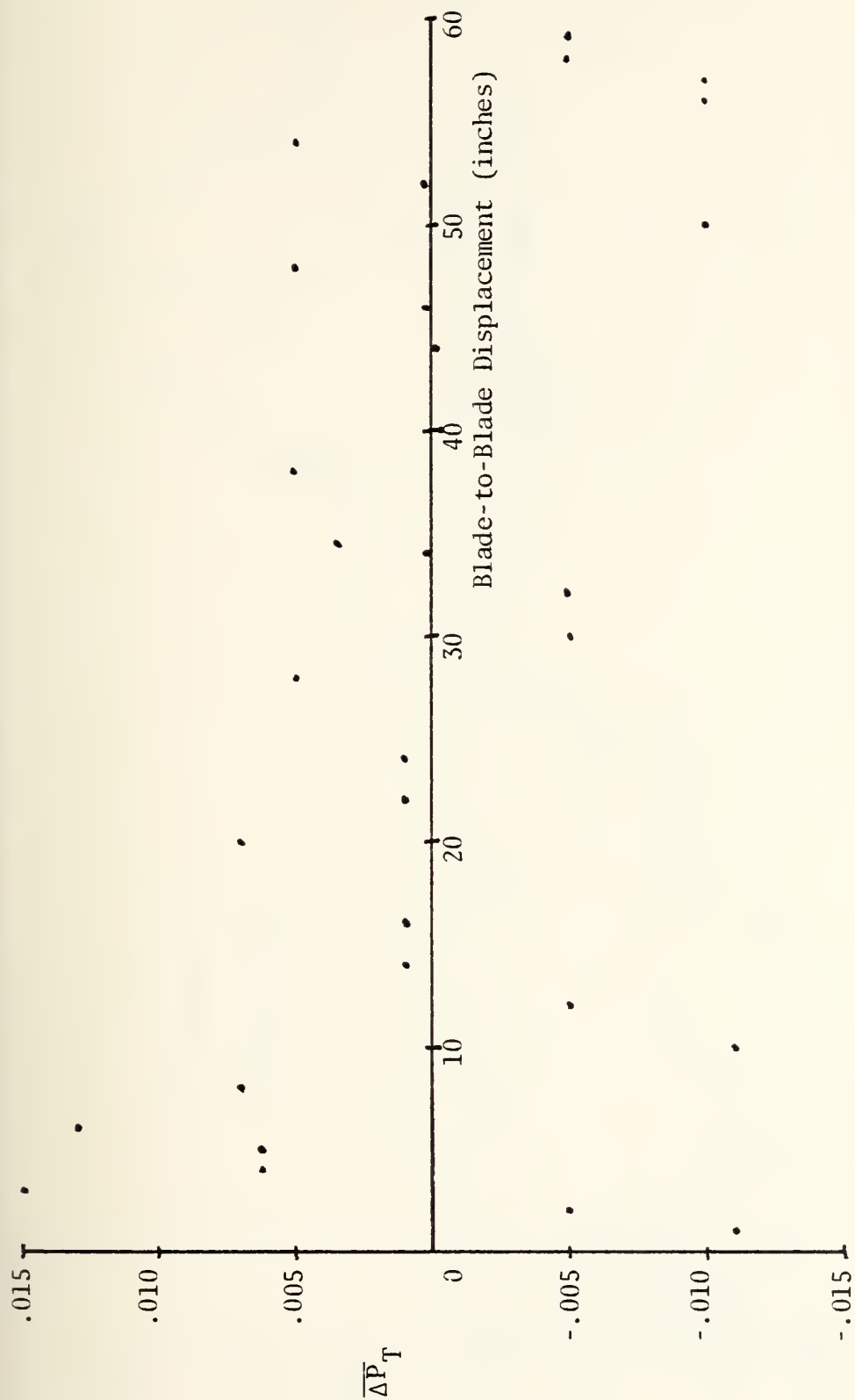


Figure 7d. Blade-to-Blade Total Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 280 ft/sec) - Configuration #1



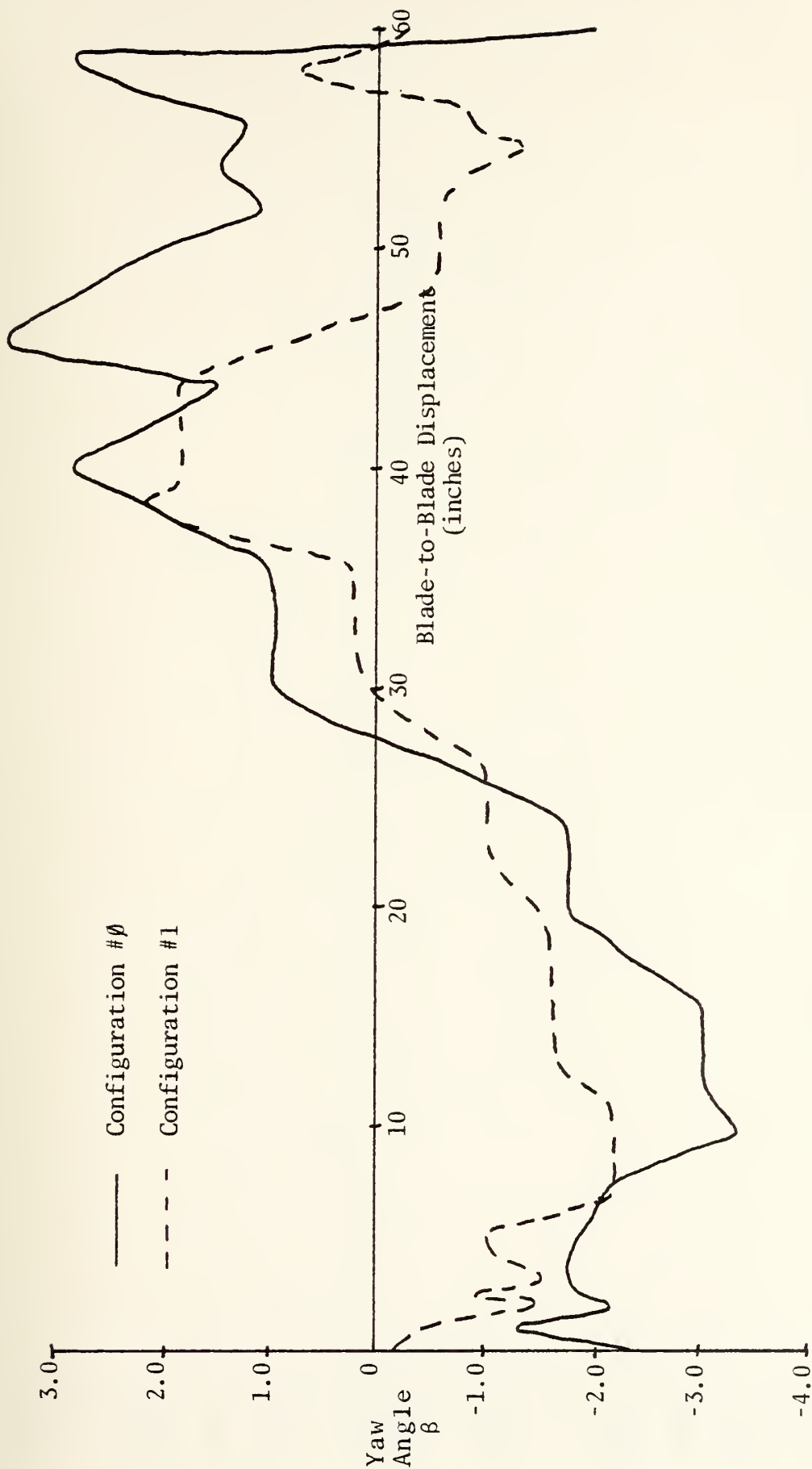


Figure 7e. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 230, 280 ft/sec) - Configuration #0 and #1





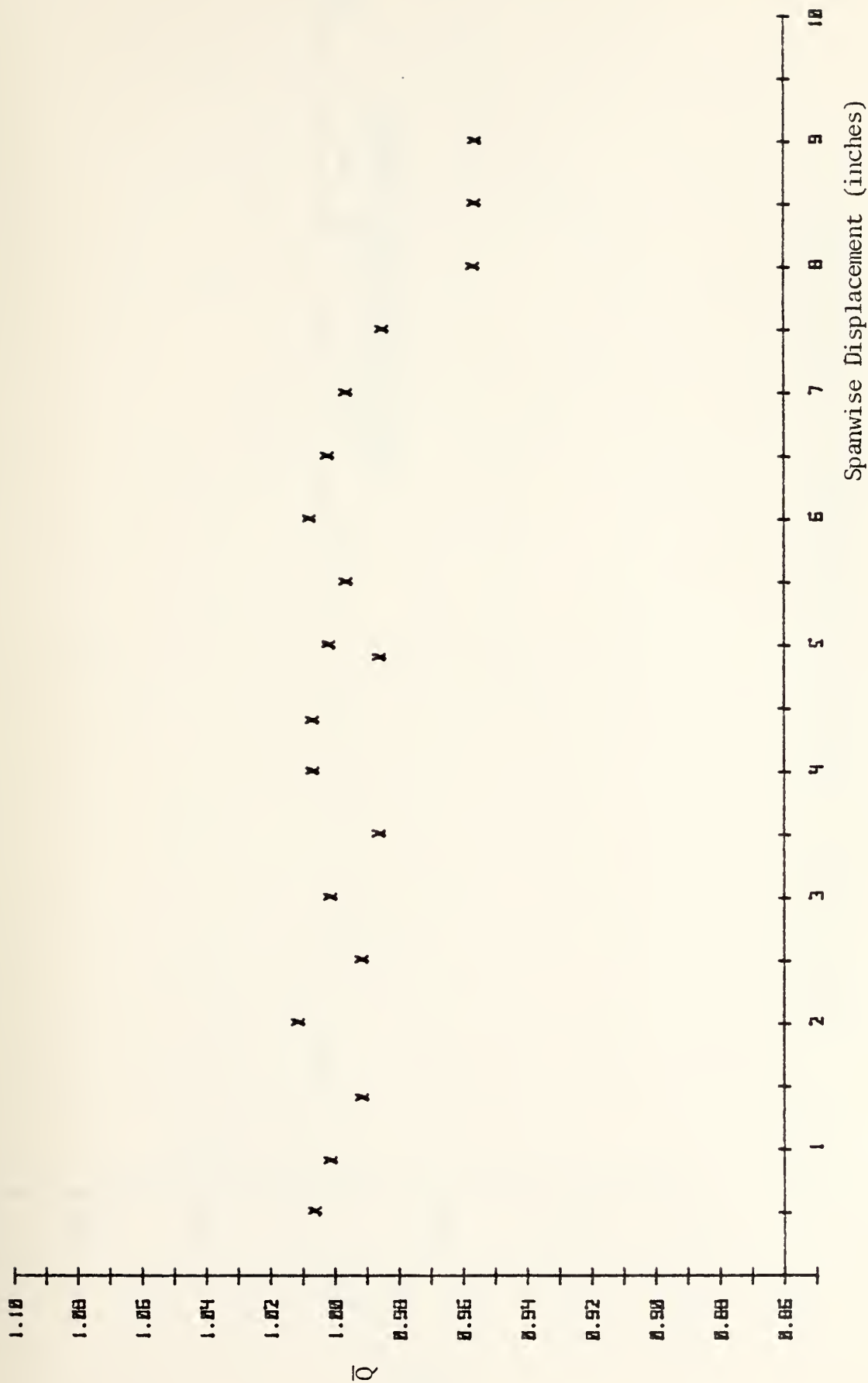


Figure 8a. Spanwise Dynamic Pressure Distribution at the Test Plane  
(at  $X = 30$  in., Velocity = 280 ft/sec) - Configuration #1





Figure 8b. Spanwise Total Pressure Distribution at the Test Plane  
(at  $X = 30$  in., Velocity = 280 ft/sec) - Configuration #1



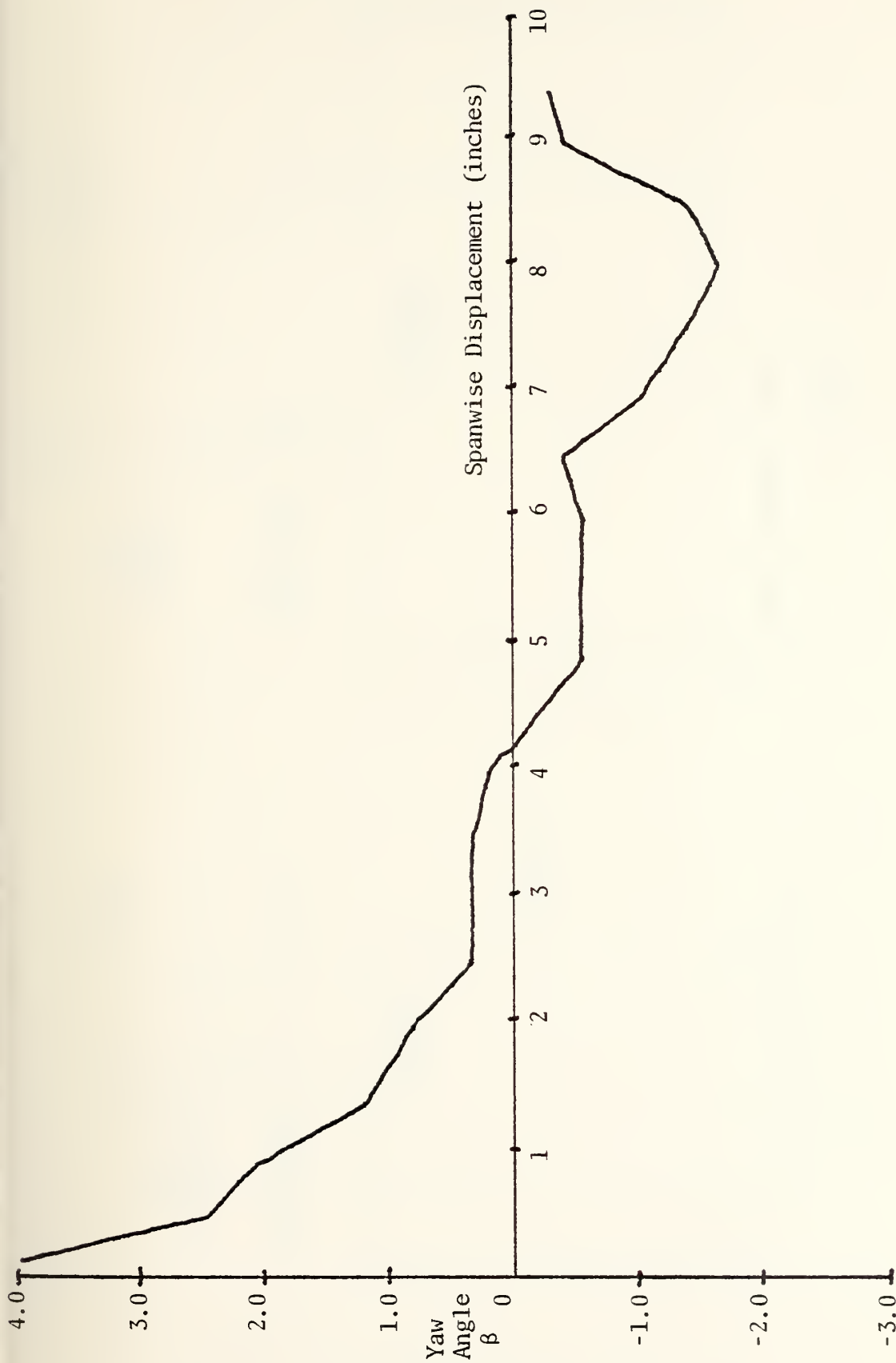


Figure 8c. Spanwise Yaw Angle Distribution at the Test Plane  
(at  $X = 30$  in., Velocity = 280 ft/sec) - Configuration #1



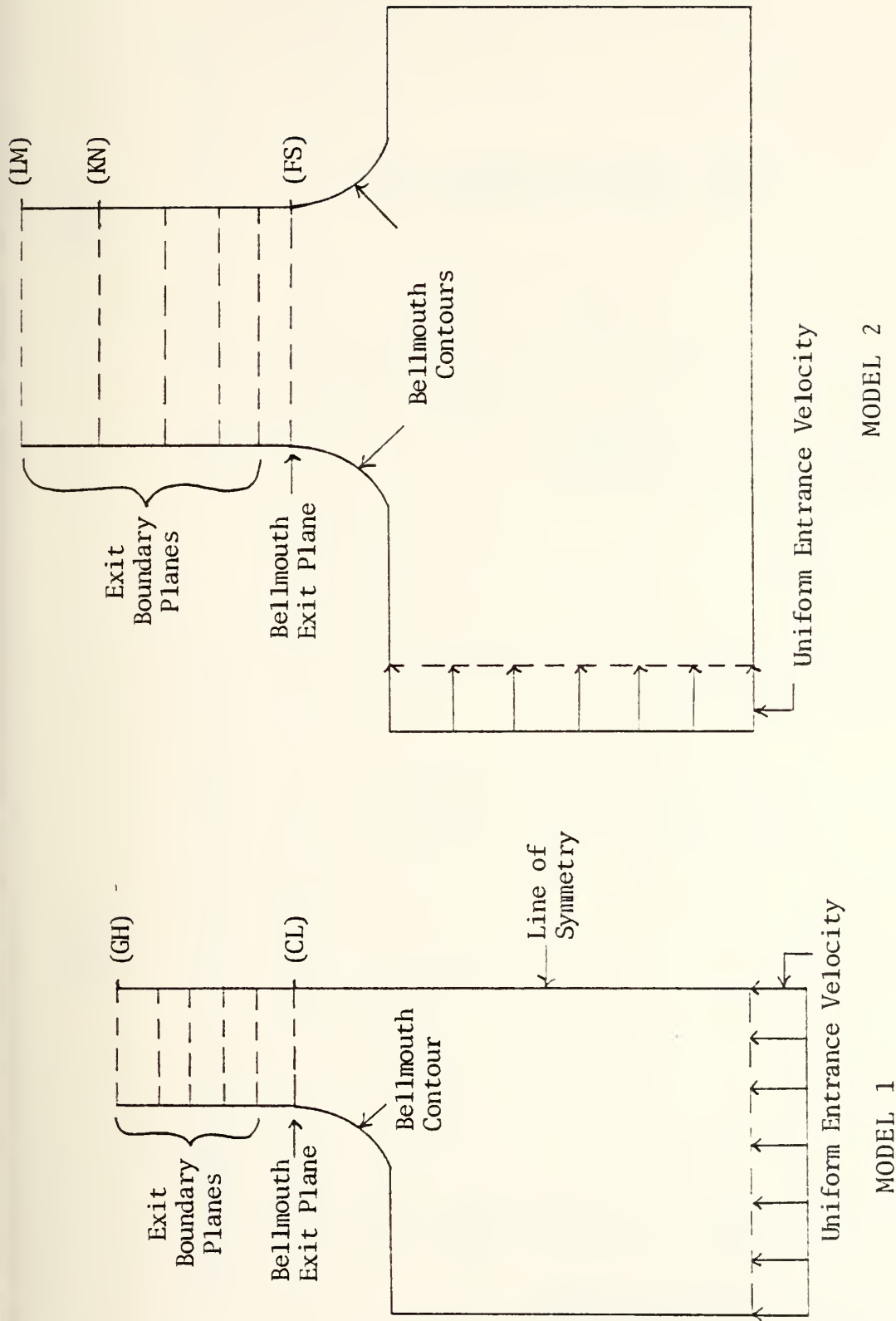


Figure 9. Plenum and Bellmouth Finite Element Flow Analysis Models





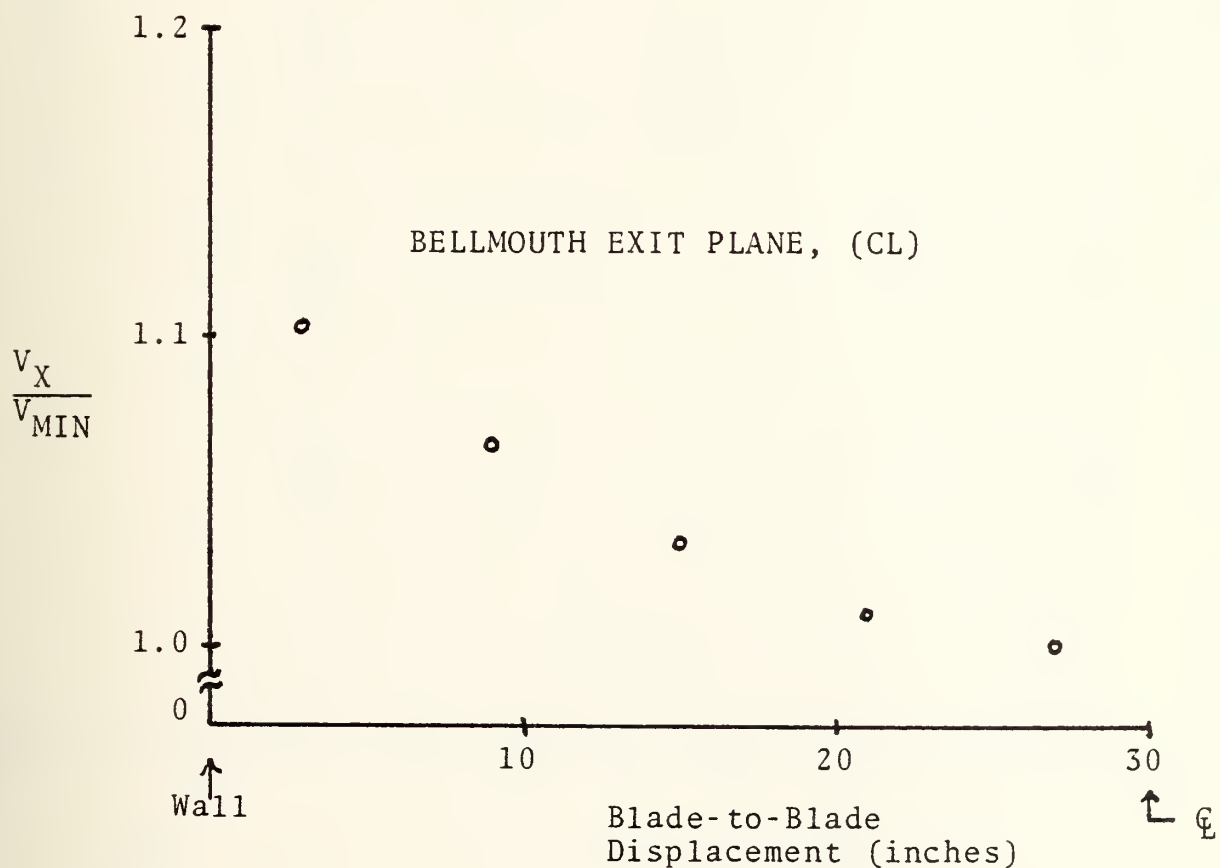
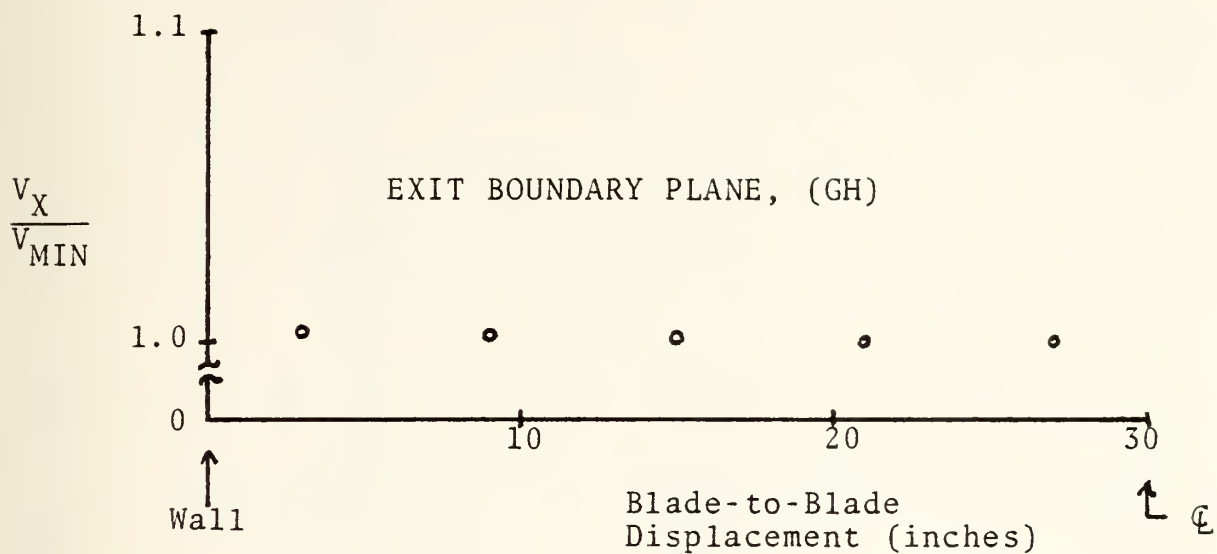


Figure 10. Finite Element Analysis, Velocity Distributions Using Model 1



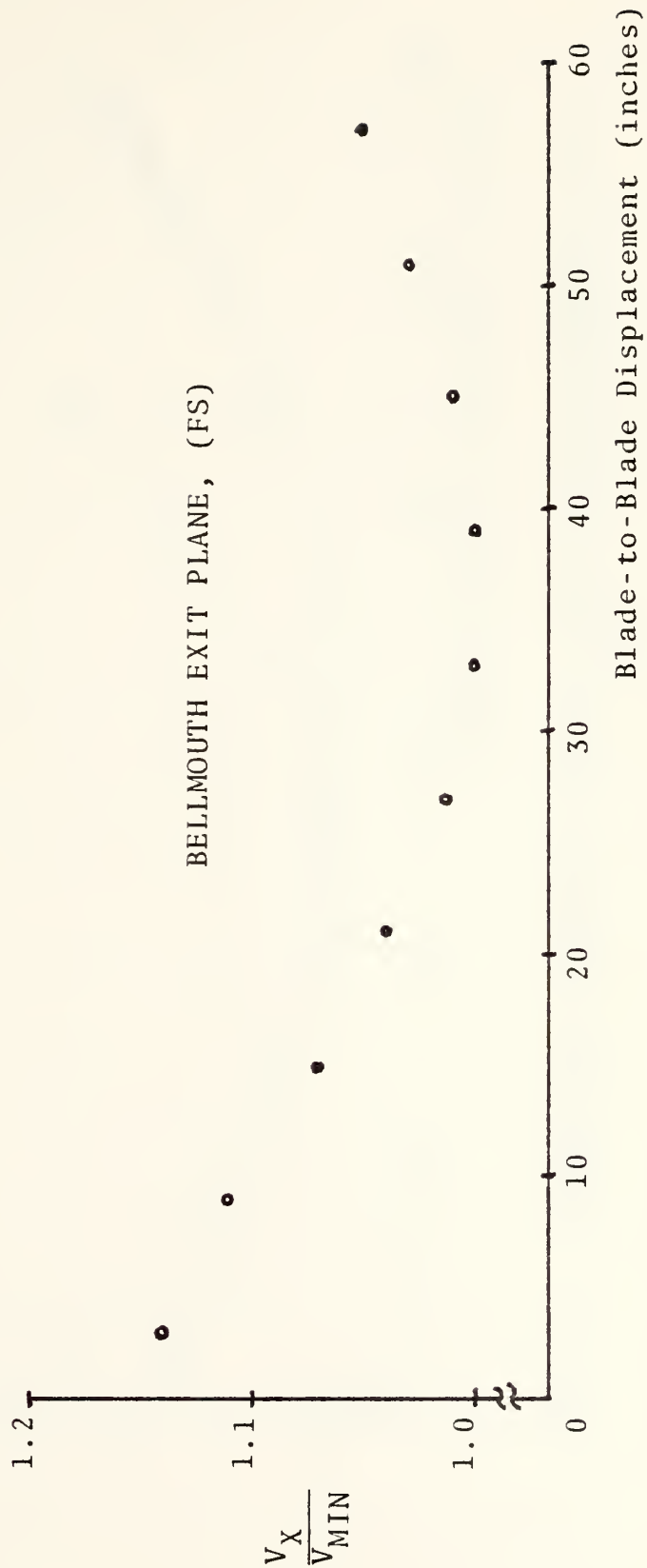
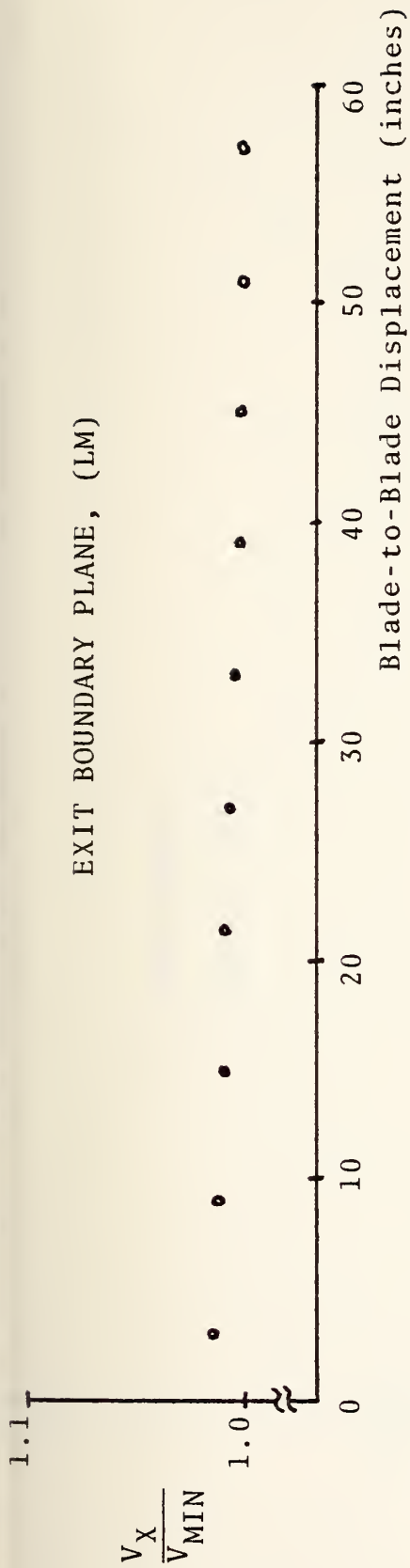


Figure 11. Finite Element Analysis, Velocity Distributions Using Model 2



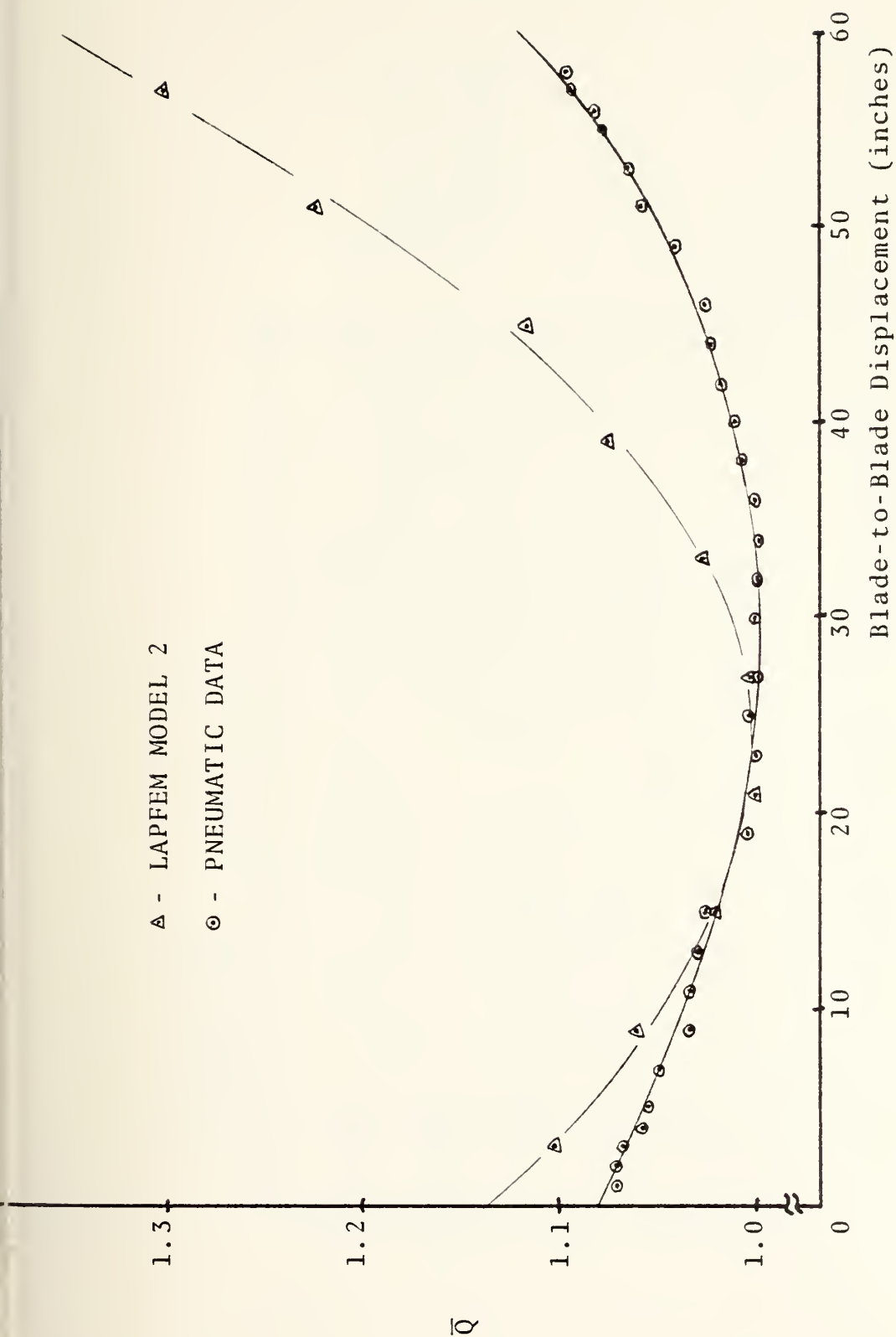


Figure 12. Comparison of Dynamic Pressure Distributions (Pneumatic Data at 50% Span, Velocity = 270 ft/sec) - Configuration #1



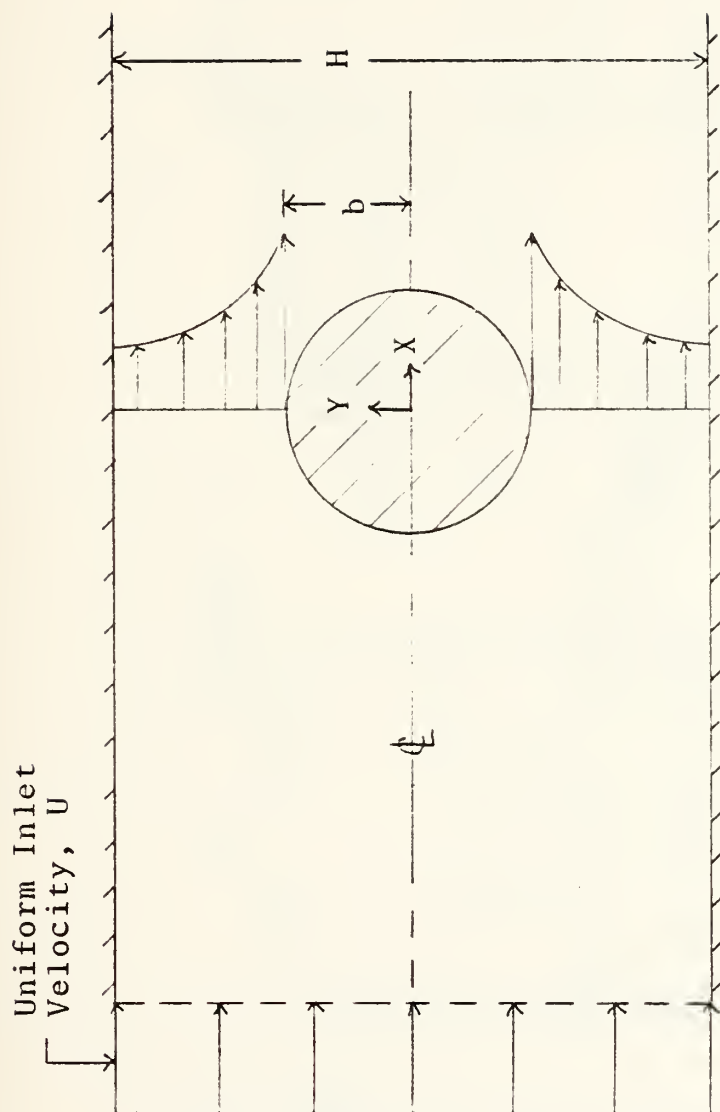


Figure 13. Flow Model Analyzed in Reference 8





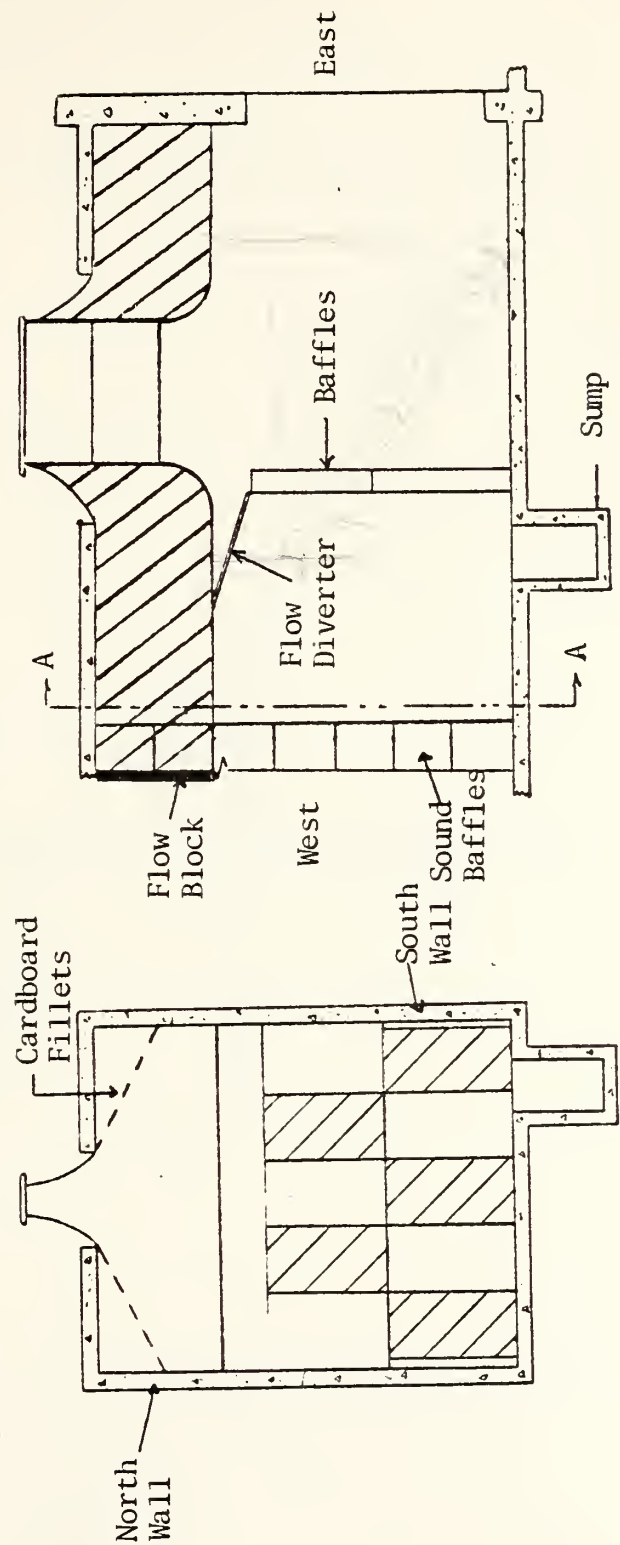


Figure 14. Plenum - Configuration #7 (Side Views)



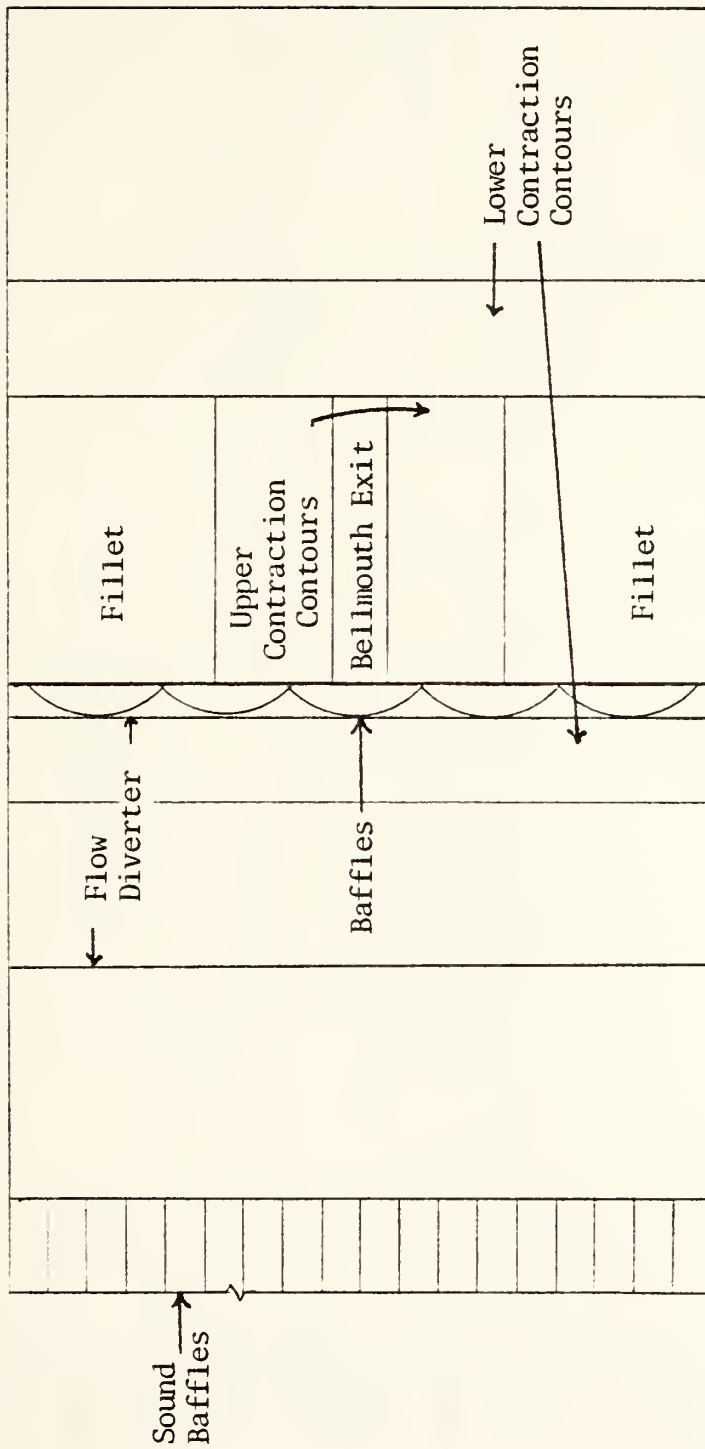


Figure 15. Plenum - Configuration #7 (Plan View)



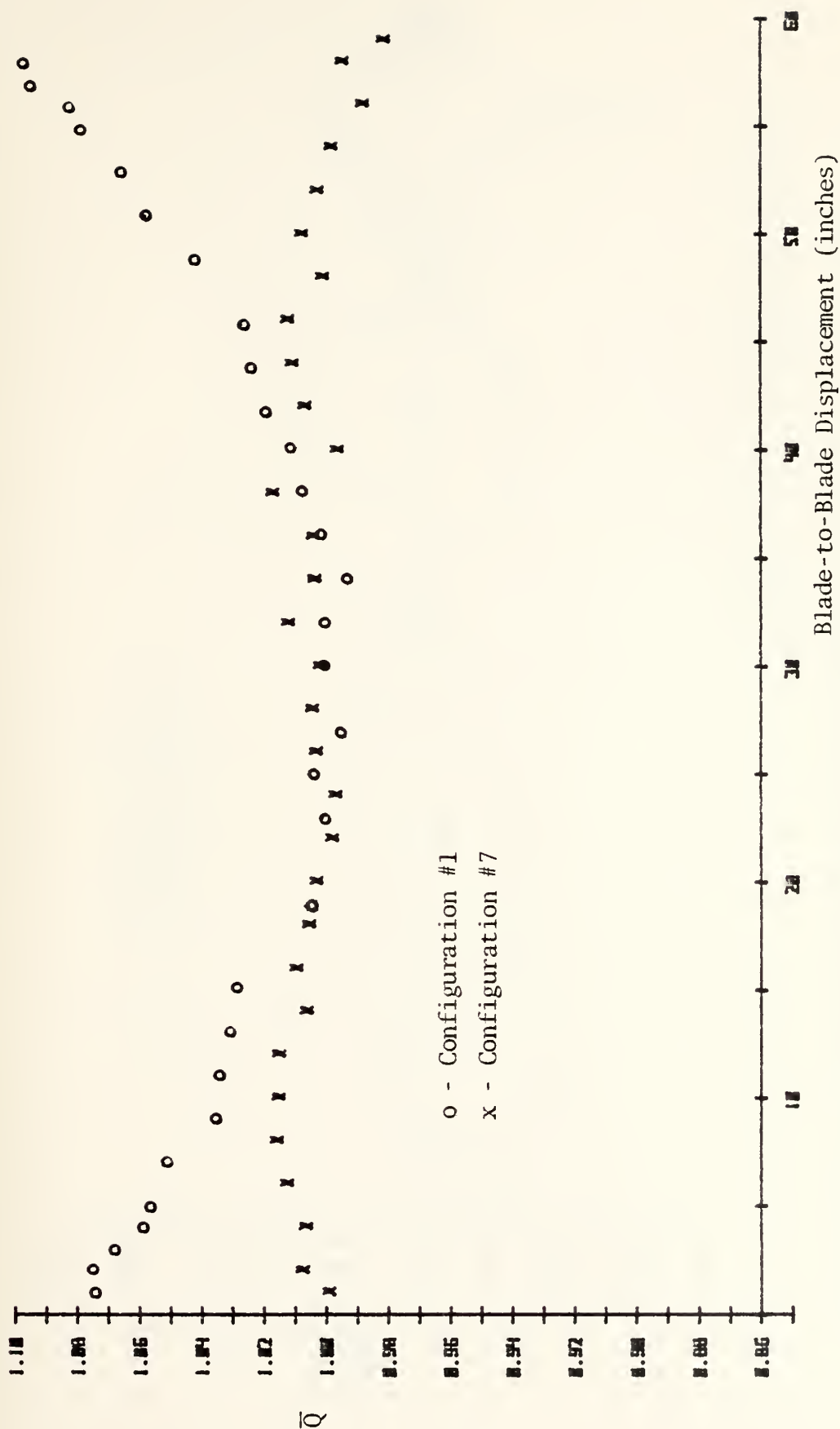


Figure 16a. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane (at 50% Span, Velocity = 270 ft/sec) - Configurations #1 and #7.



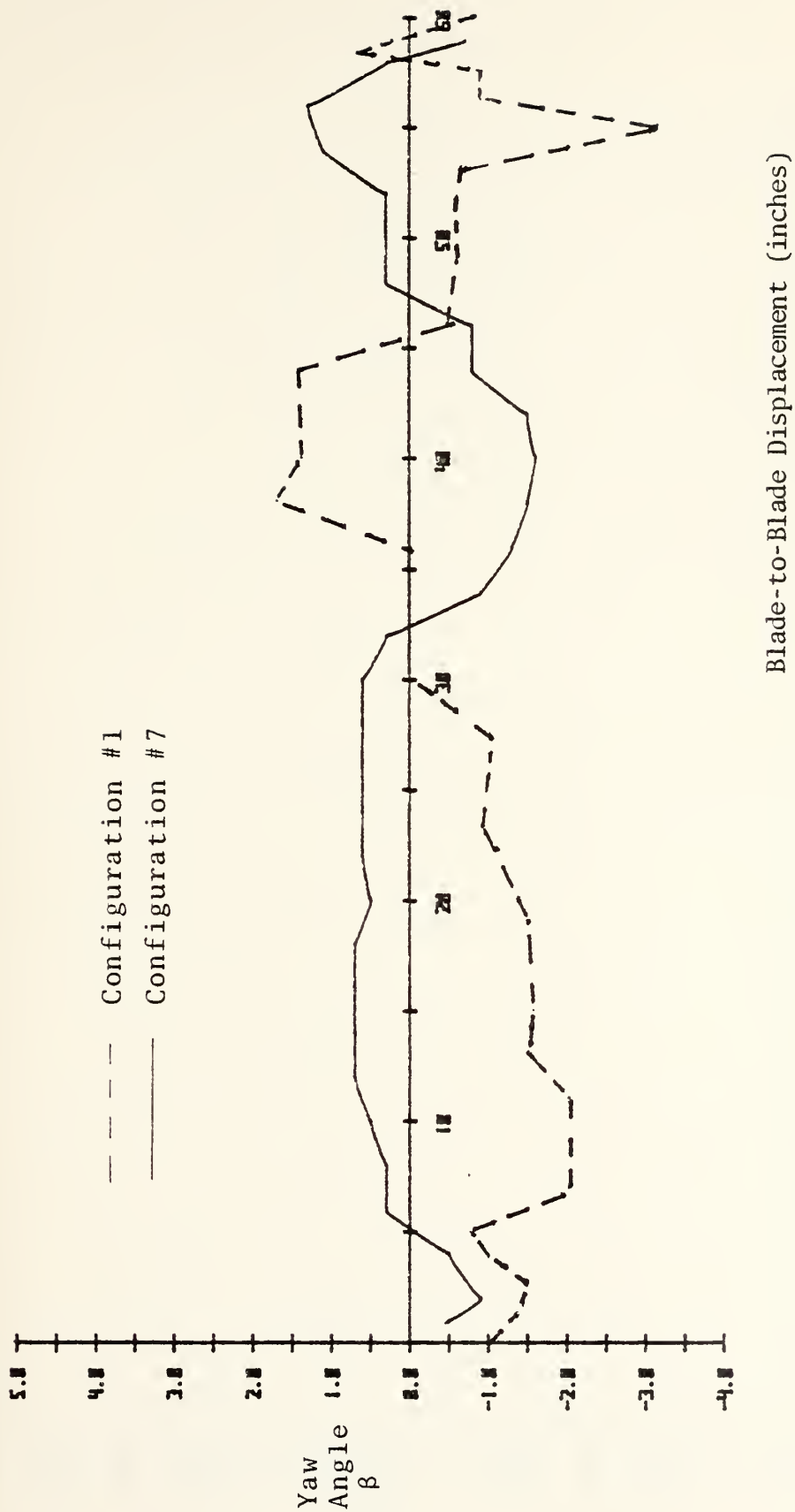


Figure 16b. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 270 ft/sec) - Configurations #1 and #7





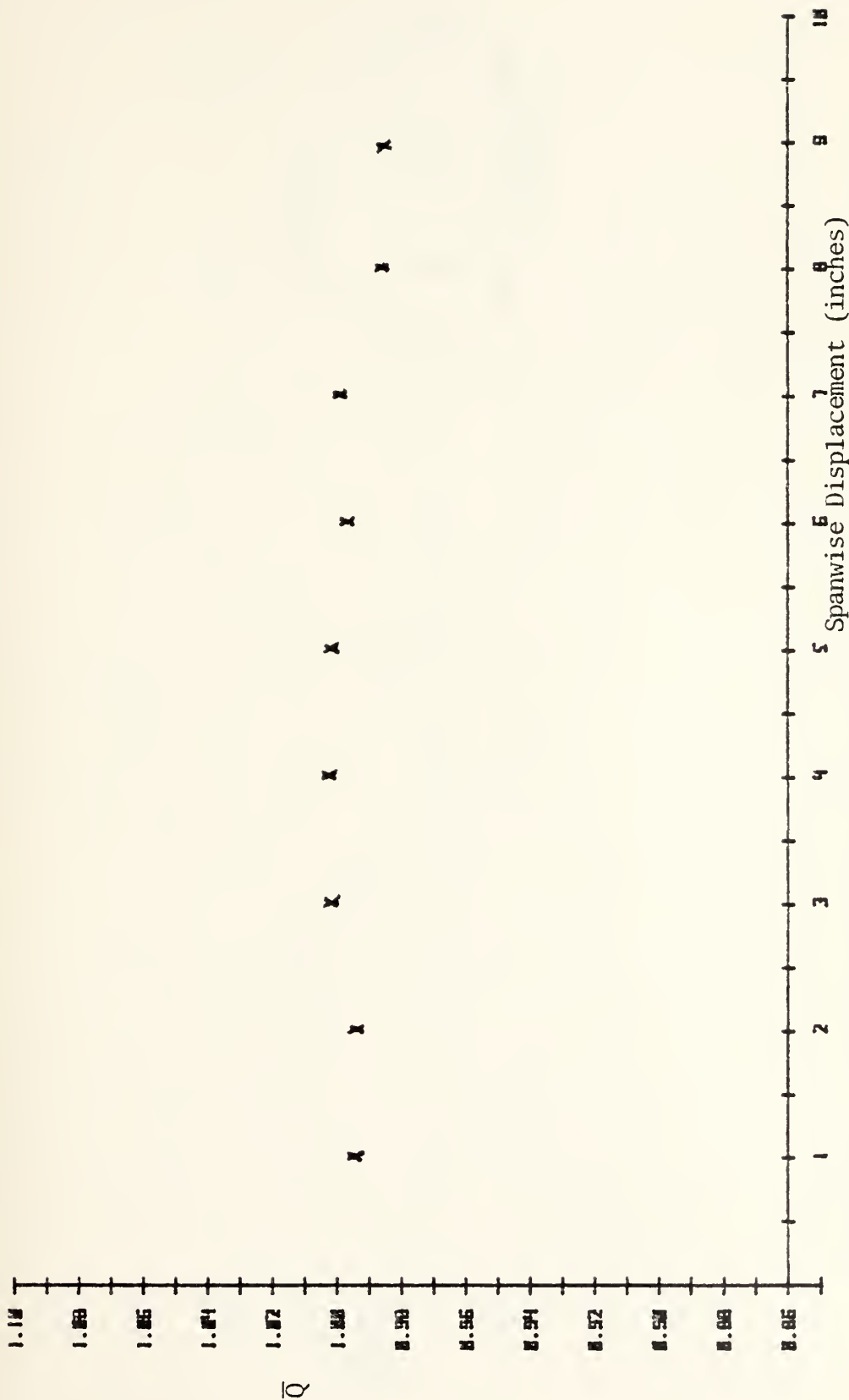


Figure 17a. Spanwise Dynamic Pressure Distribution at the Test Plane  
(at  $X = 50$  in., Velocity = 270 ft/sec) - Configuration #7



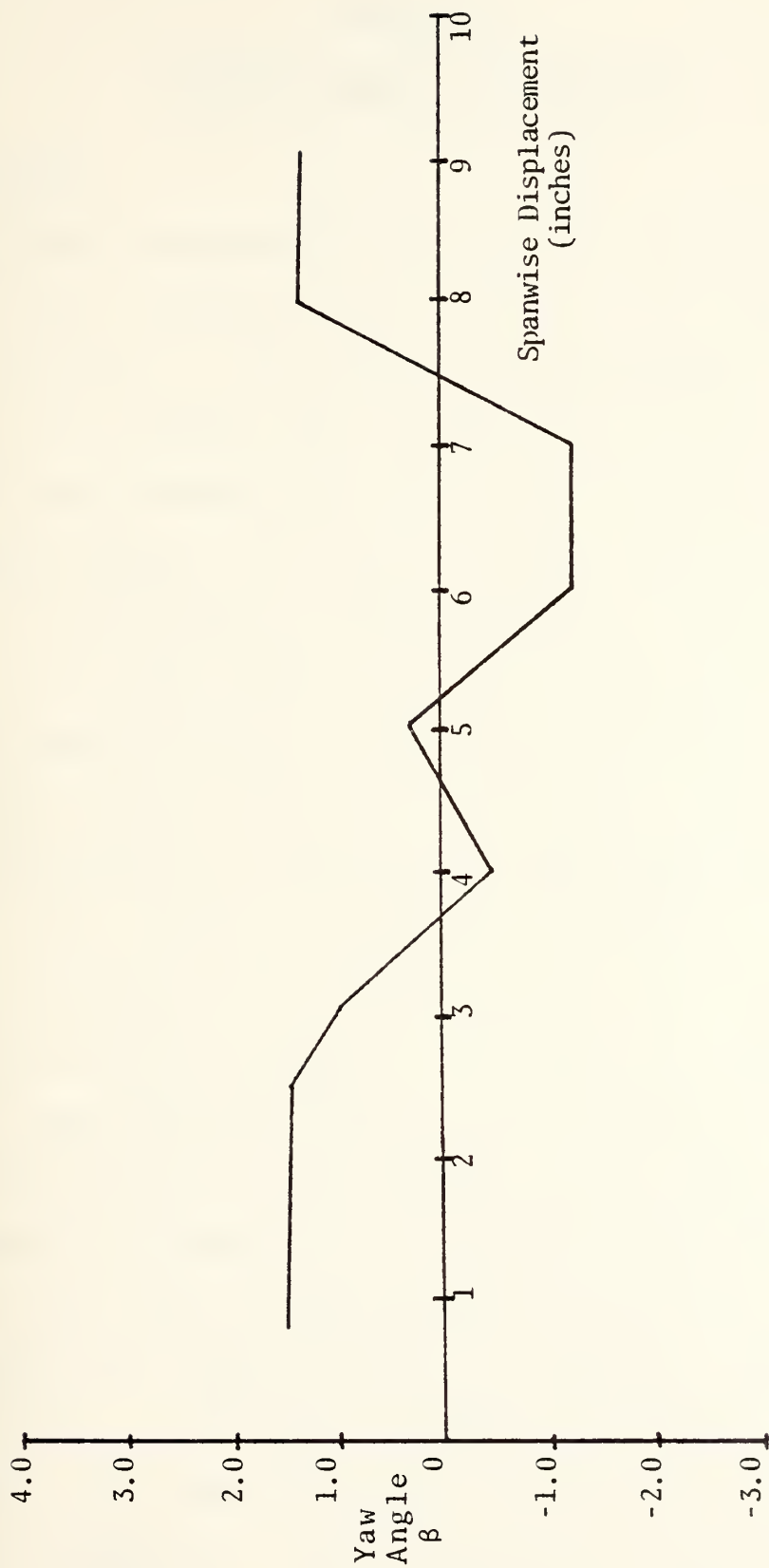


Figure 17b. Spanwise Yaw Angle Distribution at the Test Plane  
(at  $X = 50$  in., Velocity = 270 ft/sec) - Configuration #7



## APPENDIX A

### FINITE ELEMENT FLOW ANALYSIS

#### A.1 Method

A computer program was developed to solve Laplace's Equation (two-dimensional potential flow) in terms of the stream function,  $\psi$ . The purpose of the analysis was to substantiate the premise that the measured velocity distributions in the blade-to-blade directions were a result of the inlet contraction design. The finite element method analysis, FEM, was used to solve this continuum problem. The detailed procedures for an FEM analysis are presented in Ref. 9, with numerous specific fluid flow applications in Ref. 8.

An FEM analysis results in a piecewise approximation of the field variable in the governing flow equation. In the present case, the flow is governed by Laplace's Equation,  $\nabla^2 \psi = 0$ , where  $\psi$  is the field variable. The solution was sought for the flow through a two-dimensional contraction with the same physical shape as the bellmouth of the NPS Rectilinear Cascade. Two different models were used to represent the incoming flow and plenum boundaries.

The first step in applying the FEM was to divide the continuum into a discrete number of elements. Model 1 is shown in Fig. A-1 with the appropriate boundary conditions specified. Only one-half of the contraction required analysis due to symmetry. The discretization of the area ABGH



was carried out with a framework of graphically estimated lines of constant velocity potential,  $\phi = \text{constant}$ , and streamlines of constant  $\psi$ . This method was both accurate and efficient since it gave smaller elements where velocity gradients were high (or boundary geometry changes occurred, as in the contraction contour), and larger elements where gradients were small.

The next step was to assign nodes to the element vertices and to select an interpolation function to represent the variation in  $\psi$  over the element [Ref. 9, Chap. 5]. A linear two-dimensional interpolation polynomial was chosen.

The third step was to determine the matrix equations expressing the properties of a single element. Galerkin's method of weighted residuals was chosen to accomplish this task [Ref. 9, Chap. 4].

The fourth step required an assembly of the sets of matrix equations for each element into a large system matrix. The system matrix was modified at this point to introduce the specified boundary conditions [Ref. 9, Chap. 3]. The boundary conditions were specified in terms of node values of the field variable or its first derivative.

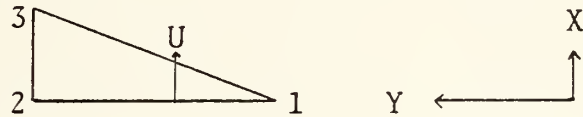
Fifth, the combined set of system equations were solved simultaneously for the nodal values of the field variable. In the present case, a computer library subroutine called "LEQTIF" was used. However, any equivalent linear equation solver could be used here.





Once the values of  $\psi$  at each of the nodes were determined, the final step was to obtain a value for the average velocity between any two adjacent nodes. Since  $\psi$  was assumed to vary linearly between nodes, and the velocity was given by, for example,

$$U = \frac{\partial \psi}{\partial Y} ,$$



the average velocity was calculated using

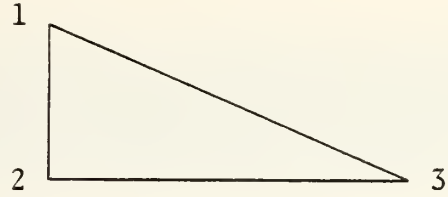
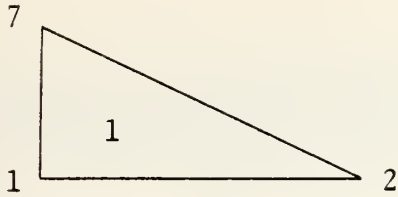
$$U = \frac{\psi_2 - \psi_1}{Y_2 - Y_1} .$$

## A.2 Computer Program

A listing of the Fortran IV FEM program "LAPFEM," written for the NPS IBM 360, is given in Table A-I. Table A-II is a listing of terms not defined in the program comments. The data required for input is given in each "READ" statement. The data used for Model 1 with (GH) as the exit plane (see Fig. 9) are given in Table A-III.

Referring to Fig. A-1, and using for example the element number 1, the correspondence between element and system coordinates was as follows:





System Element

General Element

ICOR (1,1) = 7

ICOR (1,2) = 1

ICOR (1,3) = 2

In general, the rules are as follows:

- 1) Correspond all elements in either a clockwise or an anti-clockwise manner.
- 2) If an element has 1 side on a boundary, then ICOR (I,1) and ICOR (I,2) will be the two nodes of that side.
- 3) If an element has two sides on boundaries, as does element #1, then the correspondance proceeds anti-clockwise along the bounded sides first, as shown above.

The finite element construction for Model 2 is shown in Fig. A-2. The data used when the exit plane was at (LM) are presented in Table A-IV. The results are shown in Fig. 11. As one can see, a larger number of system elements are required when there is no axis of symmetry.

The values output from "LAPFEM" are as follows:

- 1) Values of  $\psi$  at each node point.
- 2) Velocities between specified nodes.



TABLE A-I. "LAPFEM" PROGRAM LISTING

```

C**** THIS PROGRAM SOLVES LAPLACES EQLATION
C**** BY THE FINITE ELEMENT METHOD USING LINEAR TRIANGULAR ELEMENTS
C**** I TYPE=0 FOR A) INTERICR ELEMENTS W. A SINGLE POINT ON THE
C C) DIRICHLET B.C.'S ON ALL BDRY EDGES, C) ELEMENTS WITH
C NEUMANN BOUNDARY VALUE = ZERO.
C I TYPE=1,2 FOR ELEMENTS W. CAUCHY B.C.'S A) ON ECCE 1-2 ONLY B) CN EDGL
C AND 2-3
C I TYPE=3,4 FOR ELEMENTS W. NEUMANN B.C.'S A) ON ECCE1-2, B) ON EDGES 1-3
C NON-ZERO
C *****
C COMMON/MAFO/SYSA(215,215),SYSF(215),X(215),Y(215),ICOR(365,3),
C 1 I TYPE(365),NEL,NSNP,CVC(365),CVN(365)
C DIMENSION WKAREA(500),INBC(125),BCVC(125),BCVN(125),BCVD(125)
C X,ICBC(125),IDBC(125),INVN(20),INVNE(20),VEL(20),IIEI(365),VELE(20)
C *****
C I. MODEL DESCRIPTION
C *****
C ANBC IS THE NO. CF NCDES WITH NEUMANN B.C.(AVG VALUE)
C RCVN IS THE VALUE OF THE NEUMANN B.C.(AVG VALUE)
C NCBC IS THE NO. CF NCDES WITH CAUCHY B.C.
C BCVC IS THE VALUE OF THE CAUCHY B.C.(AVG VALUE)
C NCEC IS THE NO. CF NCDES WITH DIRICHLET B.C.
C BCVD IS THE VALUE OF THE DIRICHLET B.C.
C ZCMEGA IS THE VALUE OF OMEGA IN THE CAUCHY B.C.
C NVN IS THE NO. OF NODES THAT I WANT VELOCITIES BETWEEN.
C *****
C READ(5,10) NEL,NSNP,NCBC,NNBC,NDBC,NVN
C FORMAT(6I5)
C DO 77 I=1,125
C BCVC(I)=0.0
C BCVN(I)=0.0
C BCVD(I)=0.0
C ICBC(I)=0
C INBC(I)=0
C IDBC(I)=0
C CONTINUE
C WRITE(6,11) NEL,NSNP,NCBC,NNBC,NDBC
C FORMAT(/2X,'NO. OF ELEMENTS =',I5,'//2X,'NO. OF SYSTEM NCDES =',
C X,I5,'//2X,'NC. OF CAUCHY B.C. =',I5,'//2X,'NO. OF NEUMANN B.C. =',
C X,I5,'//2X,'NC. OF DIRICHLET B.C. =',I5)
C *****
C ICBC IS THE IDENTITY OF THE CAUCHY B.C. NODE } i.e. NODE NUMBER
C INBC IS THE IDENTITY OF THE NEUMANN B.C. NODE
C IDBC IS THE IDENTITY OF THE DIRICHLET B.C. NODE
C INVN IS THE IDENTITY OF THE NODES THAT I WANT VEL. BETWEEN.
C *****
C 12 FCRMAT(14I5)

```



```

13      FCRMAT(6,F10,3) GO TO 1000
      IF(NCBC.EQ.0) (ICBC(I), I=1,NCBC)
      READ(5,12) (BCVC(I), I=1,NCBC)
      READ(5,13) (ICBC(I), I=1,NCBC)
      WRITE(6,14) (ICBC(I), I=1,NCBC)
      FCRMAT(/2X,'NODE NO.',1,15,5X,'CAUCHY BCUNDARY VALUE = ',F10.2)
14      IF(NCBC.EQ.0) GO TO 2000
10CC      READ(5,12) (INBC(I), I=1,NNBC)
      READ(5,13) (BCVN(I), I=1,NNBC)
      WRITE(6,15) (INBC(I), I=1,NNBC)
      FCRMAT(/2X,'NODE NO.',1,15,5X,'NEUMANN BOUNDARY VALUE = ',F10.2)
15      IF(NDBC.EQ.0) GO TO 3000
20CC      READ(5,12) (IDBC(I), I=1,NDBC)
      READ(5,13) (BCVD(I), I=1,NDBC)
      WRITE(6,16) (IDBC(I), I=1,NDBC)
      FCRMAT(/2X,'NODE NO.',1,15,5X,'DIRICHLET E.V. = ',8X,F10.2)
16      READ(5,27) (INVN(I), I=1,NVN)
      FEAD(5,27) (INVNE(I), I=1,NVN)
27      FCRMAT(14,15)
      WRITE(6,28) (INVN(I), I=1,NVN)
28      FCRMAT(/2X,'VELOCITIES ARE WANTED BETWEEN NOCES ',14I5)
      WRITE(6,74) (INVNE(I), I=1,NVN)
74      FCRMAT(/2X,'EXIT VELOCITIES ARE WANTED BETWEEN NOCES ',
      X14I5)
30CC      READ(5,17) ZOMEGA
17      FCRMAT(F10.2) ZOMEGA
      WRITE(6,18) ZOMEGA
18      FCRMAT(/2X,'THE VALUE OF ZOMEGA IS ',F6.2)
      LOCATION OF SYSTEM NODAL POINTS
      READ(5,20) (X(I),Y(I), I=1,NSNP)
20      FCRMAT(2G10.5)
      WRITE(6,25)
25      FCRMAT(/15X,'NP',15X,'X LOCATION',15X,'Y LOCATION')
      WRITE(6,30) (I,X(I),Y(I), I=1,3)
30      FCRMAT(12X,15,18X,6I2.5,12X,6I2.5)
C *****
C      CORRESPONDENCE TABLE
C *****
      READ(5,35) (IEL(IEL), ICOR(IEL,1), ICOR(IEL,2), ICOR(IEL,3)
      X, ITYPE(IEL), IEL=1,NEL)
35      FCRMAT(5I5)
      WRITE(6,40)
40      FCRMAT(/2X,'ELEMENT',6X,'NP 1',5X,'NP 2',9X,'NP 3',9X,'ITYPE')
      WRITE(6,45) (IEL(IEL), ICOR(IEL,1), I=1,3), ITYPE(IEL), IEL=1,3)
45      FCRMAT(15,8X,15,8X,15,8X,15)
      DC 205 I=1,365
      CVC(I)=0.0
205      CVN(I)=0.0

```









```

54      FORMAT(/2X,'EXIT VELOCITY BETWEEN NODES ',I3,' ANC ',I3,
X' IS',F10.3)
      VEL(I) = (SYSF(K)-SYSF(J))/(Y(J)-Y(K))
53      WRITE(6,53) K,J,VEL(I)
52      FCRMAT(/2X,'VELOCITY BETWEEN NCDES ',I3,' ANC ',I3,' IS',F1C.3)
      CONTINUE
END
C *****
C ***** II. FORMATION OF AU=F
C *****
      SLBROUTINE FORMAF
      COMMON/MAFO/SYSA(215,215),SYSF(215),X(215),Y(215),ICOR(365,3),
      IITYPE(365),NEL,NSNP,CVC(365),CVN(365)
      INITIALIZE A AND F
      DO 100 I=1,NSNP
      SYSF(I)=0.0
      DC 50 J=1,NSNP
      SYSA(I,J)=0.0
      CONTINUE
      CONTINUE
      50
      100
C ***** CALCULATE ELEMENT A AND F CCEFFICIENTS
C *****
      WRITE(6,10)
      10 FORMAT(2X,'ELEM',6X,'B1',13X,'B2',13X,'B3',13X,'C1',13X,'C2',13X
1,'C3',13X,'AREA',)
      DO 500 IEL=1,NEL
      ESTABLISH SYSTEM-ELEMENT CORRESPONDENCE
      II=ICOR(IEI,1)
      JJ=ICOR(IEI,2)
      KK=ICOR(IEI,3)
      CALCULATE ELEMENT GEOMETRY
      B1=Y(JJ)-Y(KK)
      B2=Y(KK)-Y(II)
      B3=Y(II)-Y(JJ)
      C1=X(KK)-X(JJ)
      C2=X(II)-X(KK)
      C3=X(JJ)-X(II)
      DEL=ABS(C2*B1-C1*B2)
      AREA= DEL/2.
      CALCULATE ELEMENT A AND F COEFFICIENTS (ELA,ELF)
      AR=1/(2.*DEL)
      ELA11=((ABS(B1)**2.)+(ABS(C1)**2.))*AR
      ELA12=(B1*B2+C1*C2)*AR
      ELA21=ELA12
      ELA13=(B1*B3+C1*C3)*AR
      ELA31=ELA13
      ELA22=((ABS(B2)**2.)+(ABS(C2)**2.))*AR
      ELA23=(B2*B3+C2*C3)*AR

```







```

C *****
C DISTRIBUTE ELEMENTS COEFFICIENTS INTO SYSTEM A AND F
C *****
150 SYSA(II,II)=SYSA(II,II)+ELA11
    SYSA(II,JJ)=SYSA(II,JJ)+ELA12
    SYSA(II,KK)=SYSA(II,KK)+ELA13
    SYSA(JJ,JJ)=SYSA(JJ,JJ)+ELA22
    SYSA(JJ,II)=SYSA(II,JJ)
    SYSA(JJ,KK)=SYSA(JJ,KK)+ELA23
    SYSA(KK,II)=SYSA(II,KK)
    SYSA(KK,JJ)=SYSA(JJ,KK)+ELA33
    SYSF(II)=SYSF(II)+ELF1
    SYSF(JJ)=SYSF(JJ)+ELF2
    SYSF(KK)=SYSF(KK)+ELF3
500 CONTINUE
    RETURN
    END
LAP02410
LAP02420
LAP02430
LAP02440
LAP02450
LAP02460
LAP02470
LAP02480
LAP02490
LAP02500
LAP02510
LAP02520
LAP02530
LAP02540
LAP02550
LAP02560
LAP02570
LAP02580

```





TABLE A-II

DEFINITION OF TERMS NOT DEFINED IN  
THE "LAPFEM" PROGRAM COMMENTS

ITYPE	Specifies type of boundary condition
NEL	Total number of system elements
NSNP	Total number of system nodal points
IIEI	Number of element
ICOR	A term to correspond single element node numbers (1,2,3), with system node numbers (1,2,3,4,5,..., NEL)
A	System matrix of coefficients of $\psi$
U	Column vector of unknown values of $\psi$ ( $\psi_1, \psi_2, \dots, \psi_{NSNP}$ )
F	Column vector of external node values
SYS A(I,II)	System A matrix
SYS F(I,II)	System F matrix
EL A I II	Elemental A matrix
EL F II	Elemental F matrix
AREA	Element area
I,II	Matrix row and column respectively
X,Y	X and Y locations of each node



TABLE A-III. DATA FOR MODEL 1 ANALYSIS

05/27/79 17.40.49

FILE : LAPDAT FT05FU01 P1

NAVAL POSTGRADUATE SCHOOL

141	91	0	0	35	6									
1	2	3	4	5	6	7	12	13	18	19	24	25	26	
31	32	37	38	43	44	49	50	55	56	61	62	67	68	
73	74	79	80	85	86	91								
80.		64.		48.		32.		16.		0.0				
80.		0.0		80.		0.0		80.		0.0				
80.		80.		0.0		80.		0.0		80.				
0.0		80.		0.0		80.		0.0		80.				
0.0		80.		0.0		80.		0.0		80.				
0.0		80.0		0.0		80.0		0.0		80.0				
91	50	89	88	87	86									
91	90	89	88	87	86									
0.0														
0.0		160.0												
0.0		128.												
0.0		96.												
0.0		64.												
0.0		32.												
0.0		0.0												
27.5		160.												
28.		127.5												
28.5		96.												
29.		63.5												
29.5		31.5												
30.		0.0												
58.		160.												
58.		127.												
58.		95.												
57.5		62.5												
56.5		30.5												
55.5		0.0												
100.		160.												
95.		125.5												
85.5		92.												
79.		62.												
76.		30.												
75.		0.0												
140.		160.												
152.5		120.												
125.		104.												
110.		81.												
99.		55.												
94.		26.												
93.5		0.0												
160.		90.												
145.		81.												
131.		66.5												
117.5		47.5												
109.5		23.												
107.5		0.0												
166.5		76.5												
155.		68.												
144.		56.												
134.		38.5												
128.		18.5												
125.5		0.0												
173.		67.5												
164.		59.5												
155.5		47.												
149.		32.												
144.		16.												
143.5		0.0												
179.0		61.5												
172.		52.												
166.		40.5												
162.		28.												
159.5		14.												
158.		0.0												
185.		57.												
179.		47.												



172.	36.			
172.5	25.			
170.5	13.0			
170.	0.0			
190.	54.			
180.	44.			
185.	33.			
184.	23.0			
183.	12.			
183.	0.0			
195.	52.0			
193.5	42.5			
193.	32.			
192.5	22.			
192.5	11.5			
192.5	0.0			
201.	51.			
200.5	41.5			
200.5	32.			
200.5	22.			
200.5	11.5			
200.5	0.0			
208.	50.25			
208.	41.25			
208.	31.5			
208.	22.0			
208.	11.0			
208.	0.0			
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217.	31.5			
217.	21.5			
217.	11.0			
217.	0.0			
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3	2	3	8	0
4	8	3	9	0
5	3	4	9	0
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7	4	5	10	0
8	10	5	11	0
9	5	6	11	0
10	6	12	11	0
11	13	7	8	0
12	13	8	14	0
13	14	8	9	0
14	14	9	15	0
15	15	9	10	0
16	15	10	16	0
17	16	10	11	0
18	16	11	17	0
19	17	11	12	0
20	12	18	17	0
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24	20	15	21	0
25	21	15	16	0
26	21	16	22	0
27	22	16	17	0
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35	28	21	22	0
36	28	22	29	0



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42	32	26	27	0
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44	33	27	28	0
45	33	28	34	0
46	34	23	29	0
47	34	29	35	0
48	35	29	30	0
49	35	30	36	0
50	36	30	31	0
51	31	37	36	0
52	38	32	33	0
53	38	33	39	0
54	39	33	34	0
55	39	34	40	0
56	40	34	35	0
57	40	35	41	0
58	41	35	36	0
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68	47	41	42	0
69	47	42	48	0
70	48	42	43	0
71	43	49	48	0
72	50	44	45	0
73	50	45	51	0
74	51	45	46	0
75	51	46	52	0
76	52	46	47	0
77	52	47	53	0
78	53	47	48	0
79	53	48	54	0
80	54	48	49	0
81	54	49	55	0
82	56	50	51	0
83	56	51	57	0
84	57	51	52	0
85	57	52	58	0
86	58	52	53	0
87	58	53	59	0
88	59	53	54	0
89	59	54	60	0
90	60	54	55	0
91	55	61	60	0
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93	62	57	63	0
94	63	57	58	0
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98	65	59	60	0
99	65	60	66	0
100	66	60	61	0
101	61	67	66	0
102	68	62	63	0
103	68	63	69	0
104	69	63	64	0
105	69	64	70	0
106	70	64	65	0





107	70	65	71	0
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109	71	66	72	0
110	72	66	67	0
111	67	73	72	0
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113	75	74	69	0
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116	76	70	71	0
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154	99	93	94	0
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168	107	101	102	0
169	108	107	102	0
170	108	102	103	0
171	103	109	108	0
172	110	104	105	0
173	111	110	105	0
174	111	105	106	0
175	112	111	106	0
176	112	106	107	0



FILE : LAFDAT FT05F001 P1

NAVAL POSTGRADUATE SCHOOL

177	113	112	107	0
178	113	107	108	0
179	114	113	108	0
180	114	108	109	0
181	109	115	114	0



TABLE A-IV. DATA FOR MODEL 2 ANALYSIS

[illegible]



65.  
65.  
70.  
70.  
74.  
90.  
92.  
92.  
92.  
96.  
104.  
115.  
115.  
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117.  
125.  
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138.  
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163.  
154.  
184.  
184.  
185.  
174.  
180.  
172.  
172.  
182.  
183.  
185.  
207.  
207.

0  
35.  
75.  
107.  
153.  
190.  
237.  
0.  
35.  
75.  
108.  
150.  
182.  
0.  
35.  
75.  
107.  
145.  
215.  
275.  
318.  
0.  
35.  
75.  
108.  
188.  
313.  
365.  
420.  
0.  
35.  
75.  
109.  
151.  
0.  
35.  
75.  
110.  
141.  
190.  
219.  
251.  
280.  
342.  
420.  
0.  
35.  
75.





107.  
 157.  
 170.  
 188.  
 207.  
 228.  
 250.  
 271.  
 297.  
 310.  
 355.  
 420.  
 0.0  
 35.  
 75.  
 108.  
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 173.  
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 174.  
 185.  
 199.  
 213.  
 230.  
 244.  
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 284.  
 294.  
 175.  
 184.  
 193.  
 204.  
 217.  
 230.  
 242.  
 254.  
 266.  
 276.  
 285.

68.  
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 111.  
 199.  
 200.  
 207.  
 194.  
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 246.  
 251.  
 254.  
 252.  
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 248.  
 249.  
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 255.  
 258.



270.	179.
2269.	188.
2268.	197.
2267.	207.
2266.	218.
2265.	230.
2264.	242.
2263.	253.
2262.	263.
2261.	272.
2270.	281.
2282.	180.
2282.	189.
2281.	198.
2281.	208.
2281.	219.
2281.	230.
2281.	241.
2281.	252.
2282.	262.
2282.	271.
2290.	280.
2290.	189.
2290.	198.
2290.	208.
2290.	219.
2290.	230.
2290.	241.
2290.	251.
2290.	261.
2290.	271.
2300.	280.
2300.	189.
2300.	199.
2300.	208.
2300.	219.
2300.	230.
2300.	241.
2300.	251.
2300.	261.
2300.	271.
2300.	280.
2315.	180.
2315.	190.
2315.	200.
2315.	210.



315.	0.
315.	230.
315.	240.
315.	250.
315.	260.
315.	270.
330.	180.
330.	190.
330.	200.
330.	210.
330.	220.
330.	230.
330.	240.
330.	250.
330.	260.
330.	270.
350.	180.
350.	190.
350.	200.
350.	210.
350.	220.
350.	230.
350.	240.
350.	250.
350.	260.
350.	270.
370.	180.
370.	190.
370.	200.
370.	210.
370.	220.
370.	230.
376.	240.
370.	250.
370.	260.
370.	270.
370.	280.













64 65 70 72 71 57 72 88 75 62 67 63 68 64 69 70 67 78 68 79 69 80 97 81 82 71 84 72 85 73 86 74 87 88 75 90 76 78 78 79 79 80 80 97 81 98

56 56 65 65 65 65 57 58 61 62 62 63 63 64 66 67 67 68 68 69 69 70 82 71 72 72 73 73 74 74 88 75 77 92 78 79 79 94 95 80 97 81

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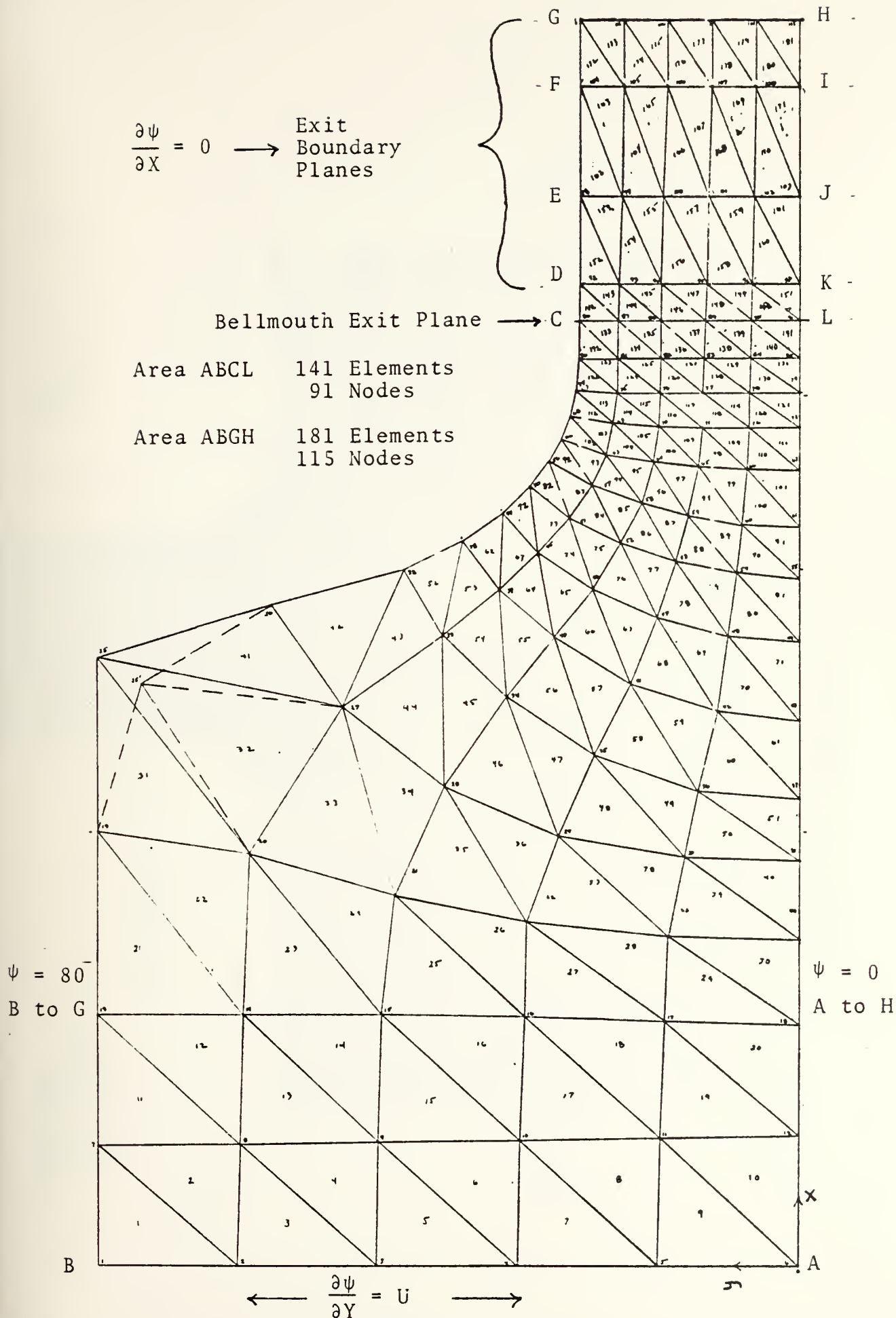


$$\frac{\partial \psi}{\partial X} = 0 \rightarrow \text{Exit Boundary Planes}$$

Bellmouth Exit Plane → C

Area ABCL 141 Elements  
91 Nodes

Area ABGH 181 Elements  
115 Nodes







Total Area  
365 Elements  
215 Nodes

Bellmouth  
Exit Plane →

Exit  
Boundary  
Planes ←  $\frac{\partial \psi}{\partial} = 0$

$$\frac{\partial \psi}{\partial X} = U$$

$$\psi = 0$$

$$Z \text{ to } V$$

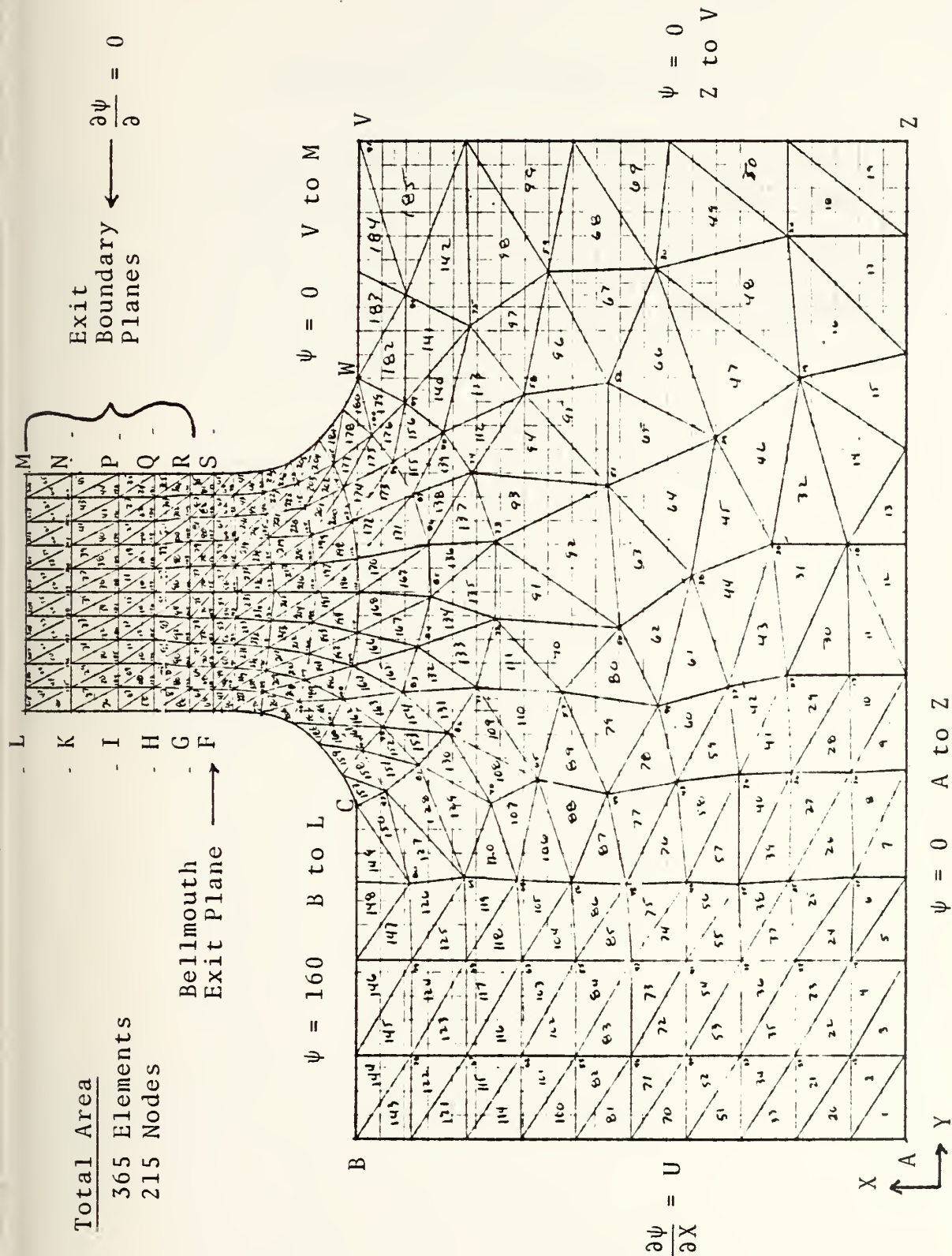


Figure A-2. FEM Model 2



## APPENDIX B

### PRELIMINARY TEST PROBE DATA

The following figures are data from tests of Configuration #0 and Configuration #1. The results showed unacceptable variations in total and dynamic pressure in both the blade-to-blade and spanwise directions. The yaw angles were also outside the required bounds.



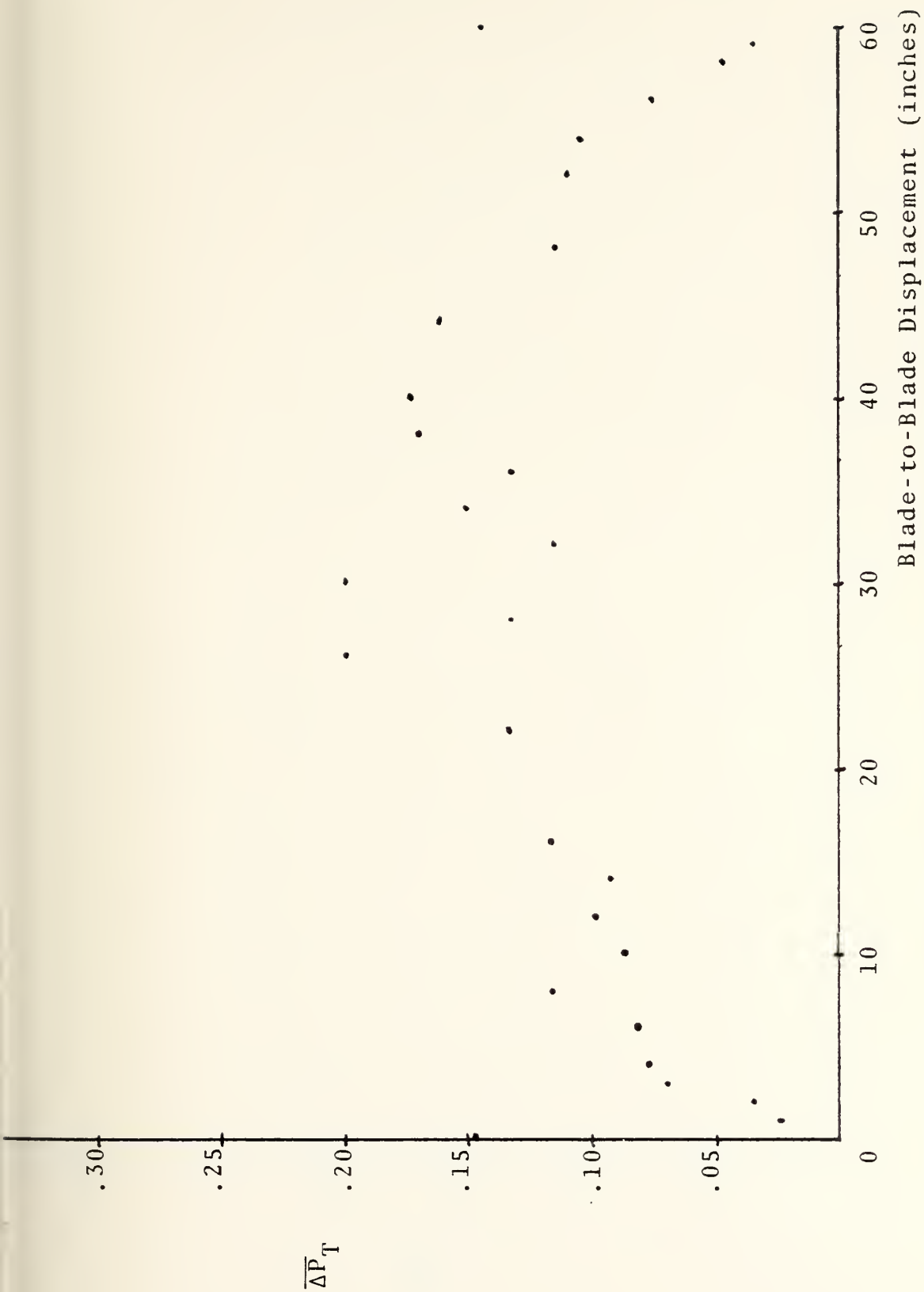


Figure B-1a. Blade-to-Blade Total Pressure Distribution at the Test Plane  
(at 10% Span, Velocity = 230 ft/sec) - Configuration #0



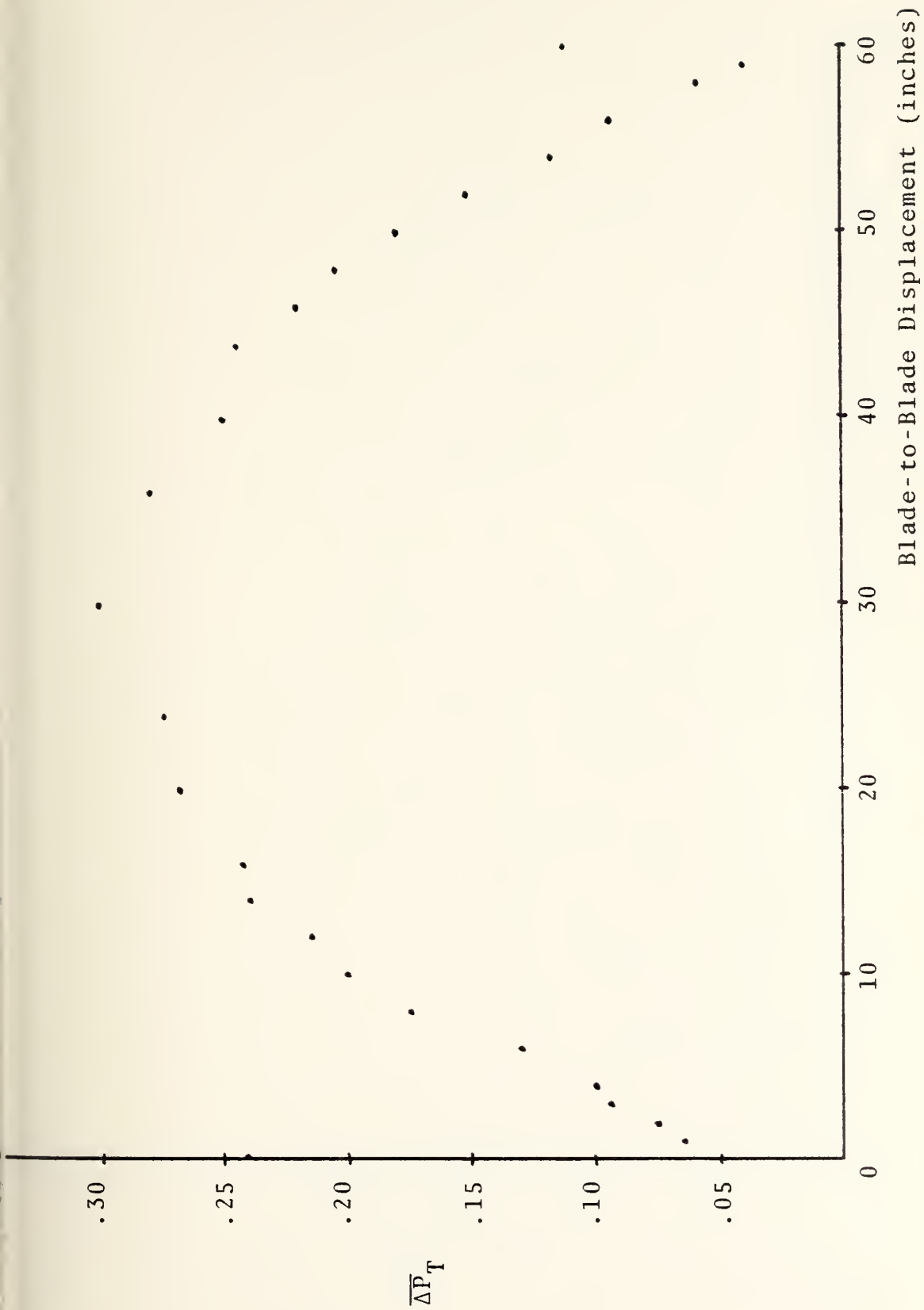


Figure B-1b. Blade-to-Blade Total Pressure Distribution at the Test Plane  
(at 30% Span, Velocity = 230 ft/sec) - Configuration #0





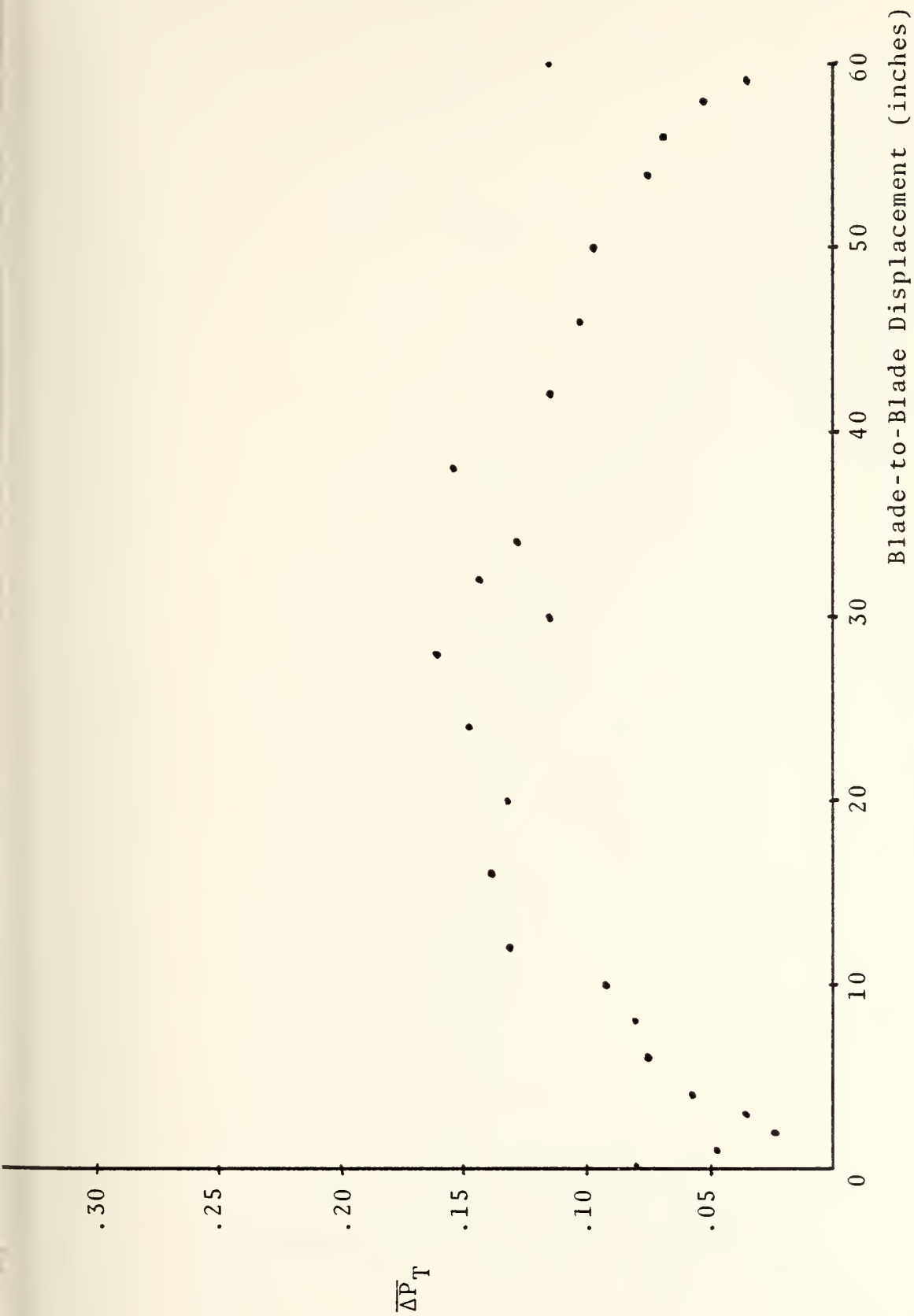


Figure B-1c. Blade-to-Blade Total Pressure Distribution at the Test Plane  
(at 90% Span, Velocity = 230 ft/sec) - Configuration #0



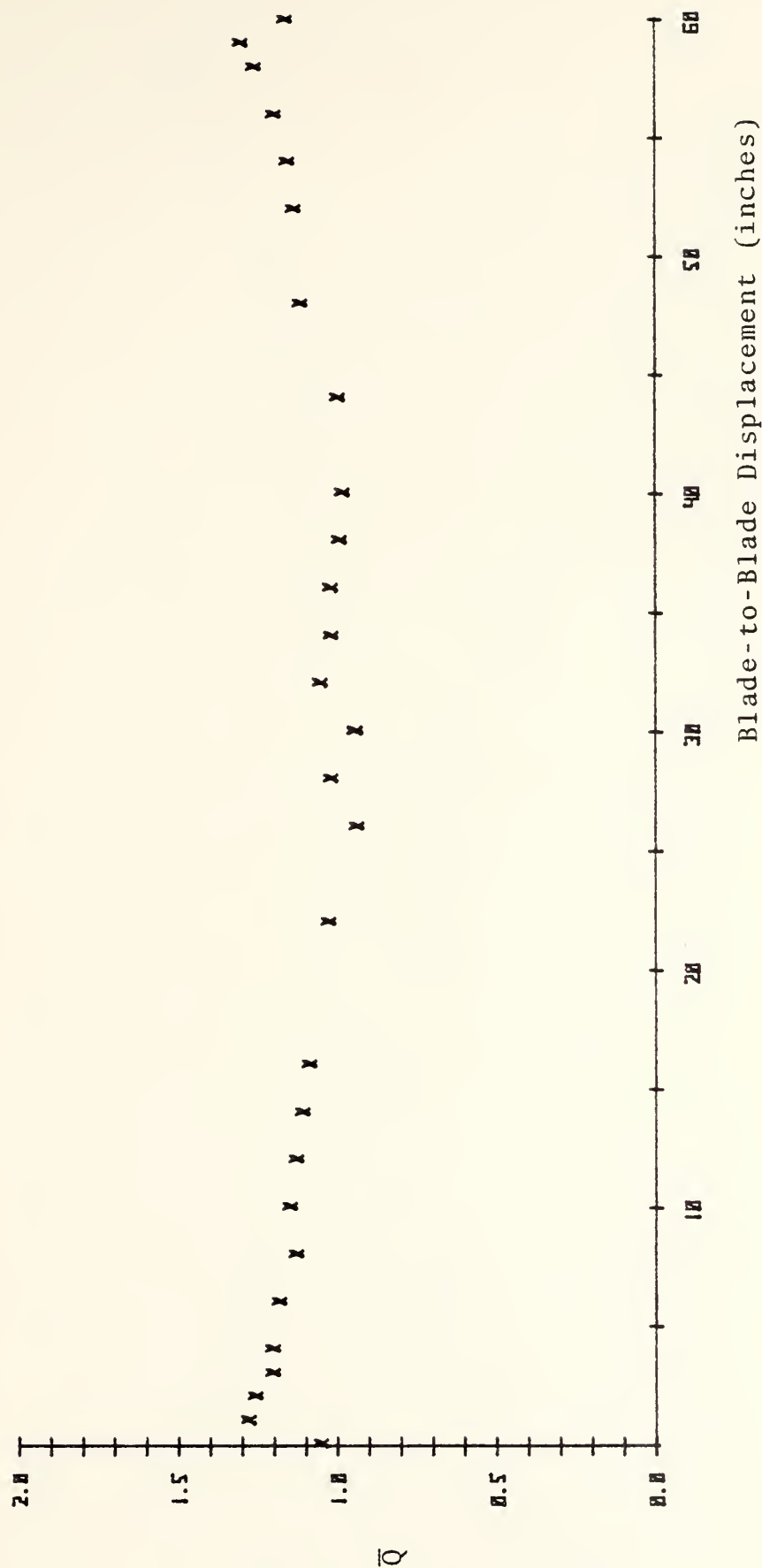


Figure B-2a. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 10% Span, Velocity = 230 ft/sec) - Configuration #0





Figure B-2b. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 30% Span, Velocity = 230 ft/sec) - Configuration #0



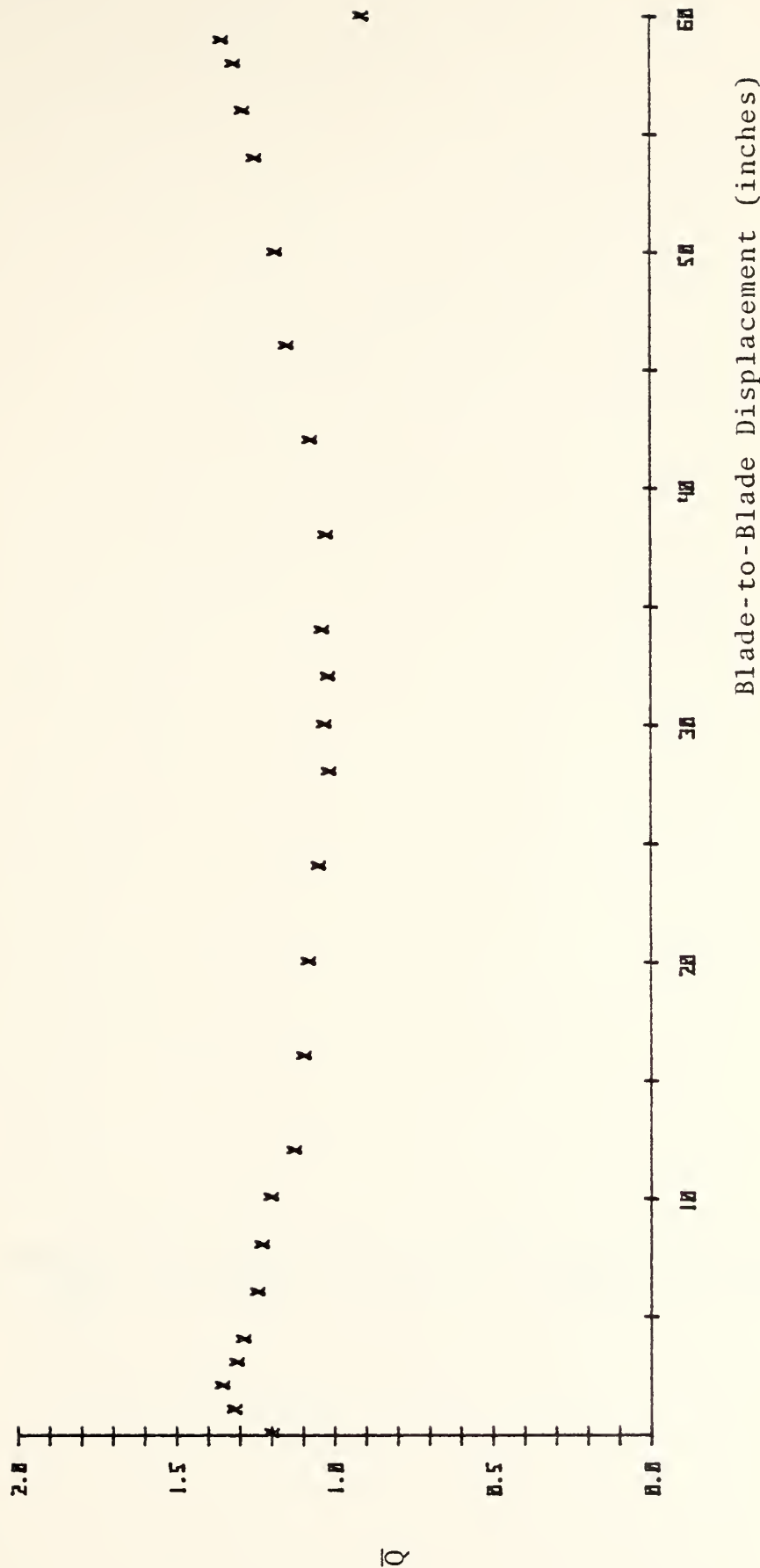


Figure B-2c. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 90% Span, Velocity = 230 ft/sec) - Configuration #0





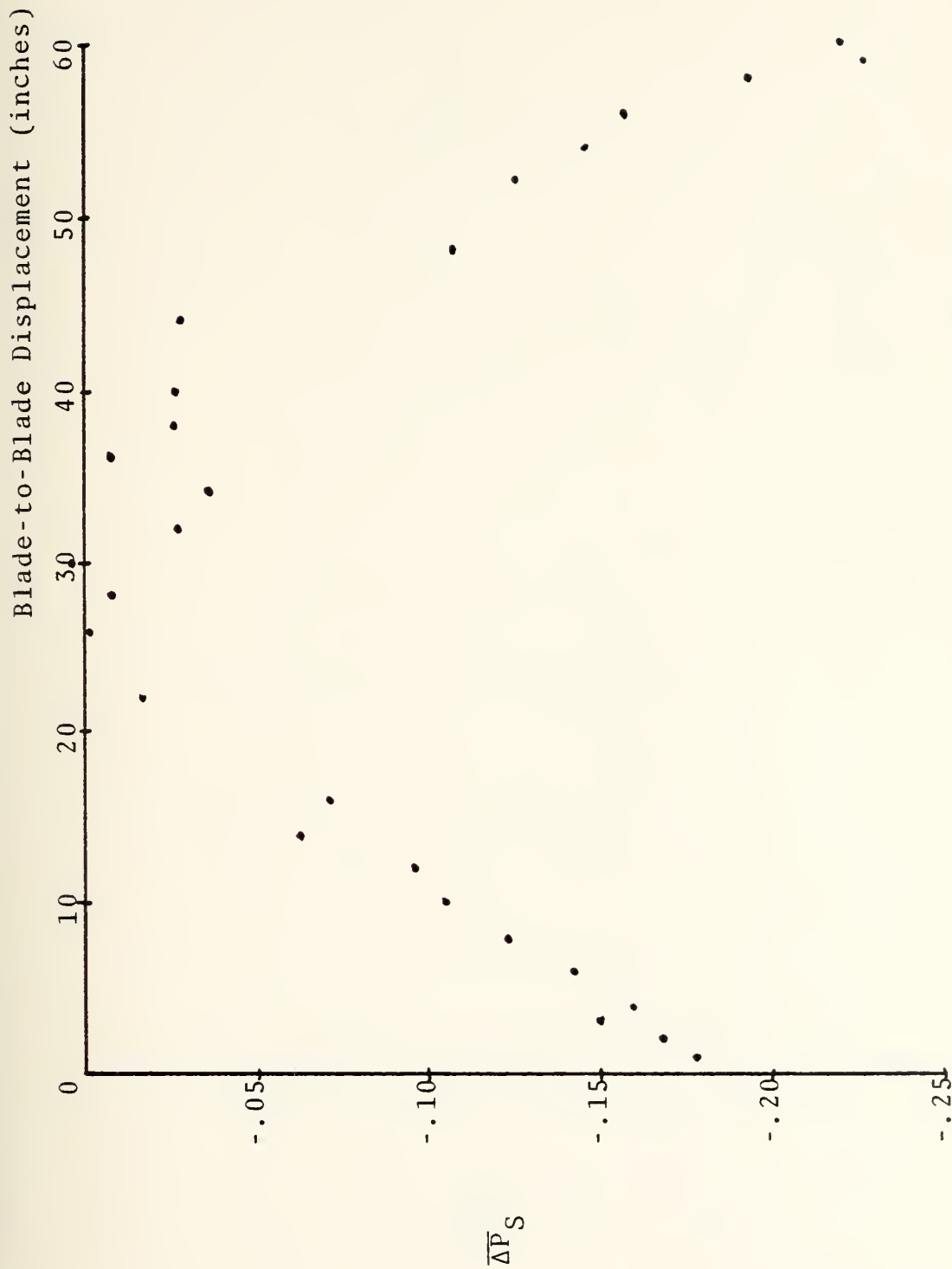


Figure B-3a. Blade-to-Blade Static Pressure Distribution at the Test Plane (at 10% Span, Velocity = 230 ft/sec) - Configuration #0



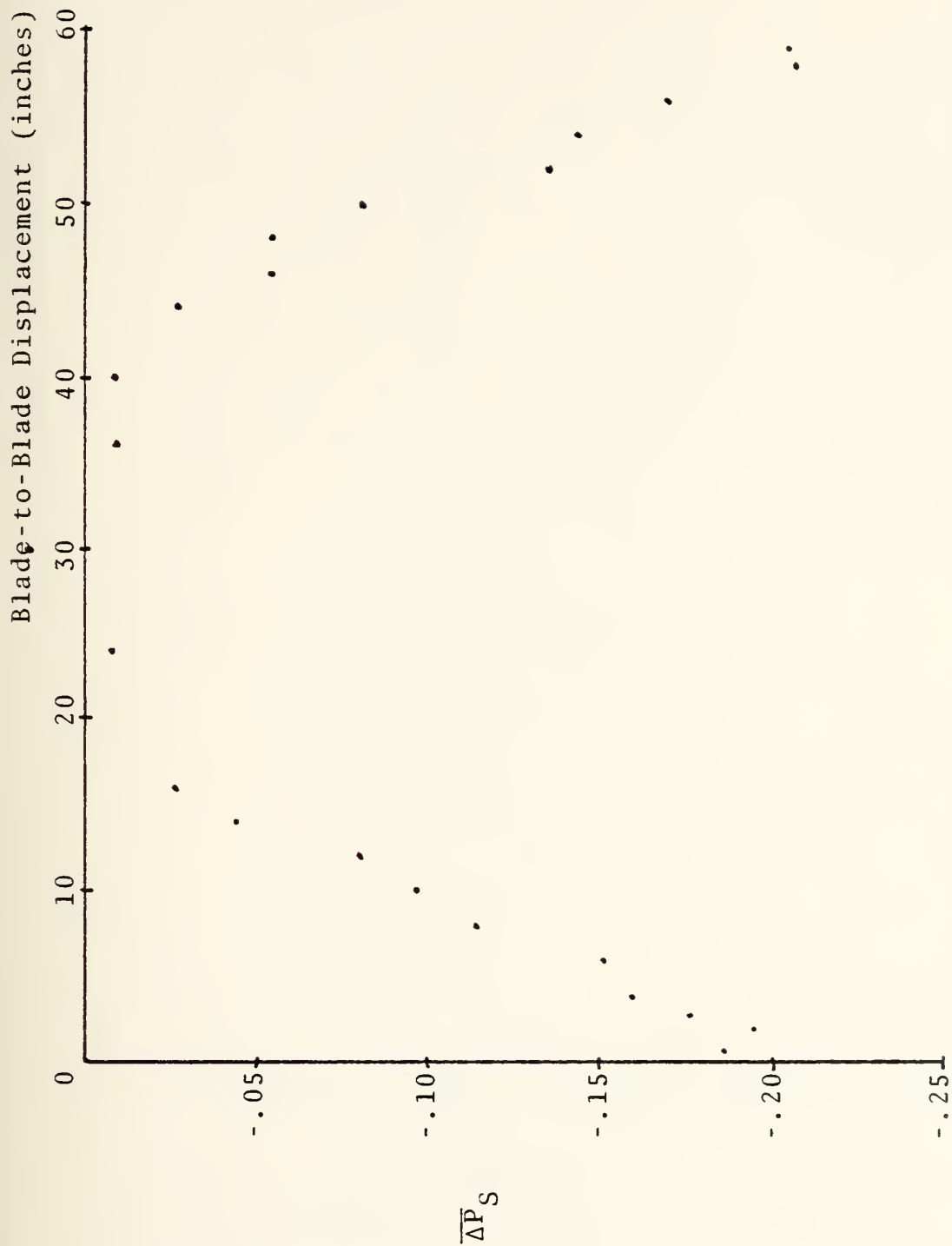


Figure B-3b. Blade-to-Blade Static Pressure Distribution at the Test Plane  
(at 30% Span, Velocity = 230 ft/sec) - Configuration #0



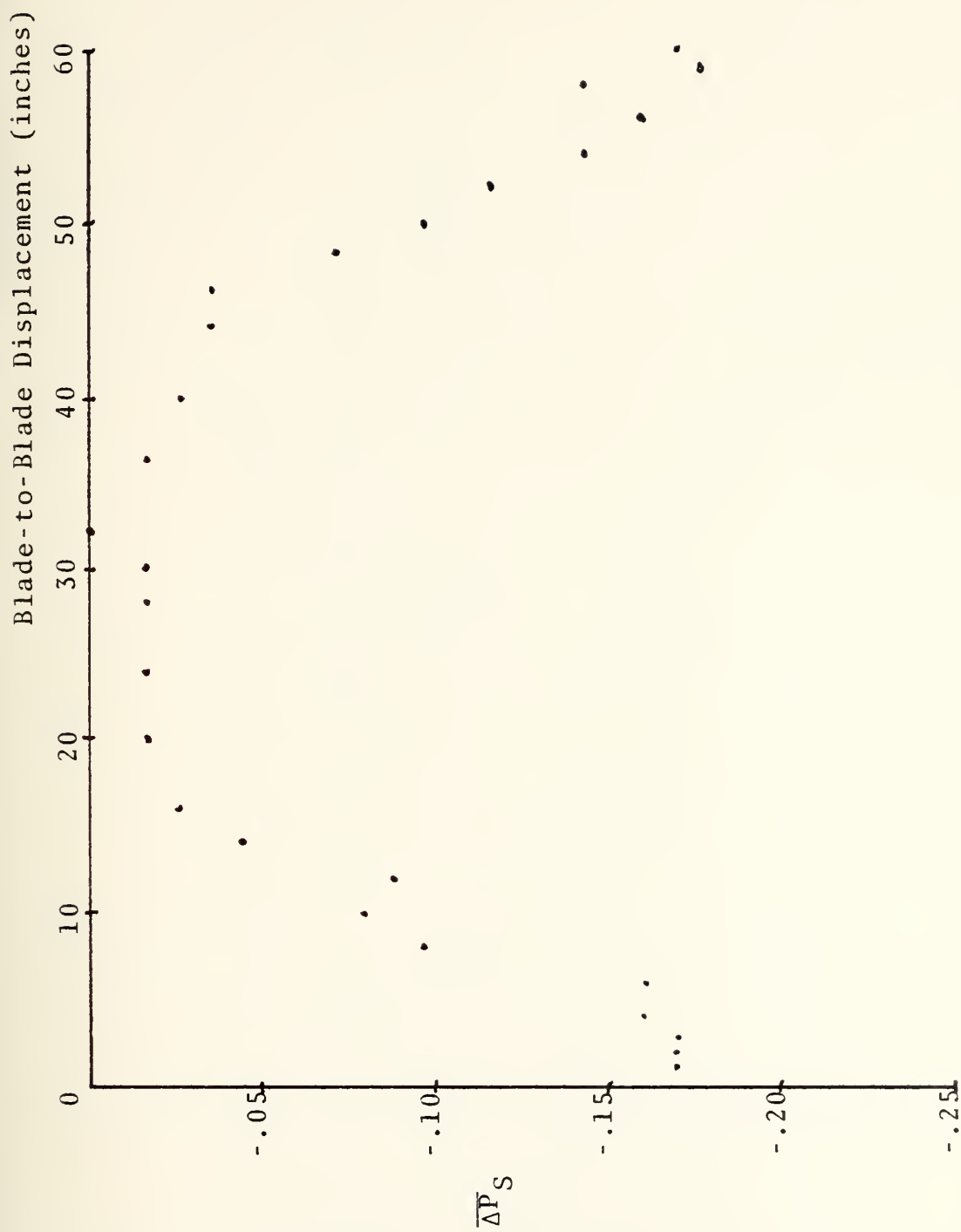


Figure B-3c. Blade-to-Blade Static Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 230 ft/sec) - Configuration #0



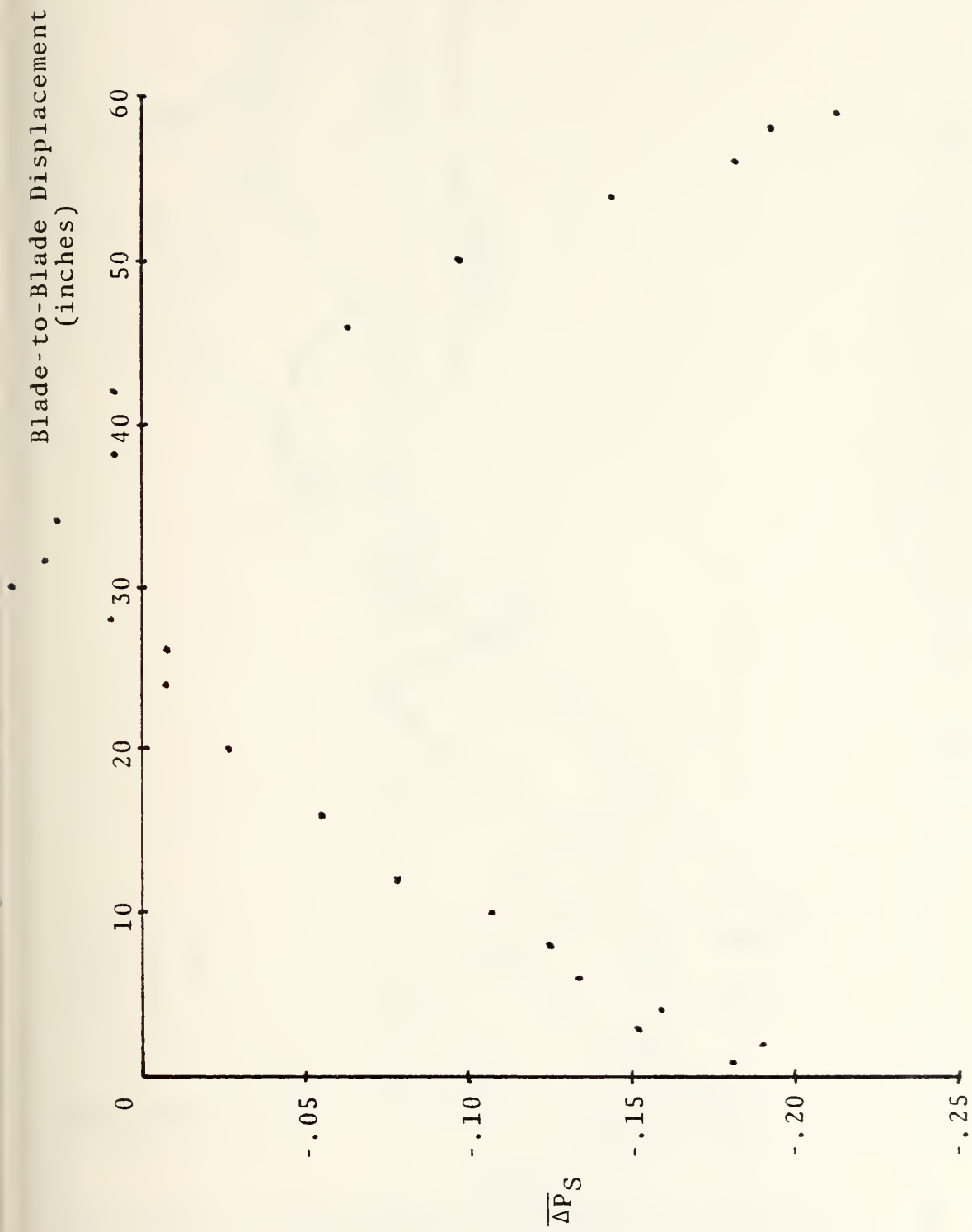


Figure B-3d. Blade-to-Blade Static Pressure Distribution at the Test Plane (at 90% Span, Velocity = 230 ft/sec) - Configuration #0





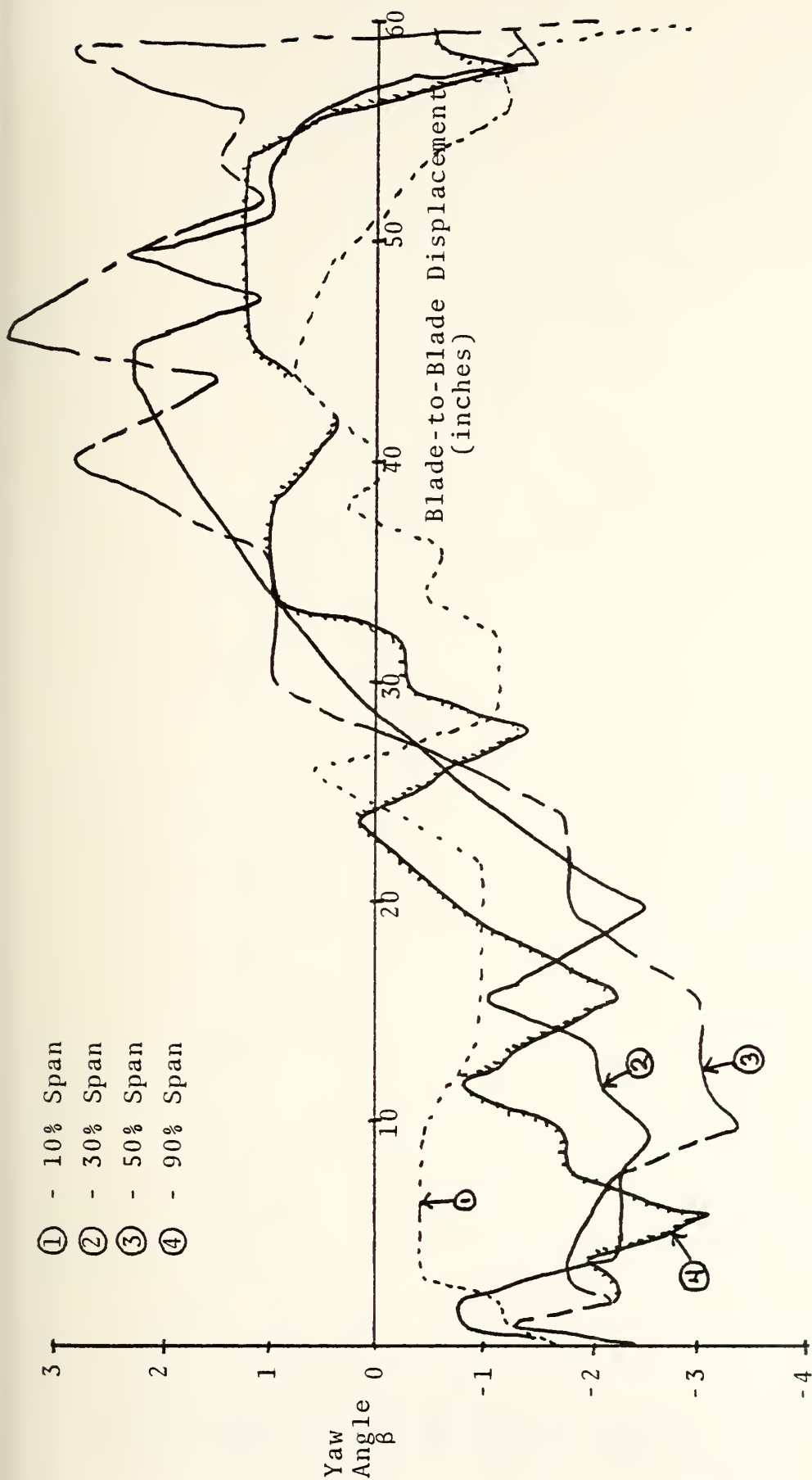


Figure B-4. Blade-to-Blade Yaw Angle Distribution at the Test Plane (at 10, 30, 50 and 90% Span, Velocity = 230 ft/sec) - Configuration #0



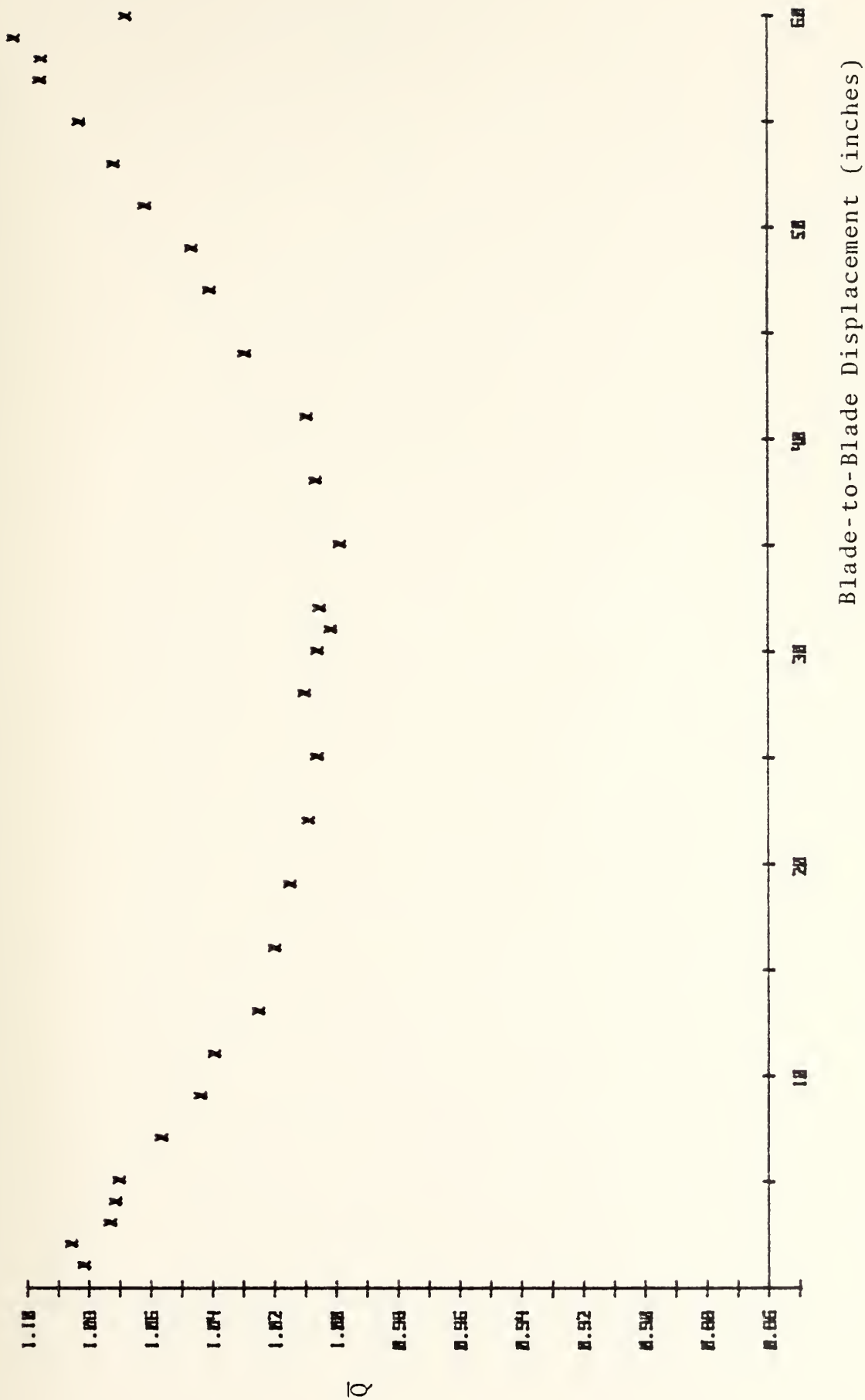


Figure B-5a. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 10% Span, Velocity = 270 ft/sec) - Configuration #1



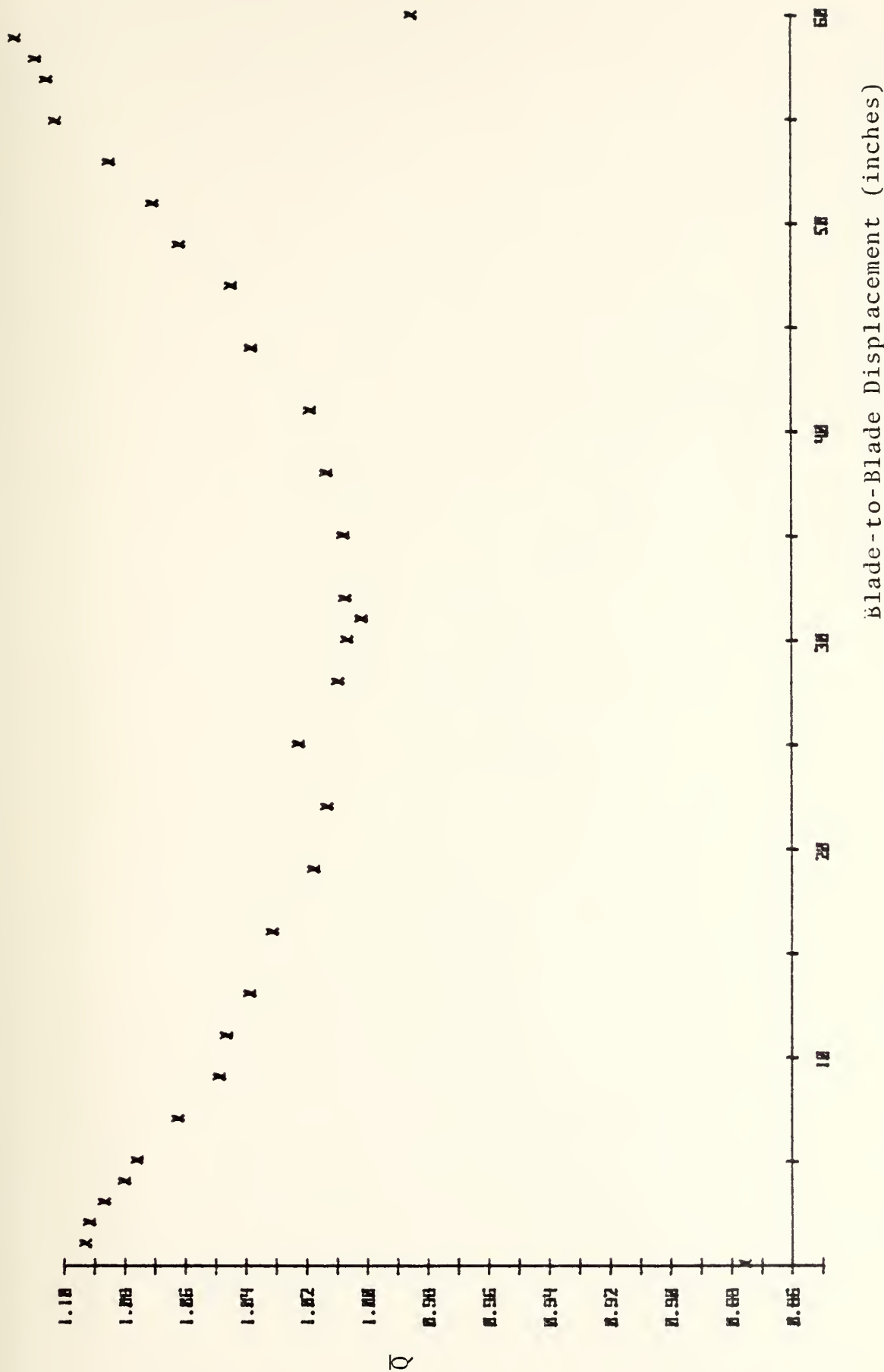


Figure B-5b. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 30% Span, Velocity = 270 ft/sec) - Configuration #1



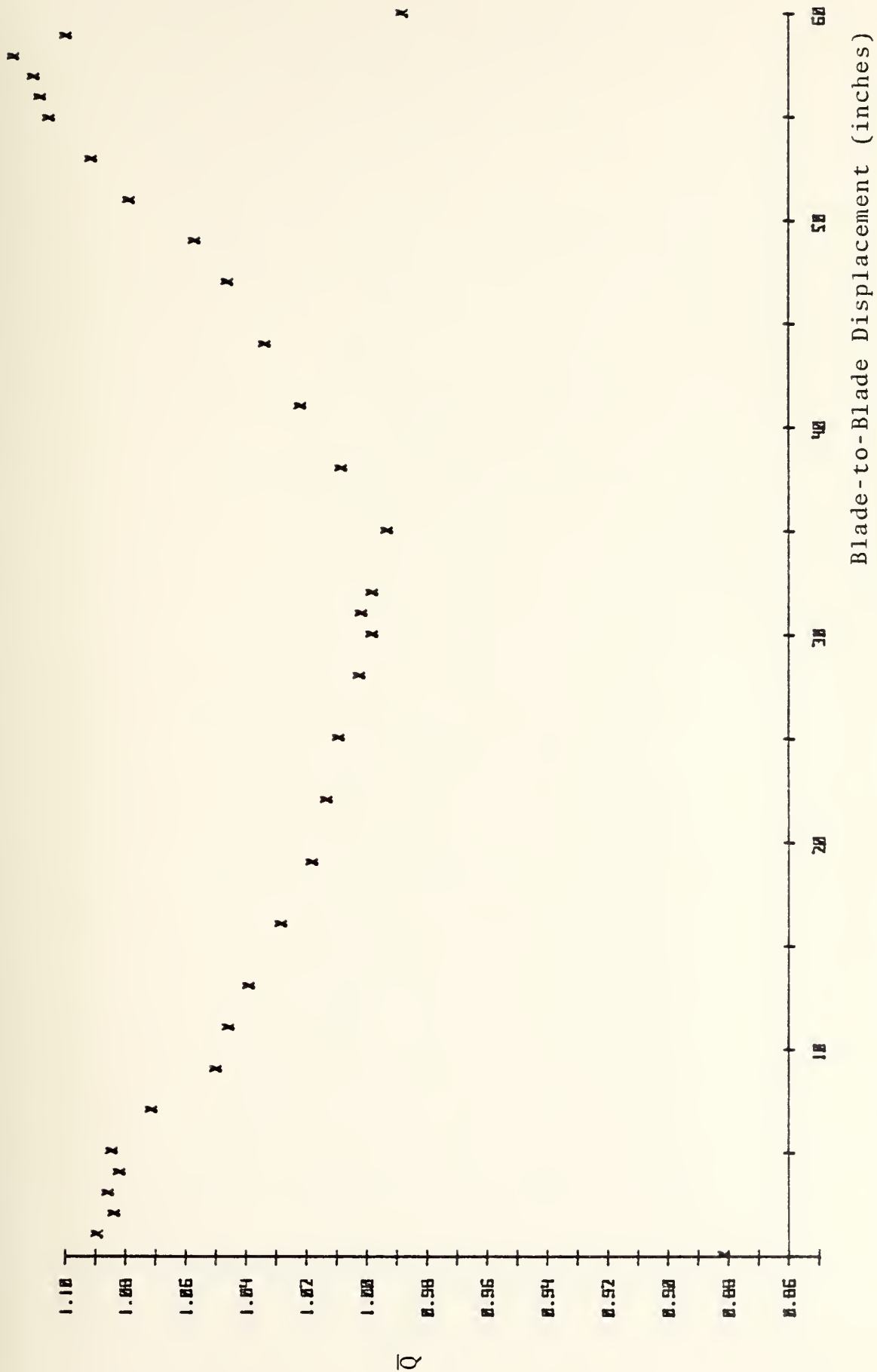


Figure B-5c. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 70% Span, Velocity = 270 ft/sec) - Configuration #1







Figure B-5d. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 90° Span, Velocity = 270 ft/sec) - Configuration #1



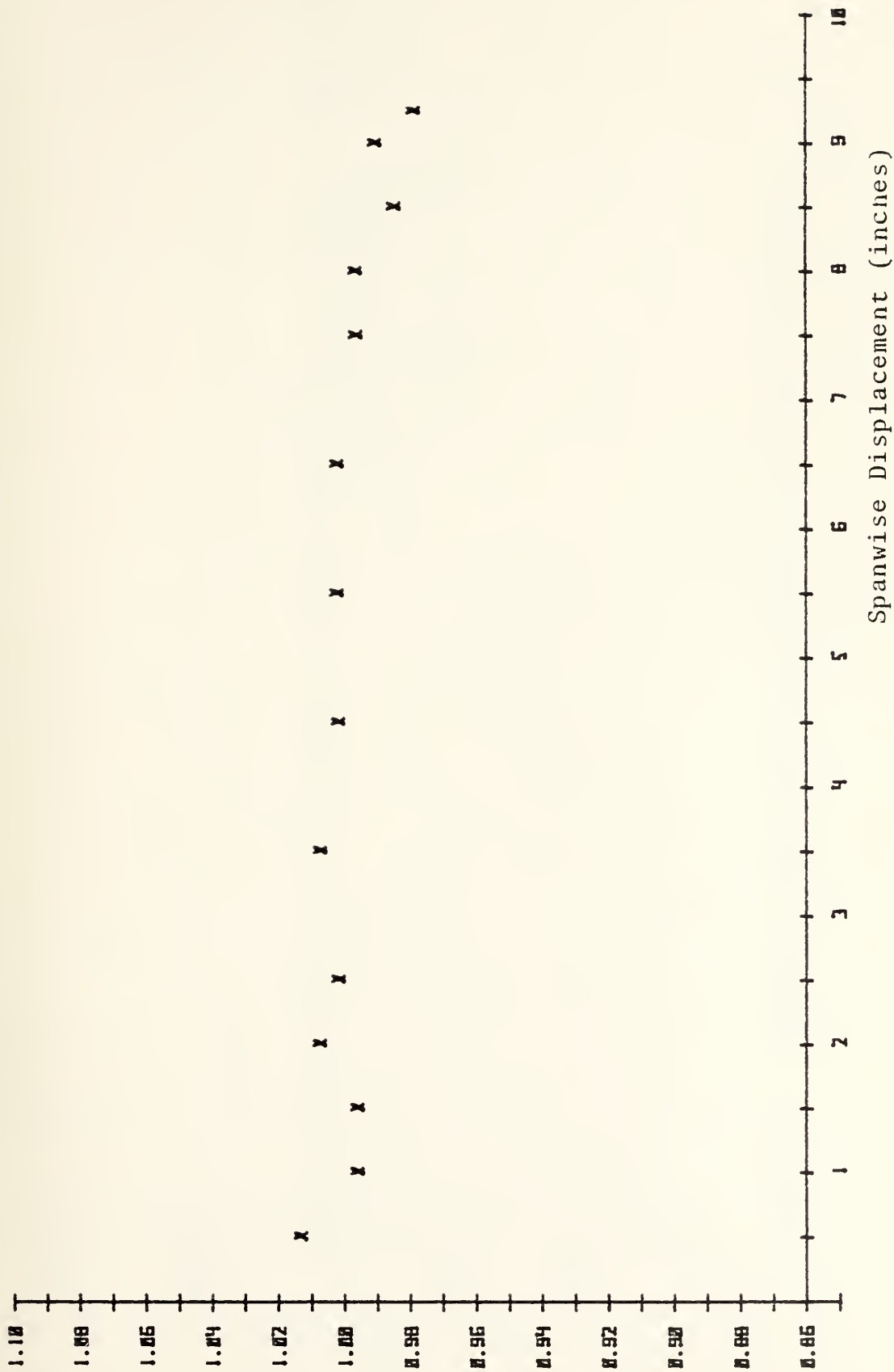


Figure B-6. Spanwise Dynamic Pressure Distribution at the Test Plane  
(at  $X = 15$  in., Velocity = 280 ft/sec) - Configuration #1



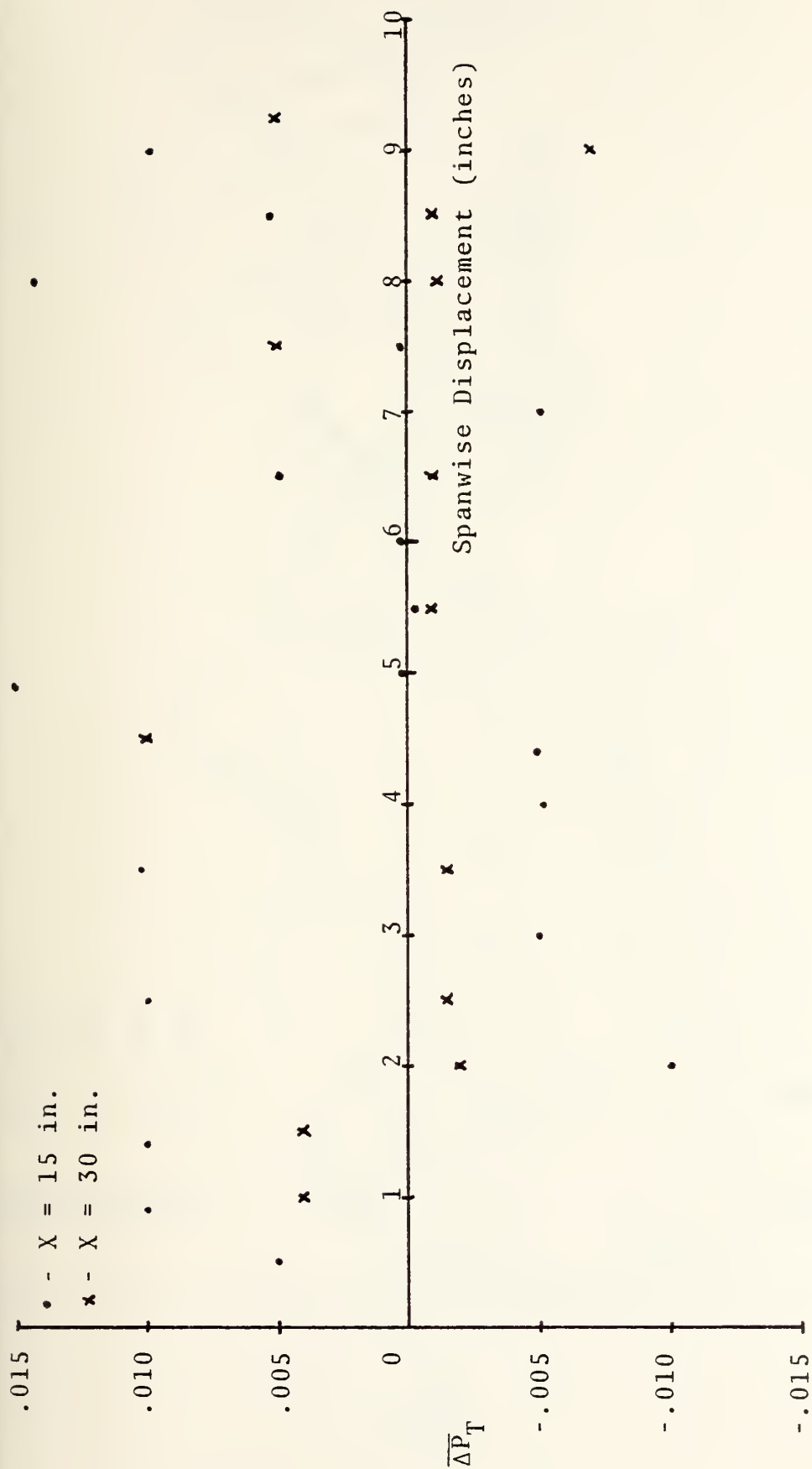


Figure B-7. Spanwise Total Pressure Distribution at the Test Plane  
(at X = 15 and 30 in., Velocity = 280 ft/sec) - Configuration #1



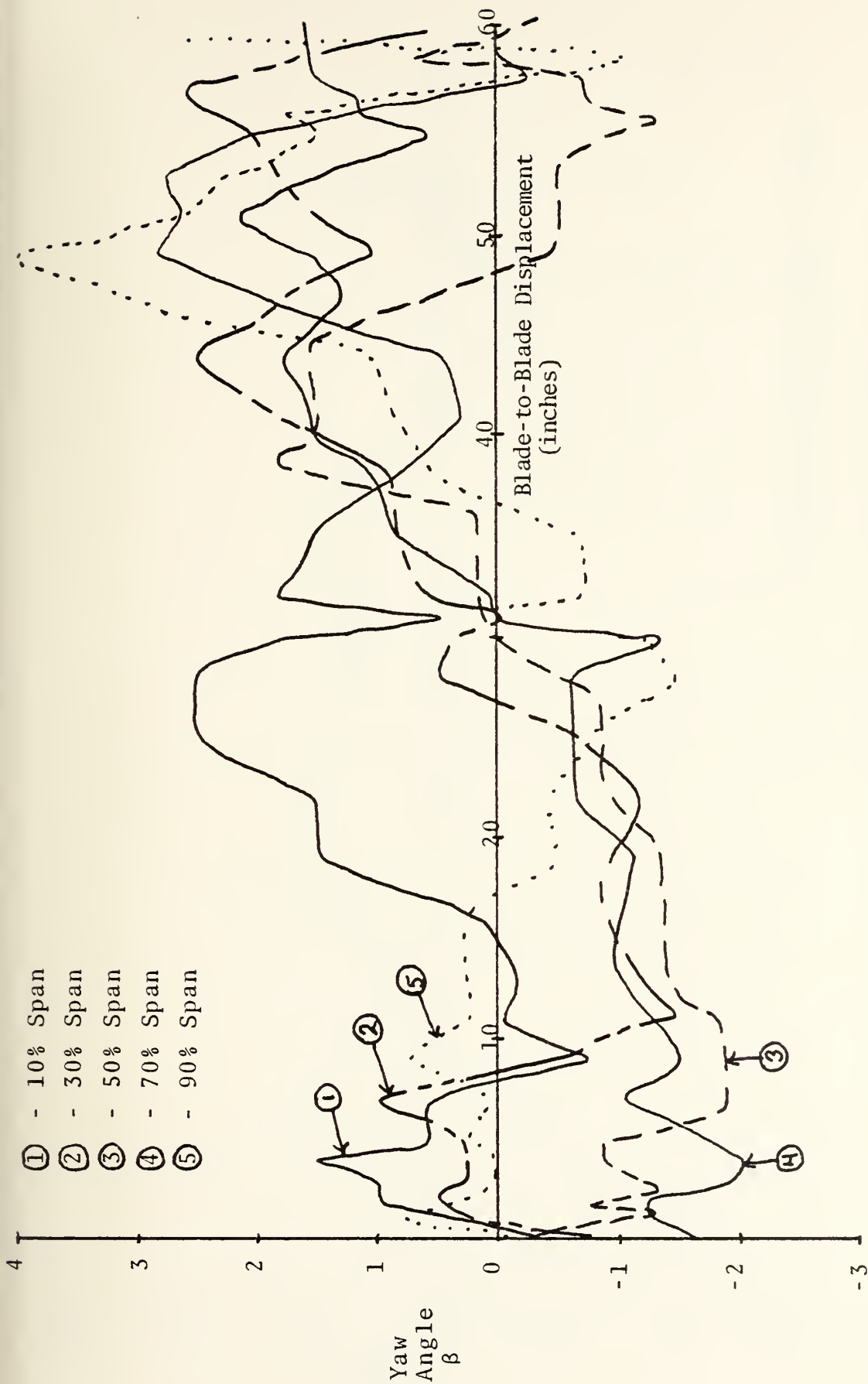


Figure B-8. Blade-to-Blade Yaw Angle Distribution at the Test Plane (at 10, 30, 50, 70 and 90% Span, Velocity = 270 ft/sec) - Configuration #1





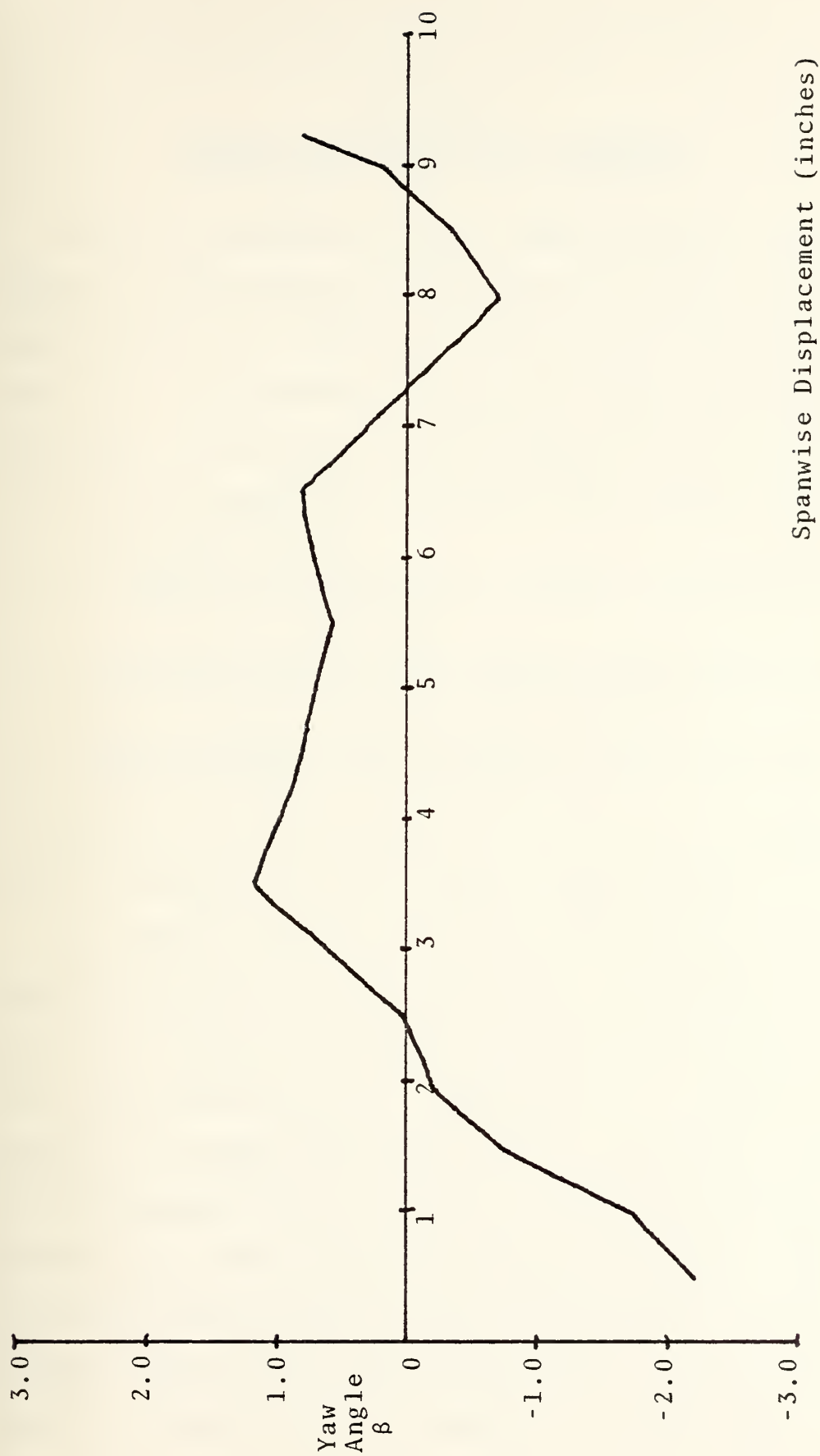


Figure B-9. Spanwise Yaw Angle Distribution at the Test Plane  
(at  $X = 15$  in., Velocity = 280 ft/sec) - Configuration #1



## APPENDIX C

### PLENUM DEVELOPMENT TEST PROGRAM

Several alternative designs were proposed to obtain a uniform, steady flow at the exit plane of the original bell-mouth contraction, which was the location for the inlet guide vanes for the cascade test section. Some considerations governing the selection of the approach to be followed were:

- 1) It was desirable to maintain the benefits of a large settling volume.
- 2) A simple construction was needed, one which was easy to alter if necessary.
- 3) Minimum boundary layer buildup was required prior to and within the cascade test section.
- 4) Equal boundary layer growth on opposite sides of the proposed design was required.

A modification of the plenum and bellmouth was carried out which satisfied all of the above requirements.

As shown by the FEM analysis in Appendix A, the velocity distribution created by an East-West contraction required an additional length of non-converging duct in order to relax (to a uniform condition). Configuration 2, Fig. C-1, was tested as a rough simulation of the "relaxation" concept. The arrangement was generated by using the existing plenum turning vanes and a sheet of 3/4 inch plywood. The measured distribution of  $\bar{Q}$  shown in Fig. C-2 showed a 2% variation through most of the tunnel width except in the deadwater regions (evidenced by  $P_T$  readings) at either end. The yaw



angles in Fig. C-3 were similar to Configuration 1 except as expected at the ends of the test plane. A major modification was made to the plenum to obtain two separate contractions in series with adequate lengths for relaxation. In Configuration 3, the east and west contours of the bellmouth were covered by vertical walls dropping down to a newly constructed set of East-West contraction contours. The construction sketched in Fig. C-4 (and detailed in Appendix D) allowed some relaxation of the velocity distribution created by the new East-West contours before the flow entered the original North-South bellmouth contours. The cardboard fillets shown in Fig. C-4 were not installed in Configuration 3. Measurements of  $\bar{Q}$  and yaw angles were made and the results are shown in Fig. C-5. The non-uniformities were acceptable through the center 40 inches of the test section. The velocity distributions, however, peaked rapidly at both ends and the sudden increase in yaw angles indicated a strong influx of fluid from the East and West walls toward the tunnel center. The close confinement of the deviant yaw angles to the tunnel ends, at  $X = 10$  inches and  $X = 50$  inches, was thought to be partially caused by the proximity of the plenum turning vanes to the lower contraction.

In Configuration 4, the vanes were moved three feet towards the east end of the plenum and the measurements were repeated. The results shown in Figures C-7 and C-8 were similar to those from Configuration 3; however, the peak



velocities at the ends were decreased slightly and the yaw angles varied across the entire 40 inch channel width.

At this point the west end lower contraction surface and vertical wall was tufted with three-inch strips of light string in order to examine the surface flow. A tufted rod was used to probe the flow outside the wall boundary layer. Figure C-9 shows a sketch of the tuft positions with air flow through the test plane at a dynamic pressure of 15 inches of water. The positions were traced from a Polaroid photograph taken from within the lighted plenum. The view was taken from the east end of the plenum looking west at a 45 degree upwards angle. The following observations were made.

- 1) Flow on the vertical wall in the vicinity of the upper contraction converged more rapidly than the contraction walls.
- 2) Large regions of deadwater existed in the upper corners.
- 3) Strong corner vortices (verified by the hand held probe) were present at the junction of the ceiling, upper contraction walls, and vertical walls.

These observations pointed to the presence of substantial secondary flow of the boundary layer fluid. Cardboard fillets were installed in Fig. C-4 to eliminate the deadwater areas. Pneumatic measurements taken with this configuration are shown in Figures C-10 and C-11. The results indicated little change in the  $\bar{Q}$  distribution; however, the peak yaw angles were notably increased.

The effect of the position of the turning vanes was questioned. When the plenum turning vanes were displaced three





feet to the east, it was apparent that very little flow was passing through any but the top turning vane passage. The vanes were therefore removed altogether as their functional benefit was questionable. The results of probe measurements for this Configuration 6 are shown in Figures C-12 and C-13. The distribution of  $\bar{Q}$  was seen to be similar to the previous one but an increase was noted in the peak values of the yaw angles. The yaw angles measured at 20% and 70% span were consistent with the proposed existence of strong contra-rotating vortices centered at about  $X = 10$  and 45 inches.

An audible low frequency pressure pulsation was noted and the static pressures in the cascade test section gave an unsteady measurement of  $\pm 0.75$  inches of water on the manometer columns. These observations suggested that the plenum turning vanes had served as a damping structure and perhaps as a spoiler which deterred the formation of large scale vortex motions.

The Datametrics Air Velocity Meter was used to probe the plenum chamber. Flow velocities through the sound baffles were quite uniform. However, at a distance of eight feet downstream of the sound baffles approximately 75% of the flow was seen to have passed through the upper one-half of the plenum at this location. It was also noted that little flow was actually moving around the east end lower contraction contour.

It was clear at this point that in order for the lower contraction to work properly, it was necessary to prevent



the flow from forming a smoothly accelerating jet from the distributed source represented by the sound baffles, to the line sink represented by the bellmouth exit plane.

Consequently, the plenum was reconfigured as shown in Section VI, Figures 14 and 15. Baffles were positioned to behave as spoilers and to better distribute the flow below the contraction. A 4-by-12 foot construction of 1/8 inch masonite was installed on the west end ceiling to divert the high velocity fluid from the ceiling down through the baffles. The measurements of  $\bar{Q}$  and flow angles for this final Configuration 7 are presented in Section VI.



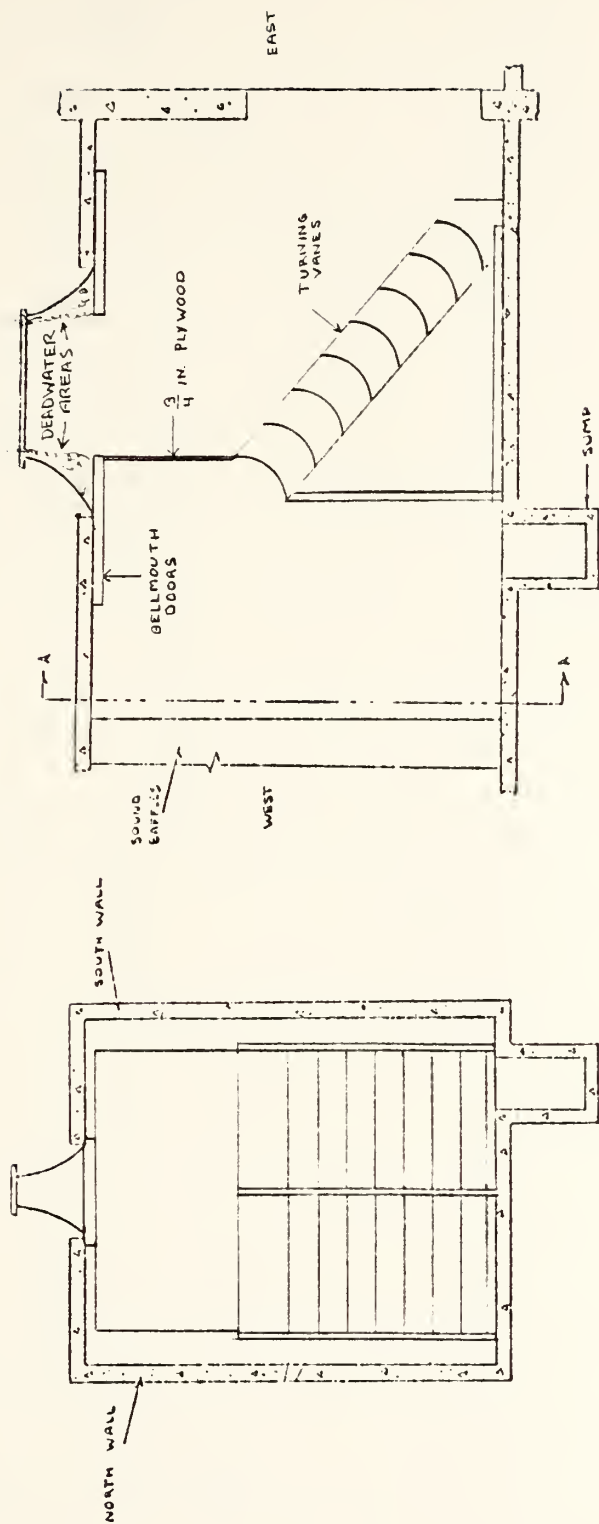


Figure C-1. Plenum - Configuration #2





Figure C-2. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 260 ft/sec) - Configuration #2





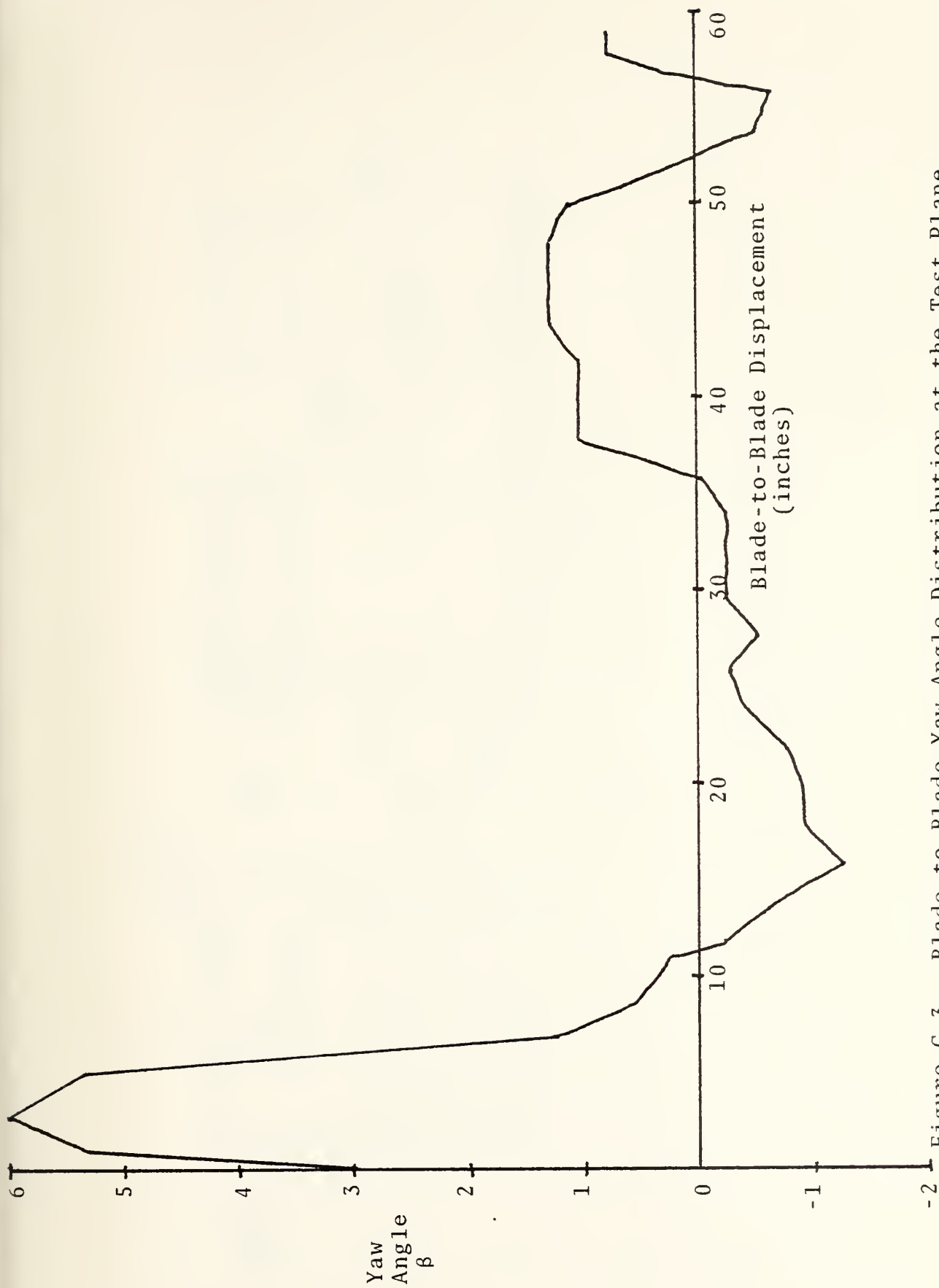


Figure C-3. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 260 ft/sec) - Configuration #2.



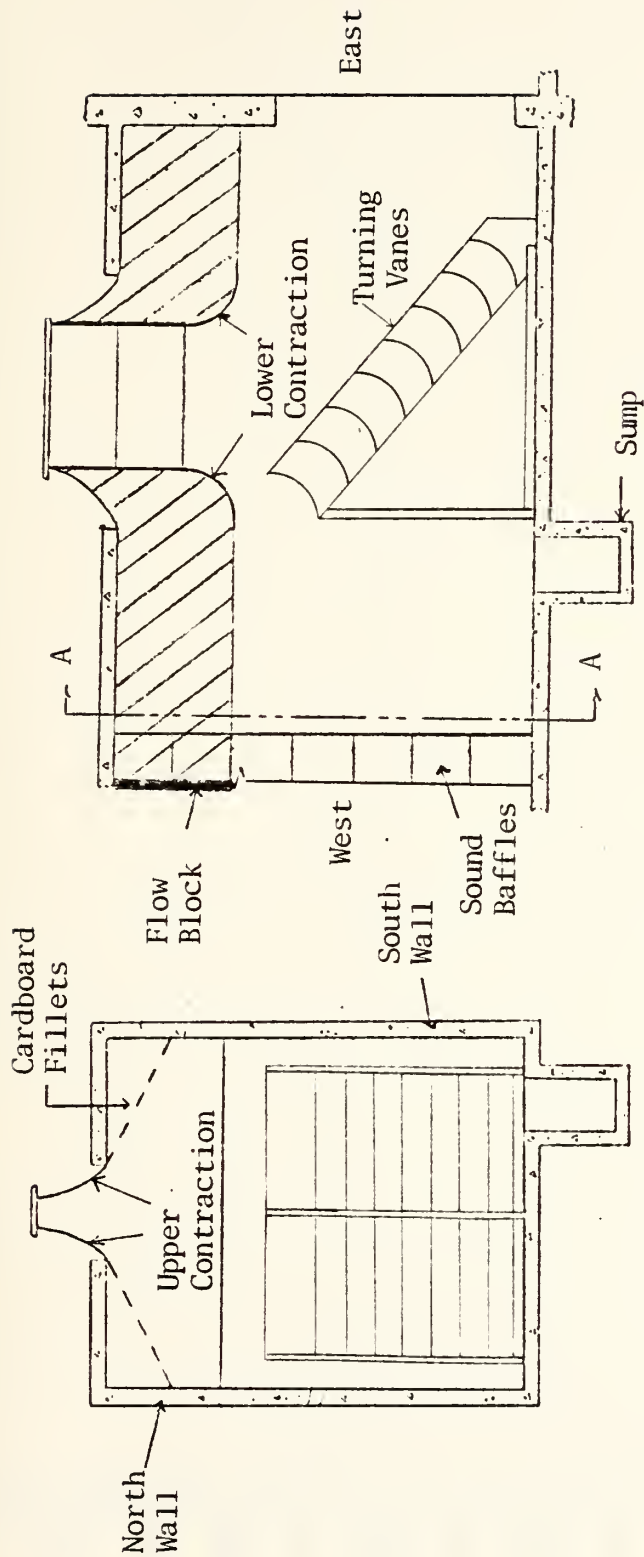


Figure C-4. Plenum - Configuration #3



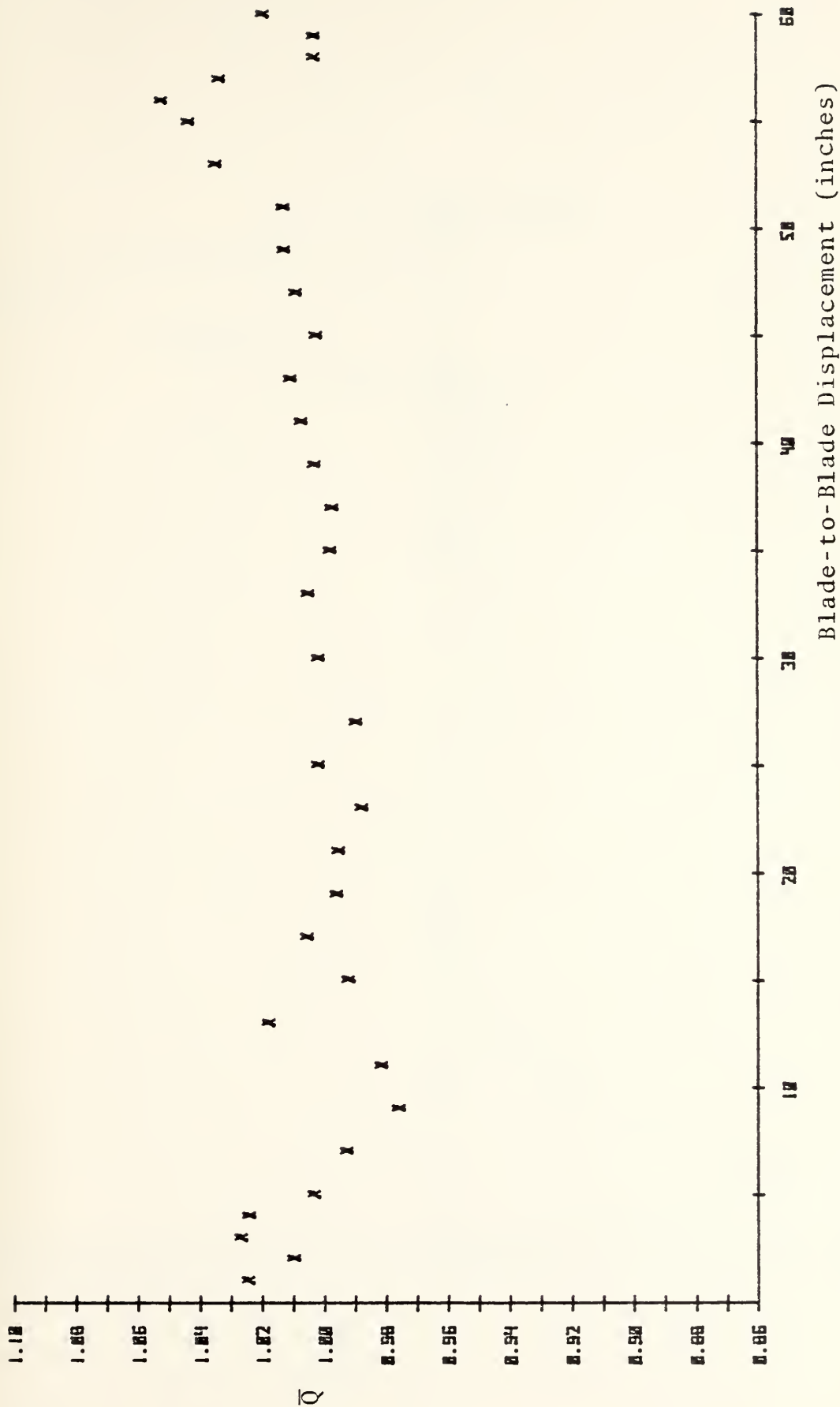


Figure C-5. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 250 ft/sec) - Configuration #3



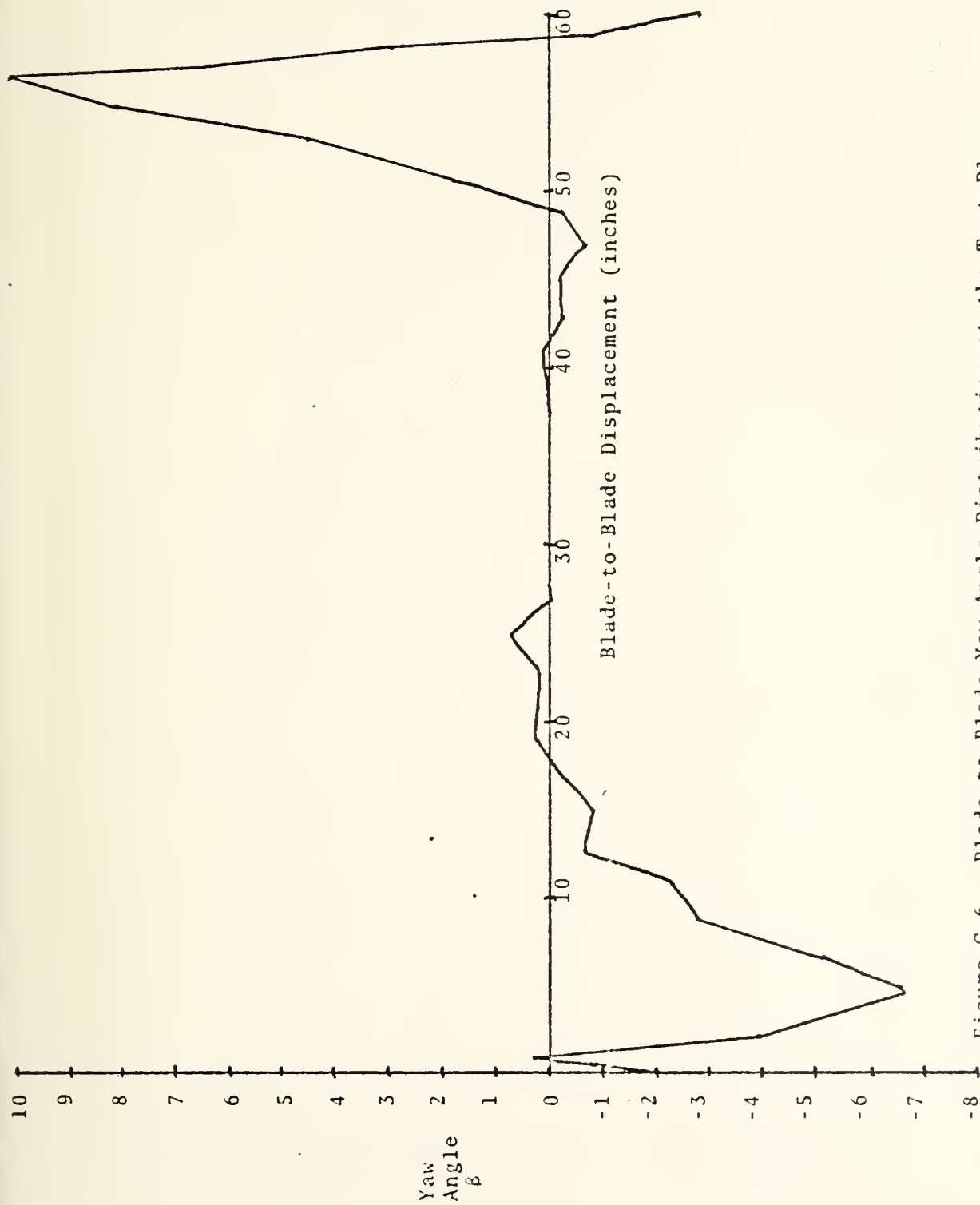


Figure C-6. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 250 ft/sec) - Configuration #3







Figure C-7. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 250 ft/sec) - Configuration #4



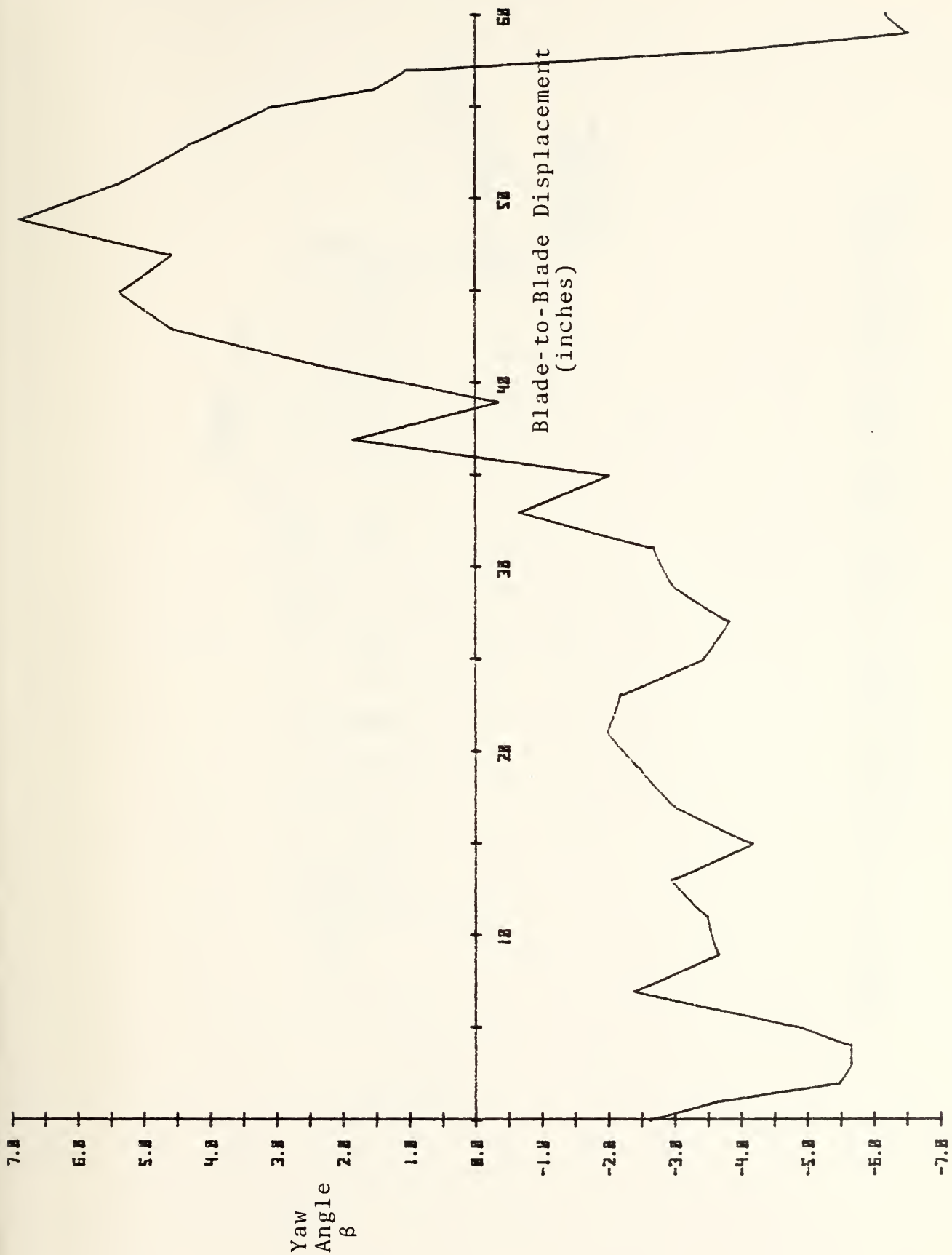


Figure C-8. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 250 ft/sec) - Configuration #4



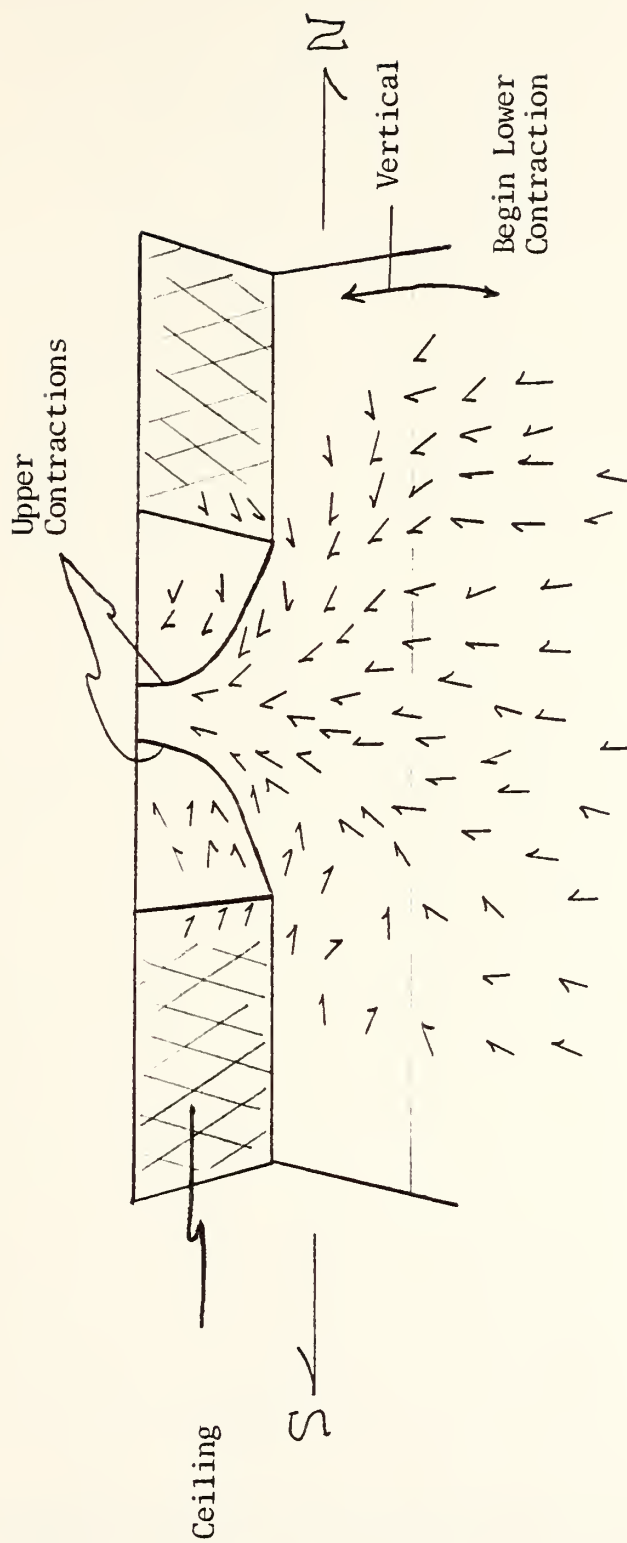


Figure C-9. Tuft Positions on West End Lower Contraction  
(Viewing East to West, 45 Degrees Upward Angle)



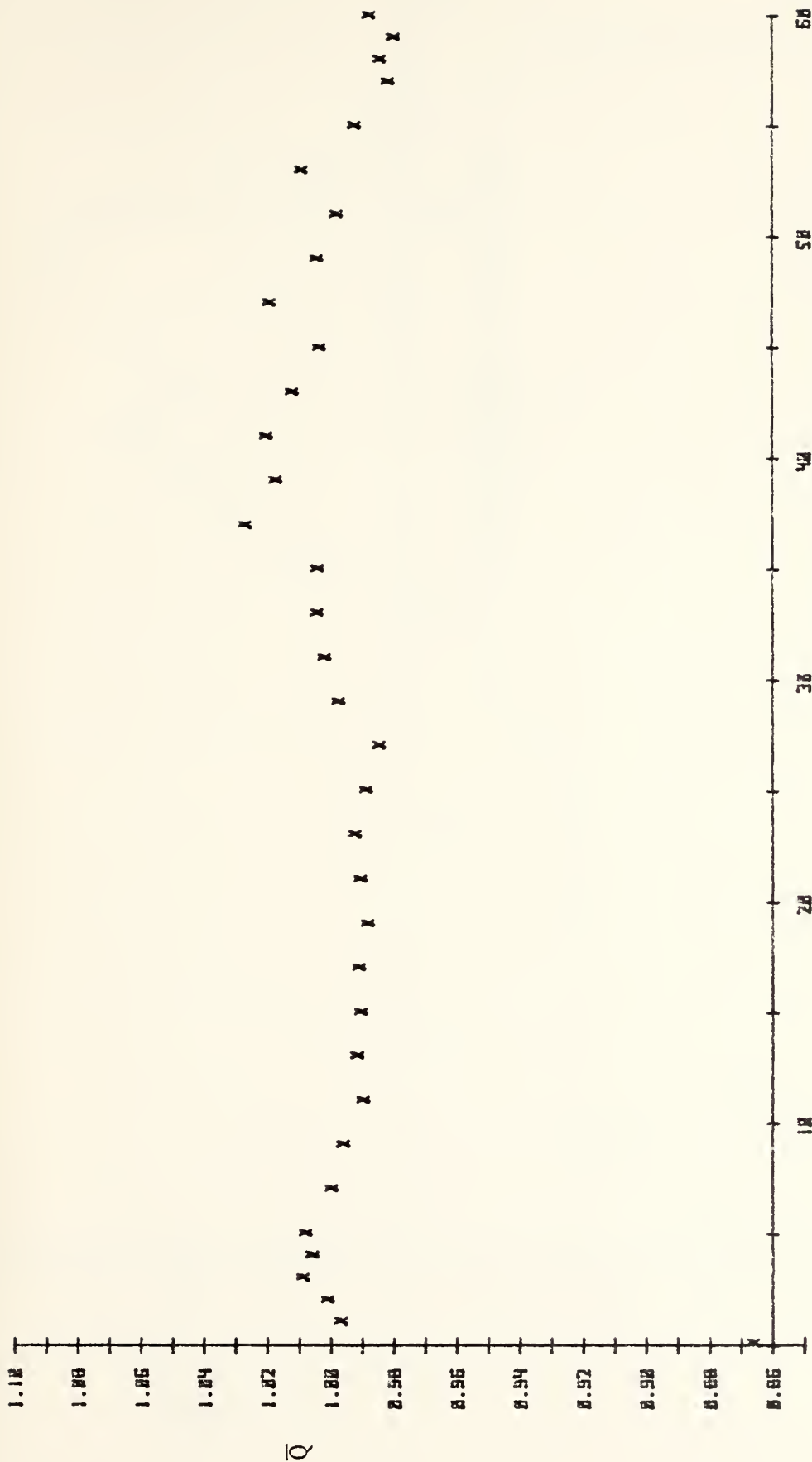


Figure C-10. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 270 ft/sec) - Configuration #5





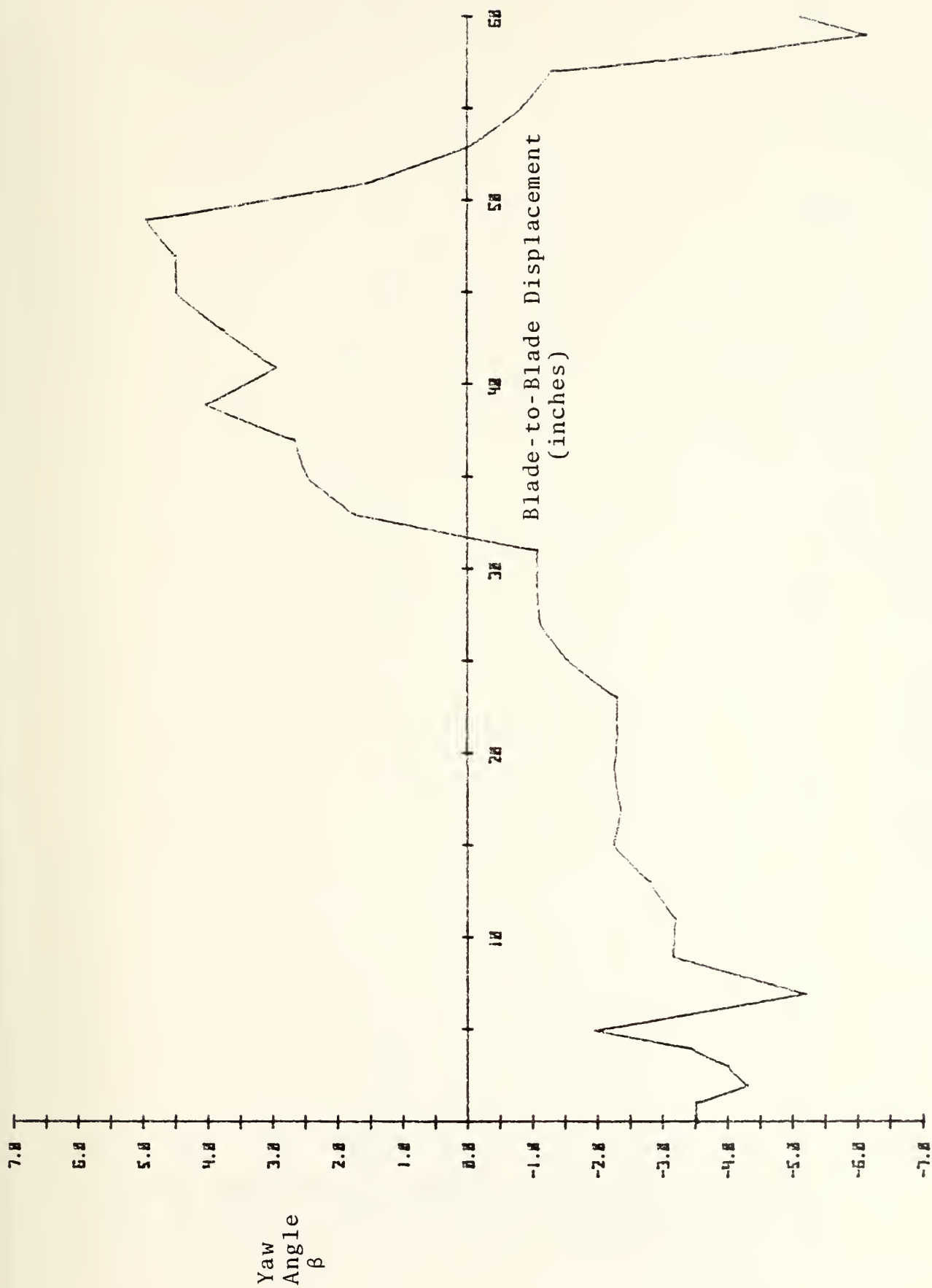


Figure C-11. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 50% Span, Velocity = 270 ft/sec) - Configuration #5





Figure C-12a. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 20% Span, Velocity = 250 ft/sec) - Configuration #6



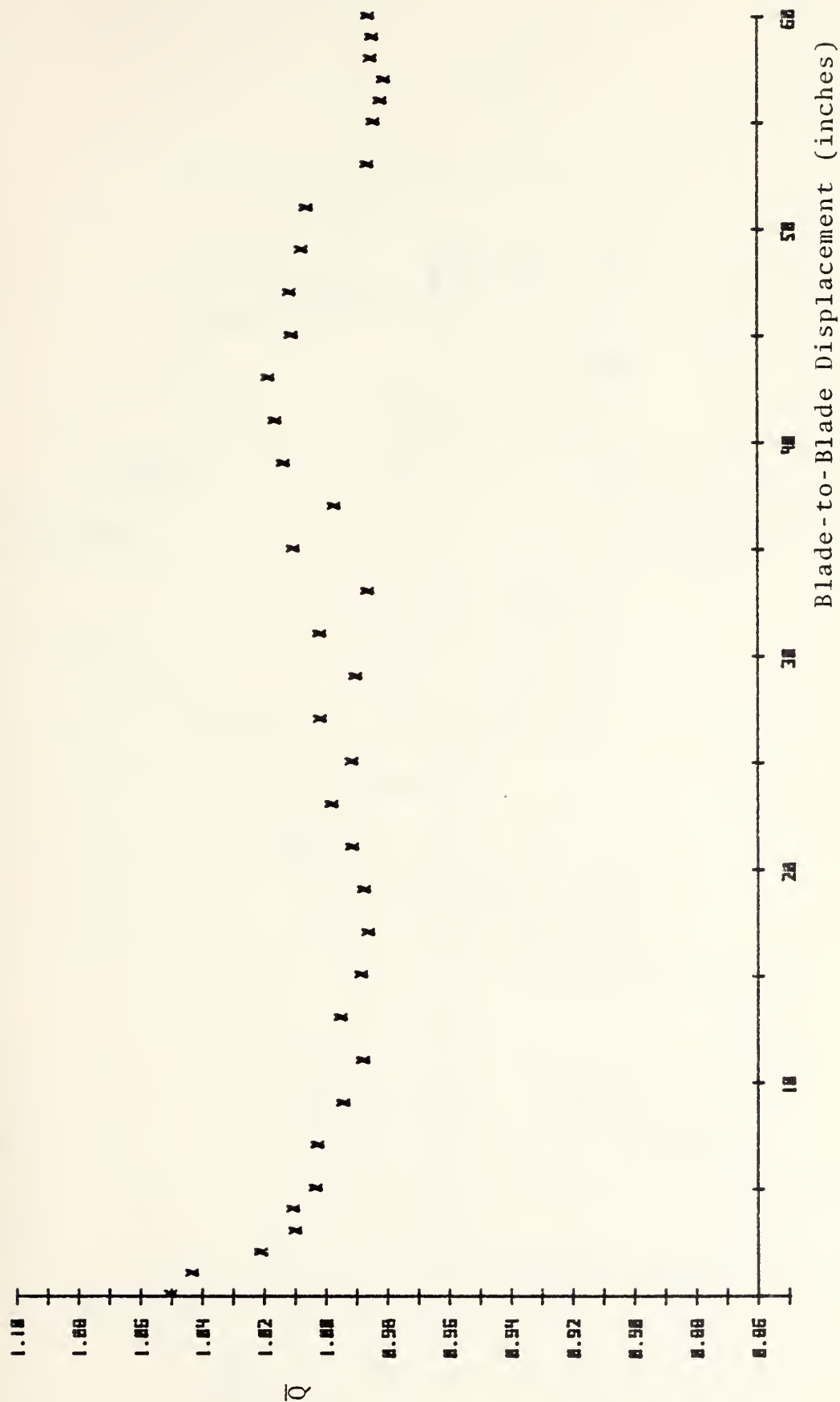


Figure C-12b. Blade-to-Blade Dynamic Pressure Distribution at the Test Plane  
(at 50% Span, Velocity = 250 ft/sec) - Configuration #6



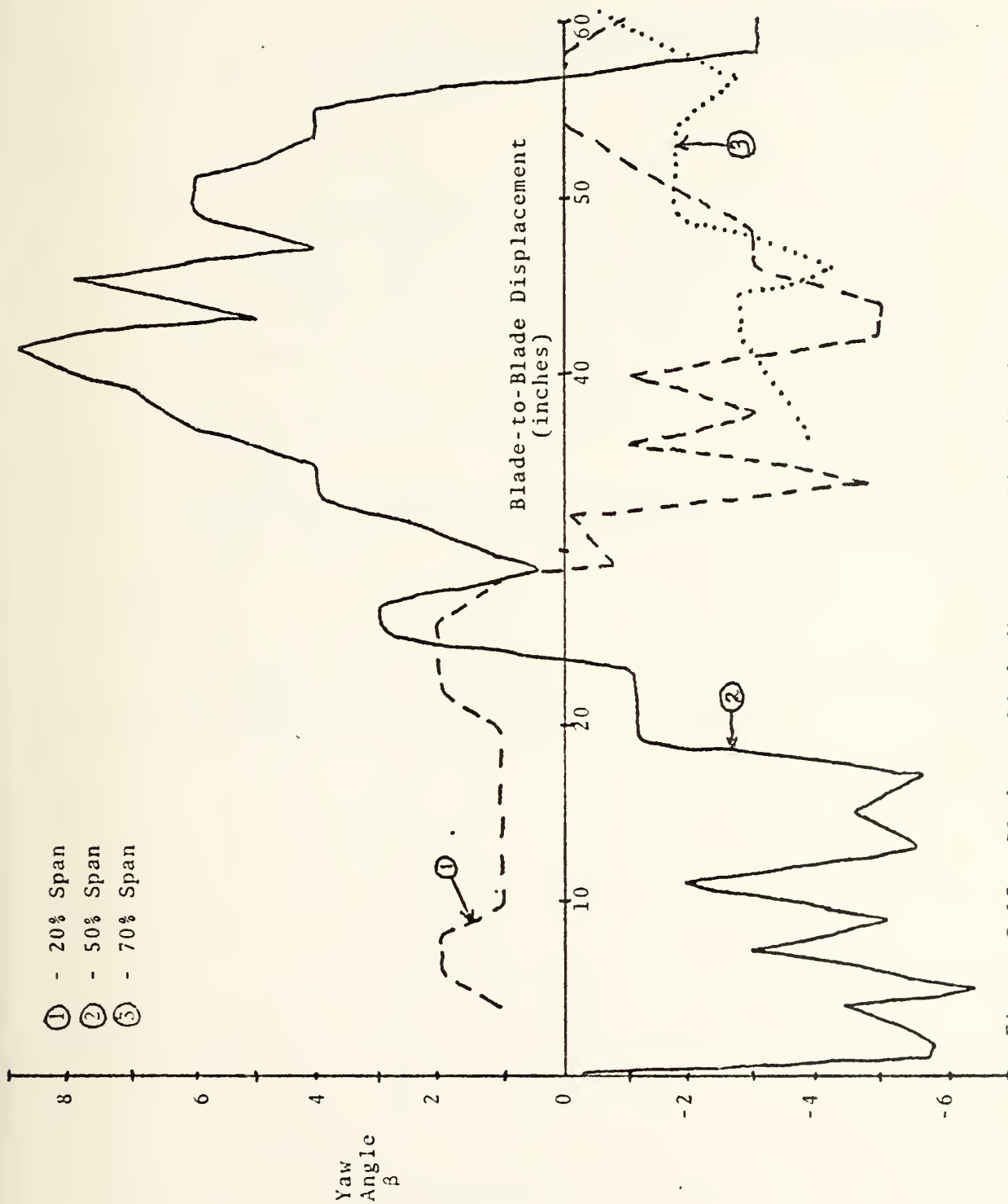


Figure C-13. Blade-to-Blade Yaw Angle Distribution at the Test Plane  
(at 20, 50 and 70% Span, Velocity = 250 ft/sec) -  
Configuration #6





## APPENDIX D

### PLENUM MODIFICATION DRAWINGS

The following drawings show the construction of the east and west lower contractions.

#### Construction Materials:

Figures D-1 through D-4

1. All framing material is construction grade pine.
2. The surface covering is 1/8 inch untempered masonite paneling in 4 x 8 ft sheets.
3. Countersunk wood screws secure the masonite to the framing.
4. Framing is secured to the cement wall with power-driven nails.

Figure D-5

The upper contour is part of the original bellmouth constructed from welded 1/8 inch steel plate.



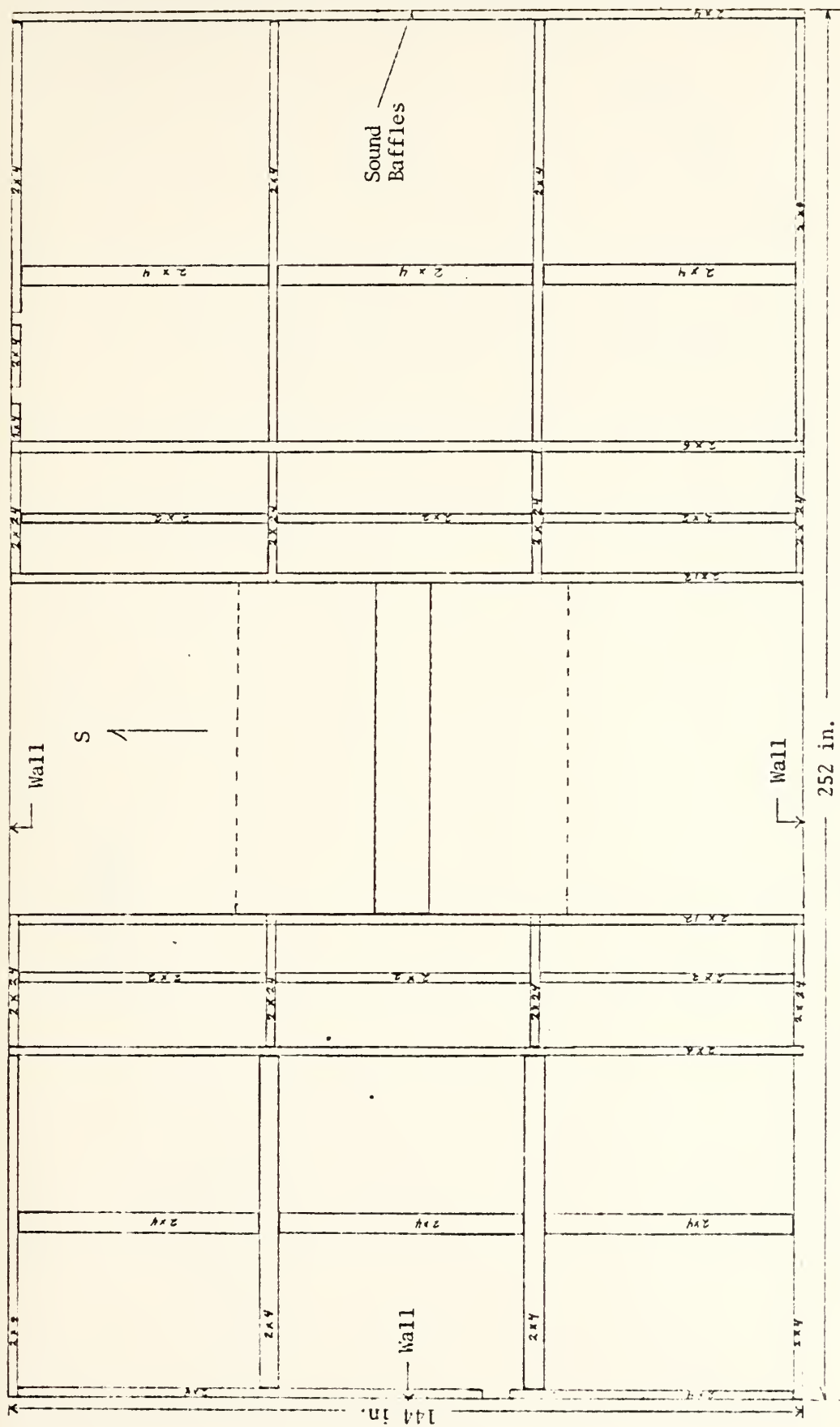


Figure D-1. Eastern and Western Lower Contraction Framework. Top View. Scale 1:25



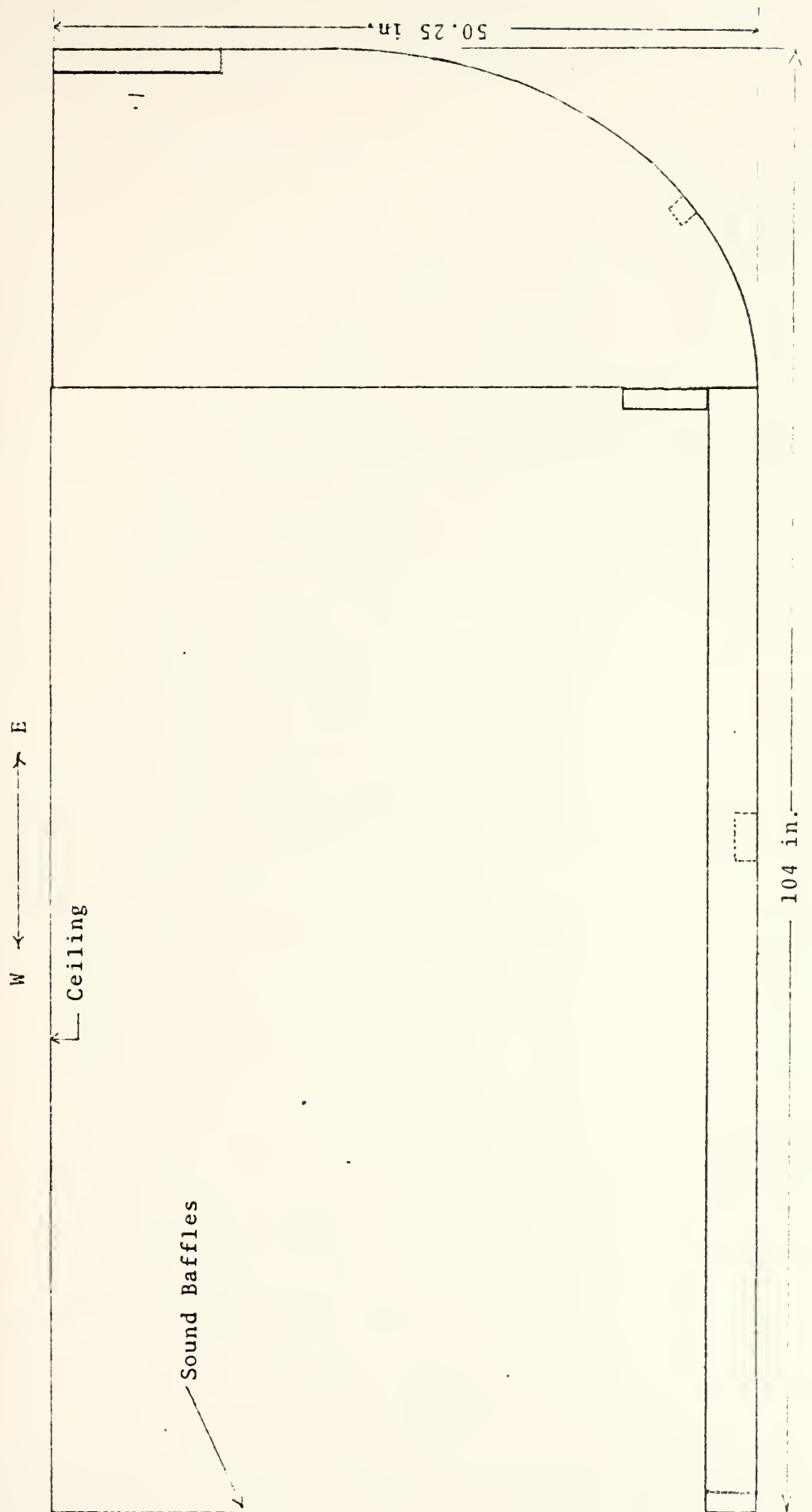


Figure D-2. W-E Elevation, Western Lower Contraction Framework  
Side View from South

Scale 1:10



E ← → W

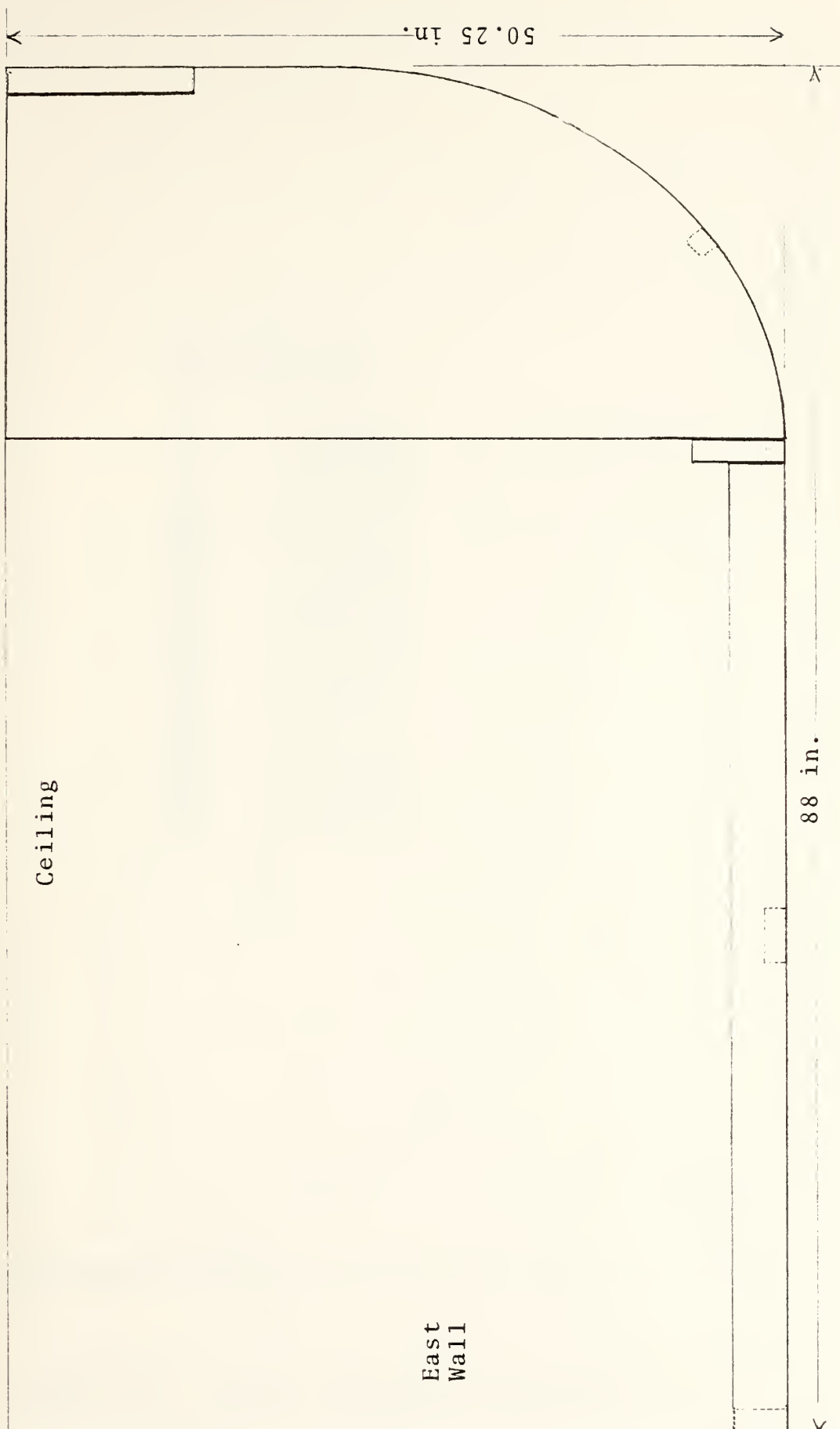


Figure D-3. E-W Elevation, Eastern Lower Contraction Framework  
Side View from North

Scale 1:10





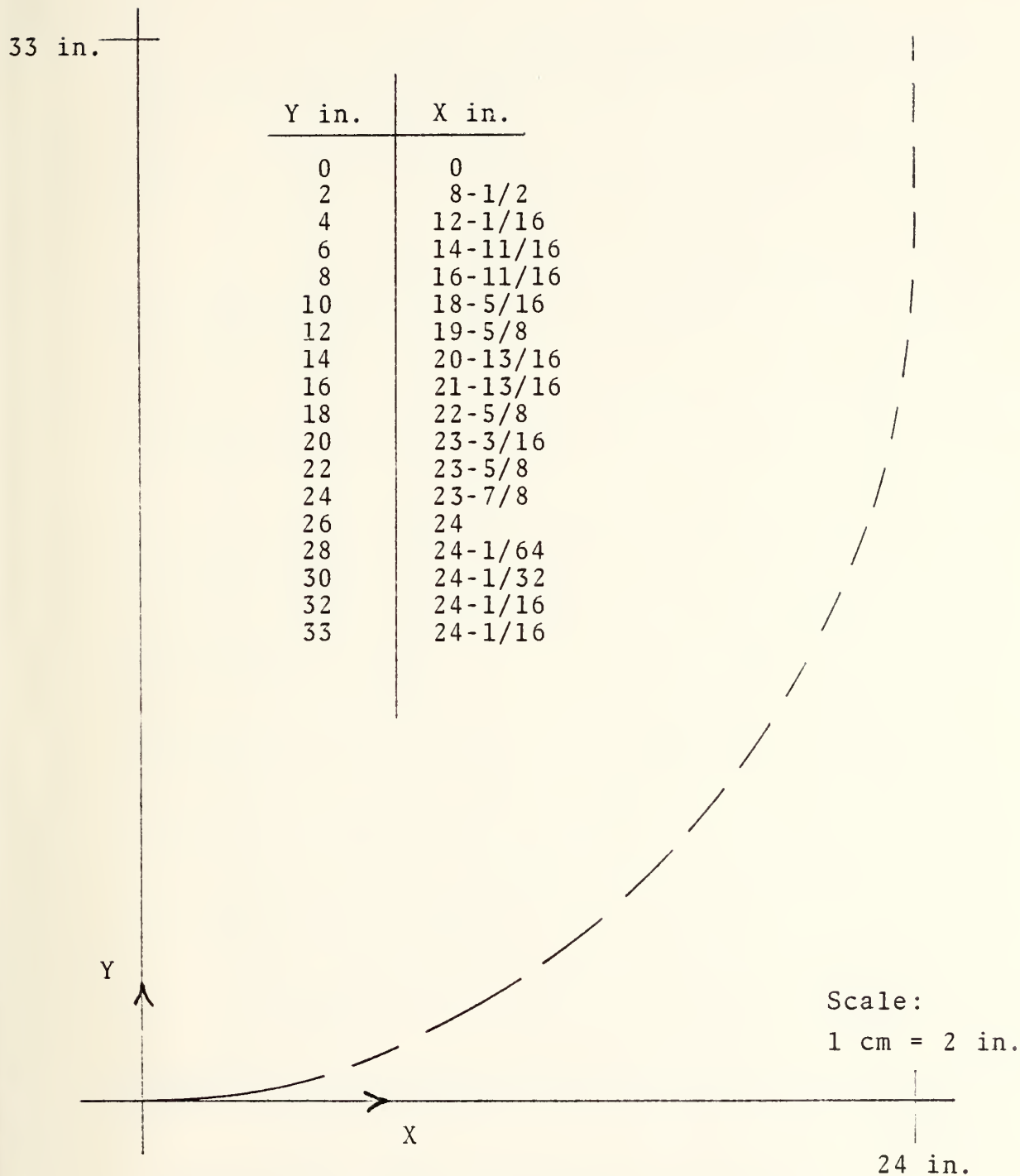


Figure D-4. Lower Contraction Contour



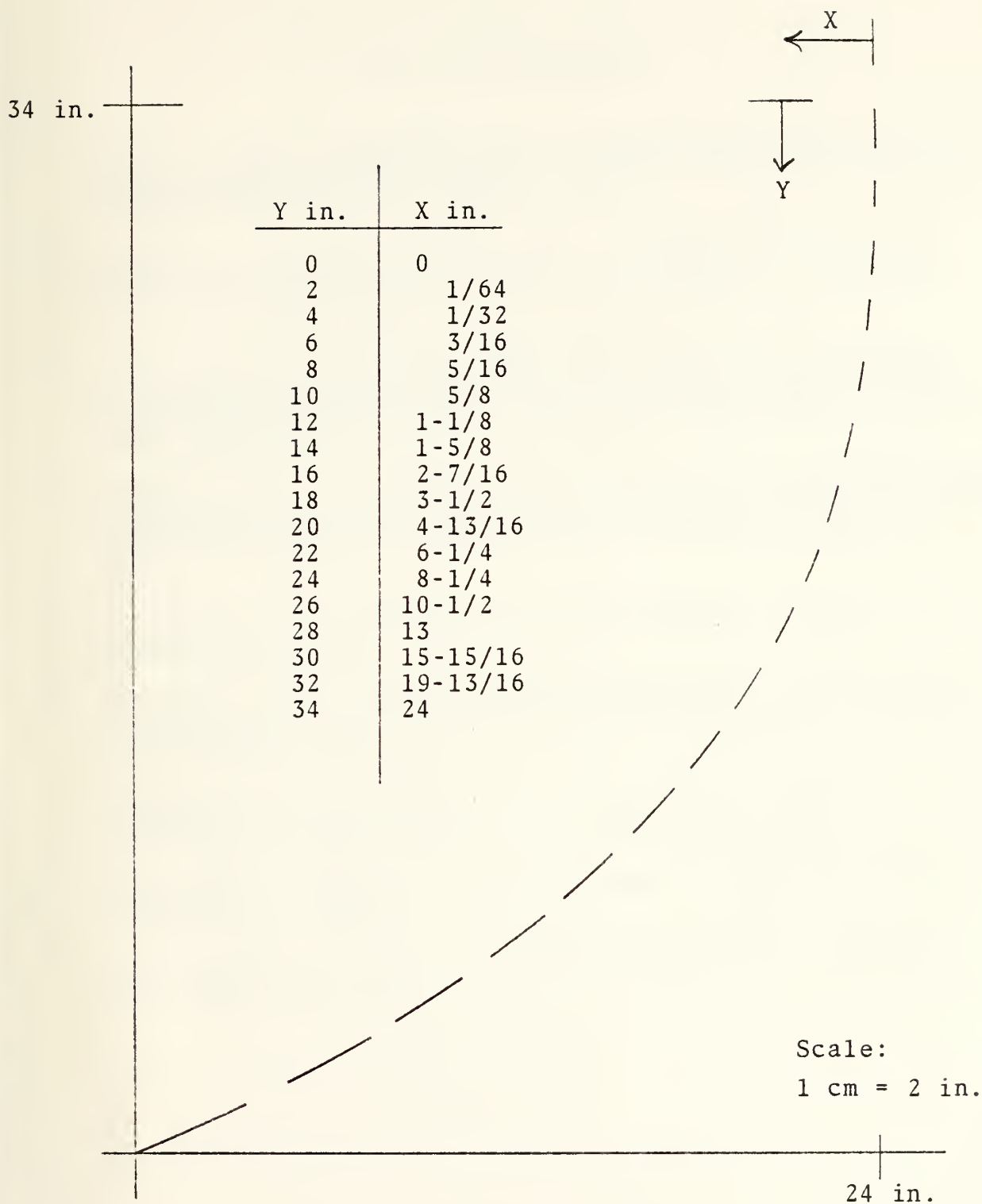


Figure D-5. Upper Contraction Contour



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